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HYDRAULIC BOOSTERS
BY-1Y, BY-10 and BY-13M

DESCRIPTION AND OPERATING INSTRUCTIONS

50X1-HUM

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INTRODUCTION

The hydraulic boosters manufactured by the plant fall by their design in three main groups: type EV-1V, hydraulic boosters, type EV-10 hydraulic boosters and type EV-13M hydraulic boosters.

The first group includes hydraulic boosters EV-1, EV-1V, EV-1M and EV-1M-PB, the second group includes hydraulic boosters EV-10, EV-10E, EV-10M, EV-5A and EV-8A, the third group includes hydraulic boosters EV-13M, EV-14M and EV-14MC.

The second group also comprises hydraulic boosters EV-9A, EV-9B and EV-9A-M, which are still used in aircraft. To improve performance of these hydraulic boosters, certain modifications in design were introduced, after which these boosters were designated EV-10, EV-10E and EV-10M.

The hydraulic booster groups differ from one another only by certain operation details and peculiarities of design. For example, type EV-10 hydraulic boosters, as distinct from type EV-1V and type EV-13M have a hydraulic lock by means of which the slide valve play is taken up automatically when the pressure is cut off. Type EV-10 hydraulic boosters are provided with a hydraulic two-way damper and types EV-1V and EV-13M hydraulic boosters are provided with a one-way vacuum damper.

In type EV-1V hydraulic boosters the cross feed, i.e. transfer of fluid from one cavity to another, is effected during the pressure cut-off by means of floating plungers, while in type EV-10 and EV-13M hydraulic boosters - by spring-loaded plungers.

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In hydraulic boosters of EV-1V type the slide valve device, the cross-feed plungers and the metering needle are arranged inside the operating rod. In type EV-10 hydraulic boosters the two latter elements are located inside the head, and the slide valve is fitted in a special case which is mounted on the hydraulic booster head.

In type EV-13M hydraulic boosters the slide valve and the ball valve are located inside the booster head, while the cross-feed plungers - inside the operating rod. Besides, in the case of EV-1V and EV-13M hydraulic boosters the power from the rods is transmitted to the slide valve with a ratio equal to 1, whereas in type EV-10 hydraulic boosters the ratio is approximately 1/2.

Besides the differences in overall dimensions and in the operating fluid pressure, a number of hydraulic boosters have certain differences in the design of individual elements as compared with the typical hydraulic booster of the group.

Hydraulic boosters EV-1M differ from hydraulic boosters EV-1V by reinforced trunnions and the extent of overlap of the slide valve pair.

Hydraulic boosters EV-10M and EV-10B differ from hydraulic boosters EV-10 by the presence of automatic delivery and drain valves for switching over to the duplicate system in case the main system fails.

Hydraulic boosters EV-5A and EV-8A differ from hydraulic boosters EV-10 in that they are provided with an auxiliary cross-feed valve located inside the operating rod cavity, and a return valve in the fluid supply pipe connection.

Hydraulic boosters EV-9A, EV-9B, EV-9AM are identical with hydraulic boosters EV-10 by their design and principle of operation.

Hydraulic boosters EV-10, EV-10B and EV-10M differ from hydraulic boosters EV-9A, EV-9B and EV-9AM, respectively by improved anti-corrosive coating, a sealing unit consisting

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of one round rubber ring and two leather rings instead of a pack of tongued cups, an improved design of duplicate system automatic switch-over valves and an improved design of the slide valve device.

Hydraulic booster EV-13M differs from hydraulic booster EV-14M only by the size of the actuating cylinder. Hydraulic boosters EV-14M and EV-14MC differ from each other by the stroke of the main and duplicate slide valves, the design of the cylinder trunnions and the rod end-pieces.

I. PURPOSE

Hydraulic boosters (Figs 1, 2 and 3) are auxiliary power units. They serve the purpose of reducing the force transmitted by the aircraft or helicopter controls to the control stick or control pedals.

During a flight at high speed a great part of the aerodynamic force acting on the control surfaces of the aircraft fitted with conventional control leverage would be transmitted directly to the pilot's hand.

Depending on the purpose and functions of the control units, the hydraulic boosters may be included either in the reversible or irreversible control systems. In the first case all the loads transmitted from the ailerons, elevators, or the swash-plate are taken directly by the hydraulic booster. Irrespective of the magnitude of the load the pilot will feel only the effort of the load spring on the pedals or the control stick. In the second case, when the hydraulic booster is connected in the reversible system, the hydraulic booster depending on the coefficient of irreversibility, will take only a part (in most cases a larger part) of the load originating from the control unit, whereas the other part is transmitted through the levers to the control stick or the control pedal. The latter part of the load is directly proportional to the total load originating from the control units, which enables the pilot to feel the change of strain on the control surfaces.

The hydraulic booster is fitted with a special automatic device (cross-feed valve), which turns the hydraulic booster

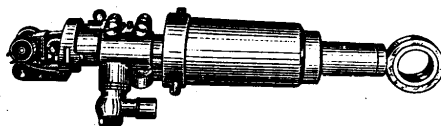


Fig. 1. Type 5V-1V Hydraulic Booster

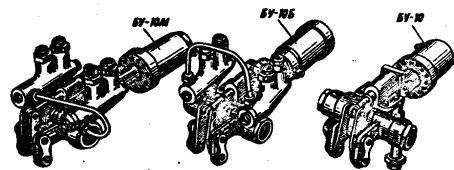


Fig. 2. Type 5V-10 Hydraulic Booster

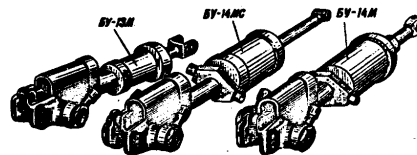


Fig. 3. Type 5V-13M Hydraulic Booster

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into an ordinary rod of the control leverage if the hydraulic system fails in operation. In this case the aircraft control is basically the same as in the case of a serviceable hydraulic booster, the only difference being that the pilot will feel a considerably greater force.

The hydraulic booster is mounted on the bracket of the aircraft or helicopter with the help of the cylinder trunnions in such a way that the booster cylinder is permitted to perform only the rocking movements.

The head of the hydraulic booster is connected to the rods linked to the control stick or control pedals, whereas the operating rod is connected with the leverage transmitting the motion of the ailerons, the elevator or the swash-plate.

II. OPERATING PRINCIPLE

Irrespective of peculiarities in the design, the operating principle of all hydraulic boosters is identical.

The hydraulic booster is essentially a two-way hydraulic power unit with a distributing device and consists of the following main elements (See Fig.4): cylinder 2, operating rod (piston) 3, and slide valve 1. The slide valve is a rod with four collars, three rows of radial ports, and with a plugged axial hole. The slide valve can be moved within certain limits relative to the operating rod. During the slide valve movement the position of the middle collars of the slide valve ensures a simultaneous symmetrical covering of channels "a" and "b" in piston 3 (See Fig.4).

As a result both cavities of cylinder 2 are disconnected simultaneously from passage "r" between the middle collars of the slide valve which is supplied with operating fluid from the pump, and from extreme passages "x" and "y", which communicate with the return line.

When the slide valve disconnects both cavities of the cylinder from the delivery and return lines, it is known to be in the NEUTRAL position. The operating rod (piston) in this case remains motionless since the hydraulic fluid is either denied admission into both cavities of the cylinder, or is admitted into both cavities in equal quantities.

Aircraft control effected with the help of a hydraulic booster is based on the follow-up system, that is the magnitude and speed of ailerons or other control surfaces are in direct proportion to the magnitude and speed of the motion of the control stick or control pedals.

The hydraulic booster is connected in the leverage system of rods in series, that is the operating rod is one of the intermediate links of the leverage and the cylinder is the guide bushing of this link.

The hydraulic booster is mounted on the ball support of bracket 7 (Fig.5) which allows it to perform rocking motions in the plane of the rods rocking. Control system rod 1 running from the pedals or from the control stick is connected to the slide valve of hydraulic booster 4, whereas the rods running from the controlled units (ailerons, elevator, etc.) are linked with the operating rod.

Control of the aircraft by means of a hydraulic booster consists in the pilot's shifting the control stick or pedal in either direction, thus changing the position of the slide valve of the hydraulic booster relative to the operating rod. For example, when slide valve 1 (Fig.6) moves to the right, both collars of the slide valve also move in the same direction and channels "a" and "b" will be partially uncovered on the left side; as a result the operating fluid will be admitted in the left cavity of the cylinder through uncovered channel "a". Under the action of the hydraulic fluid, operating rod 3 will start moving rightward relative to cylinder 2. This will force the fluid in the right cavity of the cylinder into

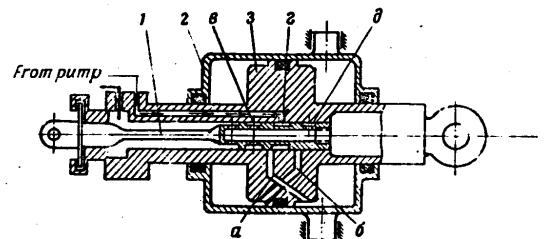
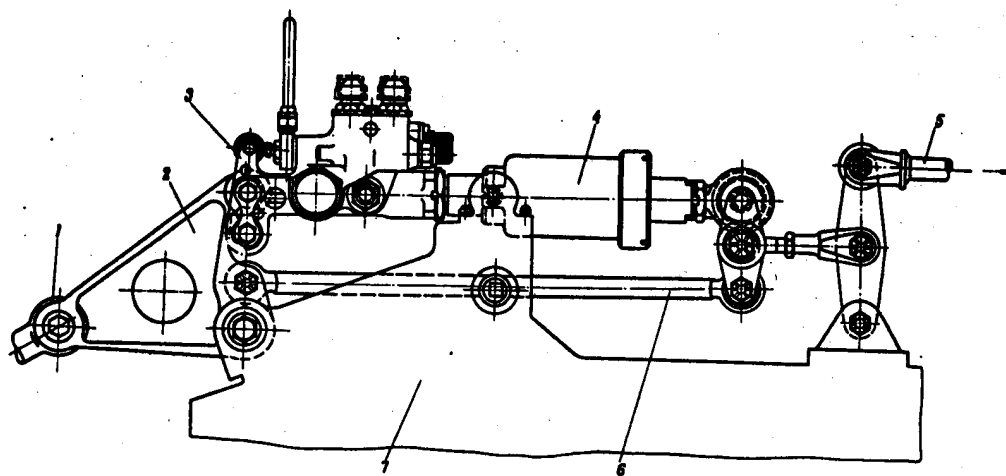


Fig.4. Booster Operation Diagram (position with aircraft control stick motionless and slide valve in neutral)
1-slide valve; 2-cylinder; 3-operating rod.



*Fig.5. Installation of Hydraulic Booster in Reversible System
1-rod from control stick (pedal); 2-bell crank of control system; 3- booster bell crank;
4-hydraulic booster; 5-rod to controlled surfaces; 6-reversible rod; 7-bracket.*

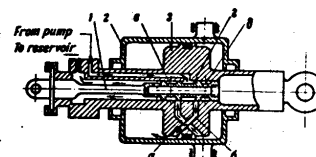


Fig. 6. Booster Operation Diagram (position with aircraft control stick moving to right and slide valve shifted to right hand position)
1 - slide valve; 2 - cylinder; 3 - operating rod.

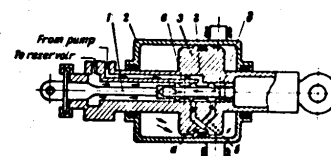


Fig. 7. Booster Operation Diagram (position with aircraft control stick moving to left and slide valve in left hand position)
1 - slide valve; 2 - cylinder; 3 - operating rod.

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the inner cavity of the slide valve along channel "a" through partially uncovered passage "3" of the slide valve and further through the radial ports into the return line.

When the control stick is shifted to the left, slide valve 1 (Fig.7) will also move to the left and uncover channels "a" and "6" on the right side; as a result, the right cavity of the cylinder will be connected with the delivery line, while the left cavity - with the return line. The operating rod (piston) will move to the left.

The hydraulic fluid will be pressed out of the left-hand cavity of the cylinder along inclined channels "6" into the inner cavity of the slide valve through uncovered passage "a" and then into the return line through the radial holes.

Consequently, the hydraulic booster will allow the pilot to overcome considerable loads originating from the ailerons or elevators of the plane, as the pilot controlling the aircraft has to apply but a minor effort to overcome friction in the hinges of the control linkage before the hydraulic booster and the friction between the slide valve and sleeve walls.

As is pointed out above, the travel, speed and direction of the movement of the ailerons or elevators are directly proportional to the travel, speed and direction of motion of the control stick or pedal. Let us now consider the process in detail.

Suppose, the slide valve is shifted 0.3 mm leftward relative to the operating rod. Under the action of the hydraulic fluid the operating rod will also start moving to the left, but it will move only till it covers the ports connecting passages "r" and "x" (See Fig.7) and channels "a" and "6", that is until the slide valve occupies the neutral position and separates passages "r" and "x" from channels "a" and "6". In other words, the operating rod similarly to the slide valve will move to the left for 0.3 mm.

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If we shift the slide valve through a certain distance to the right, the operating rod will also move to the right through the same distance, and will stop automatically. The construction of hydraulic boosters permits the slide valve to move relative to the operating rod only by several tenths of a millimetre, whereas the operating rod must be moved through tens and hundreds of millimetres. Consequently, in order to shift the operating rod, say, by 25 mm to the right with the relative slide valve travel of 0.5 mm, it is necessary to make 50 shifts of the slide valve.

In practice, however, depending on the speed of motion of the control stick or pedals, the slide valve uncovers automatically the ports, which ensures a precise coordination of the operating rod speed with the speed of motion of the control rod or pedals.

Thus, with a constant speed of motion of the control stick or pedals the slide valve will shift relative to the operating rod only at the beginning of the movement, after which it will move together with the operating rod as a single unit till the pilot stops moving the control stick or changes the speed of its travel.

If the motion of the control stick or pedals is not uniform, the ports created by the slide valve will decrease or increase in accordance with the change of speed of the control stick or pedal motion. Thus, a certain speed of motion of the control stick will determine the size of the port uncovered by the slide valve. If the speed of the control stick motion increases, the ports uncovered by the slide valve automatically increase too, and vice versa.

The size of the fully opened port ensures the maximum speed of the operating rod motion under the given load.

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III. TYPE EV-10 HYDRAULIC BOOSTERS

The hydraulic diagram of operation of these boosters differs from the operation diagram described above in that the EV-10 boosters are provided with a cross-feed system, a damping device, and a metering valve, and that they mount the distributing device on the head.

Hydraulic boosters EV-10B, EV-10M, EV-5A and EV-8A are various modifications of the EV-10 type. They differ from one another by certain design elements, assembly sizes and basic specifications depending on the type of aircraft or helicopter where they are installed.

Hydraulic boosters EV-10 are installed in helicopters; two hydraulic boosters incorporated in the irreversible system of pitch-throttle control, four hydraulic boosters - in the reversible system of lateral control, and four hydraulic boosters - in the reversible system of longitudinal control.

Hydraulic boosters EV-10B are also employed in helicopters; one hydraulic booster is installed in the reversible system of lateral control, one - in the reversible system of longitudinal control, and one - in the reversible system of tail rotor control.

Hydraulic booster EV-10M is installed in helicopters in the irreversible system of pitch-throttle control.

Hydraulic booster EV-5A is installed in aircraft in the irreversible system of ailerons control.

Hydraulic booster EV-8A is installed in aircraft in the reversible system of elevator control.

Hydraulic boosters EV-10B and EV-10M differ from hydraulic boosters EV-10 by the presence of drain and delivery automatic valves which supply the hydraulic boosters with the operating fluid from the duplicate system if the pressure in the main line drops to 30 kg/cm².

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Hydraulic boosters EV-10E and EV-10M differ from each other only by the position of the cylinder relative to the piston, and by the cross-feed pipes.

Hydraulic boosters EV-5A and EV-8A differ from hydraulic boosters EV-10 by the presence of an auxiliary cross-feed valve inside the operating rod and a return valve in the fluid feed pipe connection.

Hydraulic boosters EV-5A and EV-8A are almost identical and differ from each other only by the size of the cylinder and piston, angles of the swivel nipple turn and by certain technical data.

Of all the modifications of the EV-10 type, hydraulic boosters EV-10E and EV-10M are used most widely.

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1. Basic Specifications

Characteristics	EV-5A	EV-8A	EV-10, EV-10E, EV-10M
1	2	3	4
1. Type	Double-acting		
2. Operating principle	Hydro-mechanical		
3. Control system	Follow-up		
4. Maximum power	Not less than 3000 kg with $P = 90 \text{ kg/cm}^2$	1900 ± 100 kg. with $P = 120 \text{ kg/cm}^2$	Not less than 750 kg with $P = 60 \text{ kg/cm}^2$
5. Operating pressure at hydraulic booster	80 - 120 kg/cm^2	80 - 120 kg/cm^2	40 - 60 kg/cm^2
6. Operating rod travel	20.5 ± 1 mm	90 ± 1 mm	54 ± 1 mm
7. Swivel angle of operating rod	56 mm (4.9 mm on either side of neutral position)	85 mm (46 mm to the head, 39 mm to the fork)	50 mm (25 mm on either side of the neutral position)
8. Swivel angle of slide valve travel	2.3 - 2.6 mm	Not more than 1.9 mm	Not more than 1.9 mm

1	2	3	4
9. Distributing rod travel (at connection with aircraft bell crank)	1 - 1.2 mm	Not more than 0.65 mm	Not more than 0.85 mm
10. Dead zone of hydraulic booster:			
(a) along slide valve travel	Not more than 0.3 mm	Not more than 0.3 mm	Within 0.1 - 1 mm
(b) at connection with aircraft bell crank	Not more than 0.14 mm	Not more than 0.14 mm	Not more than 0.45 mm
11. Speed of operating rod travel under load	Not less than 80 mm/sec.		Not less than 70 mm/sec.
12. Friction and damping in distributing device (at connection with aircraft bell crank)	Not more than 2.1 kg. (a load of 2.1 kg must shift the slide valve and cause the beginning of piston motion within not more than 2 sec. from the extreme right position)		
13. Operating rod friction with hydraulic system switched off	Not more than 12 kg	Not more than 12 kg.	Not more than 12 kg
14. Hydraulic booster weight	Not more than 5.8 kg	Not more than 4.8 kg	EY-10E and EY-10M: not more than 5.1 kg, EY-10: not more than 3.2 kg

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1	2	3	4
15. Operating fluid	AMT-10	AMT-10	AMT-10
16. Operating temperature range (during hydraulic booster operation)	+60°C	+60°C	+60°C
17. Guaranteed service life	200 flying hours during 4 - 5 years	200 flying hours during 4 - 5 years	300 flying hours during 4 - 5 years

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2. Operating Principle of EV-10B and EV-10M Hydraulic Boosters

The distributing device of hydraulic boosters EV-10B and EV-10M is mounted inside the booster head and is essentially a slide valve (plunger) with four collars.

The operating fluid is supplied under pressure into the recess between the middle collars. When the slide valve is in the neutral position, the middle collars cover the cylinder delivery and return channels, thus locking the operating fluid in both cylinder cavities.

When the distributing slide valve moves in either direction from the neutral position, the channels become uncovered and the operating fluid is forced into one of the cavities of the cylinder and out of the other cavity into the return line.

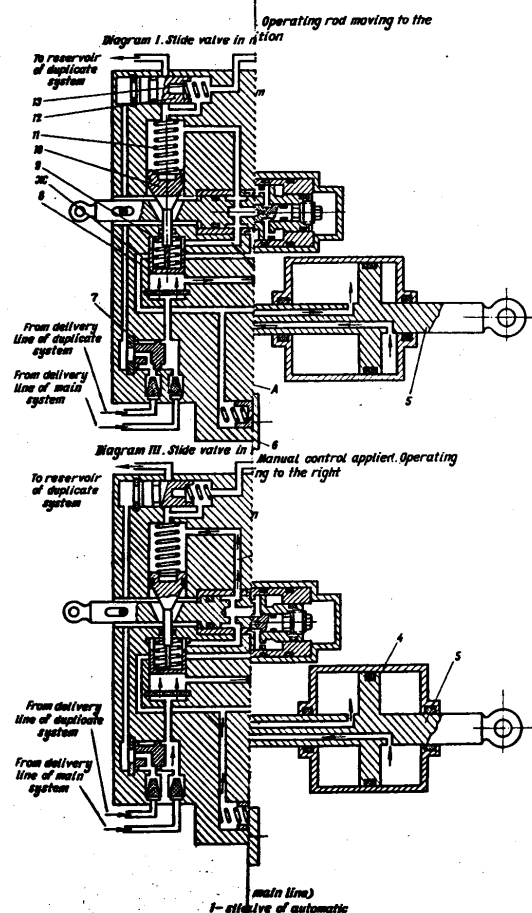
The damping device provided to ensure uniform operation of the slide valve protects the latter from vibrations that are likely to arise in the system.

The hydraulic booster head accommodates metering valve 6 (Fig. 8) which ensures smooth and uniform motion of the operating rod under load.

At the moment when the pressure in both cavities of the cylinder is balanced and the metering valve occupies a neutral position, the sealing band of the plunger cuts off the operating fluid supply.

If the pressure difference in the cylinder cavities is equal to or above $5 - 6 \text{ kg/cm}^2$, the plunger of the metering valve sends the operating fluid into the cavity with a higher pressure.

The hydraulic boosters are provided with an emergency cross-feed system of cylinder cavities and a locking device which are set in operation if the pressure in the hydraulic



Boosters

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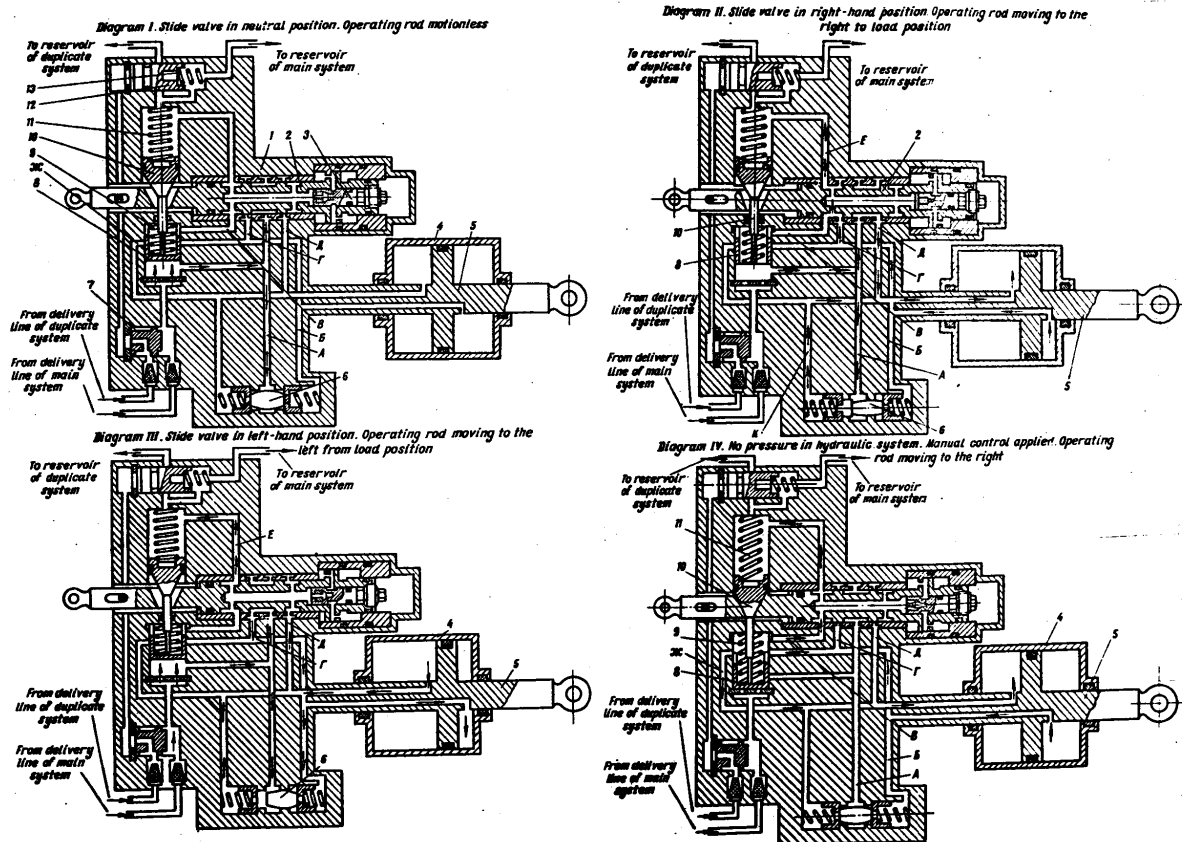


Fig. 8. BV-10B and BV-10M Hydraulic Boosters. Operation Diagram (position with operating fluid supplied from main line)
1 - slide valve body; 2 - slide valve; 3 - needle of dumping device; 4 - cylinder; 5 - operating rod; 6 - metering valve; 7 - slide valve of automatic delivery valve; 8 - plunger; 9, 11 and 12 - springs; 10 - stop; 13 - slide valve of automatic drain valve.

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system drops to zero. The locking device in this case prevents the possibility of the slide valve motion relative to the operating rod, otherwise it would result in the control stick or pedals switching over to idle motion.

Fig.8 illustrates the operation diagram of a hydraulic booster supplied from the main hydraulic system with different positions of the slide valve (distributing rod).

The operating fluid from the hydraulic system pump is supplied through the inlet pipe connection filter and the automatic delivery valve (AKII) to plunger 8 which moves upward and disconnects channels B and X communicating with the cylinder cavities. At the same time plunger 8 shifts up stop 10 and opens channel A. The operating fluid is forced through channel A to the middle recess of the slide valve and to metering valve 6.

When stop 10 moves up, the clearance in the taper connection of the slide valve and the stop that appears as a result of the stop motion permits the slide valve to shift relative to the operating rod for 1.9 mm.

Diagram I (See Fig.8) shows slide valve 2 in the neutral position; the middle collars of the slide valve cover channels A and I, the operating fluid is prevented from entering the cavities of cylinder 4, and operating rod 5 is motionless. So is metering valve 6.

Diagram II shows slide valve 2 right of the neutral position; channel A connects the left cavity of cylinder 4 with the middle recess of the slide valve, while channel I connects the right cavity of the cylinder with the extreme left recess of the slide valve and, through channel B, with the return line.

The operating fluid is forced under pressure from the middle recess through channel A into the left cavity of the cylinder and shifts the operating rod to the right of the neutral position. The operating fluid in the right cavity

is forced out of the cylinder into the main hydraulic system reservoir through channels B and F, left recess of the slide valve, channel E and the automatic drain valve.

Due to a pressure difference in the working cavities of the cylinder, the metering valve moves rightward in the direction of a lower pressure. The left cavity of the cylinder communicates with channel A through channel K, which permits additional quantities of the operating fluid to enter the left cavity from the delivery line, by-passing the distributing device.

In diagram III the slide valve is shifted to the left of the neutral position; channel F connects the right cavity of the cylinder with the middle recess of the slide valve, whereas channel A connects the left cavity of the cylinder with the right extreme recess of the slide valve, with channel E and with the return line.

The operating fluid is forced under pressure from the middle recess of the slide valve through channel F into the right cavity of the cylinder and shifts the operating rod to the left of the neutral position (load relief).

The operating fluid in the left cavity is forced out of the cylinder into the main hydraulic system reservoir through channel A, the recess and the hole in the slide valve, channel E and the automatic drain valve.

During the leftward motion of the operating rod to its middle position the pressure in the left cylinder cavity will exceed that in the right cavity. This excess pressure in the left cavity of the cylinder balances the total load resulting from the operating fluid pressure and the load transmitted from the control surfaces. Therefore, similarly to the previous case, the metering valve is shifted to the right, and the left cavity of the cylinder communicates directly with the delivery line. When the operating rod passes the middle position, that is when it starts moving leftward to build up load,

the metering valve will also shift to the left and connect the right cavity directly with the delivery line.

Diagram IV illustrates the case when the control is effected mechanically (manually) by means of the control stick or control pedals. The necessity to operate control surfaces mechanically will arise if there is no pressure in the hydraulic system. In this case the control surfaces are deflected by the physical effort of the pilot which is applied to the control stick or pedals and is transmitted through a system of rods.

Operating rod 5 functions as one of the rods in the control system. The play of the distributing slide valve, and, consequently, the idle motion of the control stick or pedals is eliminated by stop 10.

If the pressure in the hydraulic system drops to 15^{+5} kg/cm², spring 11 will force stop 10 to move down. As a result, the play in the taper connection of the slide valve and the stop will be taken up and the slide valve will be prevented from moving relative to the piston. At the same time plunger 8 under the action of spring 9 will move down and will open channels E and K, thus ensuring a free flow of the operating fluid from one cylinder cavity to the other and the removal of the excess fluid from both cavities into the return line through channel B.

Fig. 9 illustrates the operation diagram of hydraulic boosters EV-10B and EV-10M when the operating fluid is supplied to the hydraulic booster from the duplicate system.

In this case the operation of the hydraulic boosters is the same as in the case of supply from the main system. The pump of the duplicate system, with the main system under pressure, runs idle and delivers the operating fluid to the duplicate system reservoir.

If the pressure in the main system drops below 30 kg/cm², the TA-59 hydraulic unit and the automatic drain and delivery

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valves cut the hydraulic booster out of the main system and connect it to the duplicate system, with a result that the pressure in the duplicate system rises to 60 kg/cm^2 . The operating fluid from the duplicate system pump flows through the filter in the pipe connection and the hole in slide valve 7 of the automatic delivery valve into cavity N connected with cavity M through channel K. Under the action of the working pressure in cavities N and M, slide valves 7 and 12 are shifted to the extreme right position and cut off the supply of the operating fluid into the hydraulic booster and the return of the operating fluid into the main system, and connect the hydraulic booster with the supply and return lines of the duplicate system.

When the pressure in the main system rises to 60 kg/cm^2 , the pump of the duplicate system is automatically switched to the idle operation. Under the action of the working pressure slide valve 7 of the automatic delivery valve will be shifted to the left, thus disconnecting the hydraulic booster supply line from the duplicate system and connecting it with the main system. Spring 13 will shift slide valve 12 of the automatic drain valve to the left, disconnecting the return line of the hydraulic booster from the duplicate system and connecting it with the main line.

3. Operating Principle of EV-10 Hydraulic Boosters

Hydraulic boosters EV-10 (Fig.10) differ from the EV-105 and EV-10M modifications by the absence of automatic valves for switching over the booster to the duplicate system.

In EV-10 hydraulic boosters the operating fluid is supplied from the pump to the swivel nipple instead of the pipe connection of the automatic delivery valve. Otherwise,

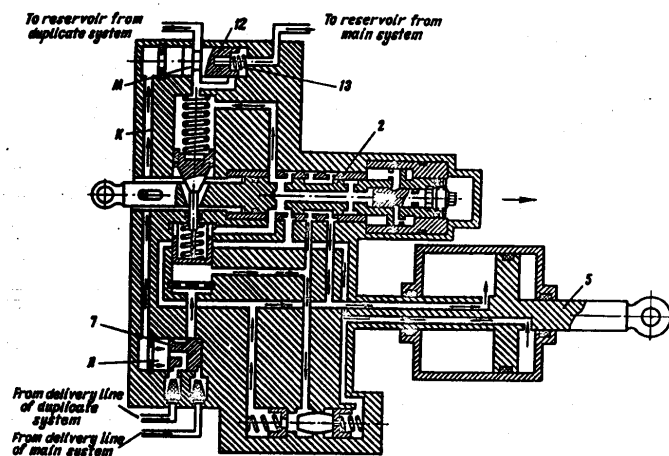


Fig.9. EV-105 and EV-10M Hydraulic Boosters. Operation Diagram (position with operating fluid supplied from duplicate system)
2 - slide valve; 5 - operating rod; 7 - slide valve of automatic delivery valve; 12 - slide valve of automatic drain valve; 13 - spring.

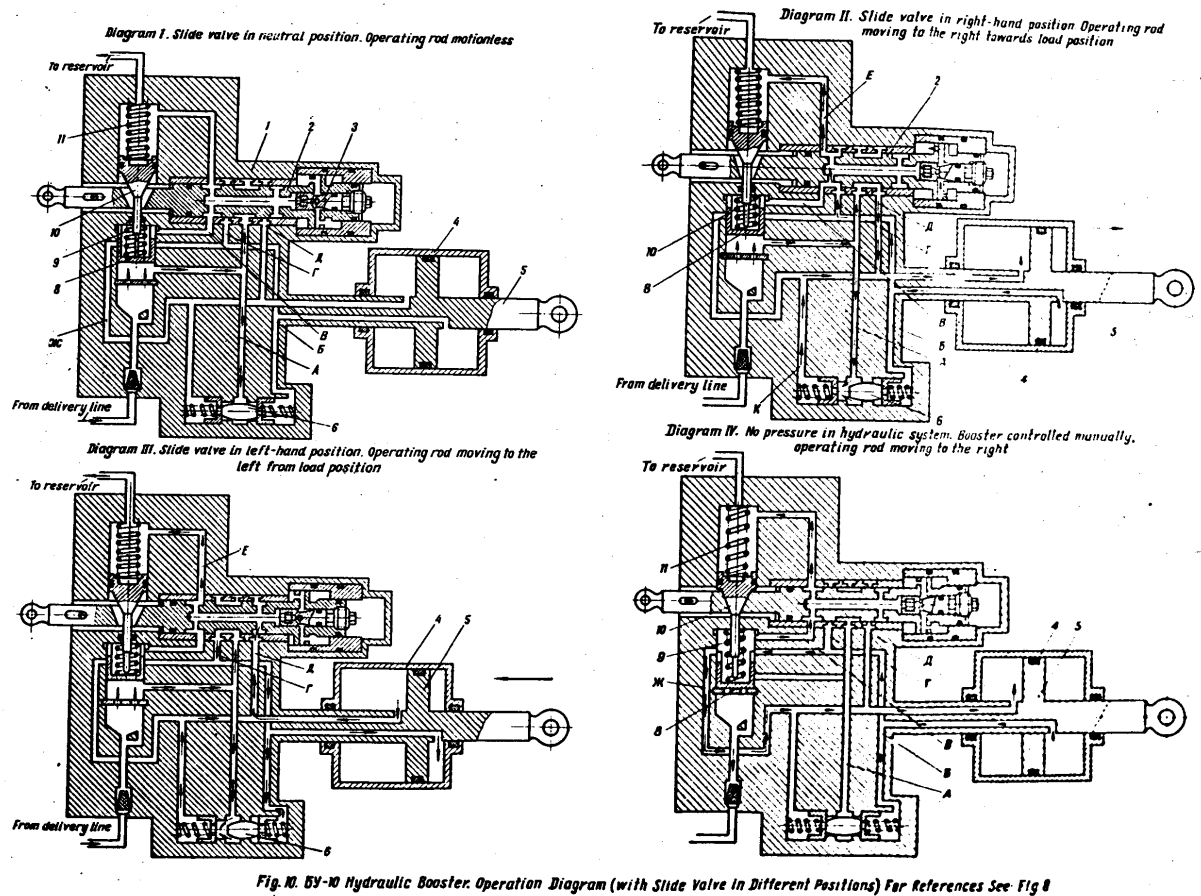


Fig. 10. 6Y-10 Hydraulic Booster. Operation Diagram (with Slide Valve in Different Positions) For References See Fig 8

- 25 -

the operation diagram of EV-10 hydraulic boosters is identical with that of EV-10E and EV-10M boosters (See Fig.8).

4. Operating Principle of EV-5A and EV-8A

Hydraulic Boosters

Hydraulic boosters EV-5A and EV-8A differ from hydraulic boosters EV-10E and EV-10M by the absence of automatic valves for switching over to the duplicate system. In these boosters the operating fluid is supplied from the pump to the swivel nipple instead of the automatic delivery valve pipe connections. Besides, the hydraulic diagram of these boosters differs by the presence of additional cross-feed valve 14 (Fig.11). Otherwise, the hydraulic diagram is identical with that of hydraulic boosters EV-10E and EV-10M. The latter modifications differ from each other only by certain parameters and by overall dimensions.

5. Description of EV-10 Booster Design

Fig.12 illustrates the external view of hydraulic boosters EV-10E. The assembly drawing is presented in Fig.13, while the external view of individual units and parts is presented in Fig.14.

Hydraulic booster EV-10E includes head 1 (See Fig.14), piston 2, cylinder 3 and other units and parts which serve for connection of the hydraulic booster with the rods of the control system and with the aircraft hydraulic system.

The head incorporates the distributing and damping devices, the cross-feed system, the metering valve, the filter and serves also for connecting control levers.

The head unit includes slide valve 1 (Fig.15), hinge 2, bell crank 3, automatic delivery valve 4 and automatic drain valve 5.

The slide valve unit is mounted on the body of the head and consists of slide valve body 1 (Fig.16), sleeve 2, slide valve 3 and the damping device.

Slide valve body 1 is a steel forging with press-fitted sleeve 2 which is a hollow bushing. On its outer surface the sleeve has four annular grooves. Milled in two grooves are two rectangular ports machined and finished to very close tolerances. Two other grooves have four drilled holes each.

The sleeve accommodates distributing slide valve 3 with the damping device.

Distributing slide valve 3 is a hollow rod with three recesses on its outer surface which form two service collars. The linear sizes of the collars are accurate within 0.02 mm.

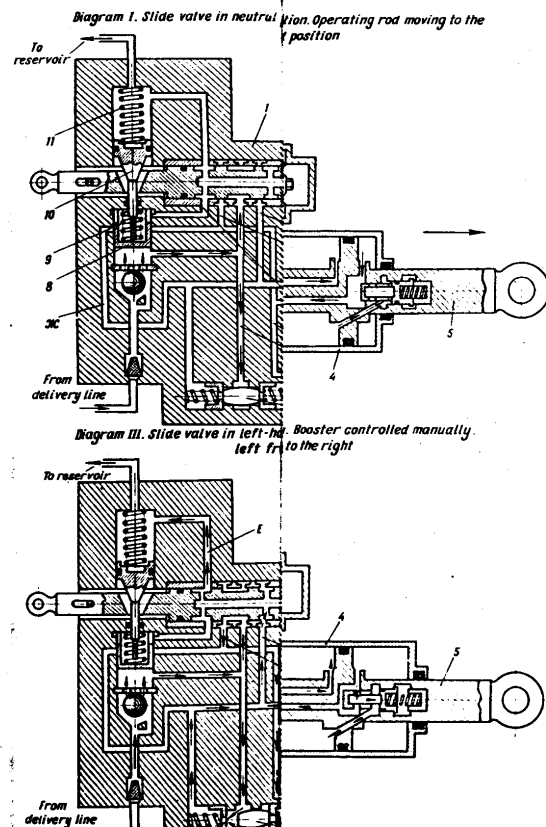
When the slide valve is in the neutral position, the service collars cover both ports in the sleeve simultaneously and symmetrically, and when the slide valve moves aside, they open both ports simultaneously. The mating surfaces of the slide valve and sleeve are machined to the tenth class of precision.

The right end of the slide valve is provided with a shoulder, 20 mm in dia., which serves as piston of the damping device. The damping device consists of the piston (the larger shoulder of the slide valve) and two damping cavities "a" and "b" which are formed by the slide valve body and the damper piston.

The body of damper 4 is secured to the slide valve body by means of four screws. The sealing between the damper body and the slide valve body is ensured with two sealing rings 5 inserted into the grooves on the outer surface of the damper body.

The needle unit is mounted in the slide valve recess. The outer surface of needle 7 is bevelled.

The damping cavities are interconnected by means of radial channels "c" and "f" in the damper piston and a taper



mounted on the body of the head
body 1 (Fig.16), sleeve 2, slide

steel forging with press-fitted
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in the slide valve recess.
is bevelled.
interconnected by means of
in the damper piston and a taper

Diagram I. Slide valve in neutral position. Operating rod motionless

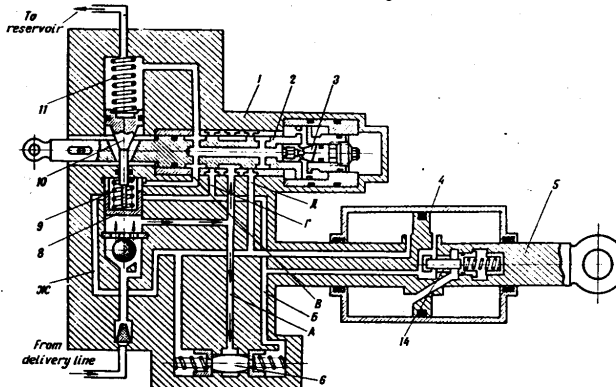


Diagram III. Slide valve in left-hand position. Operating rod moving to the left from load position

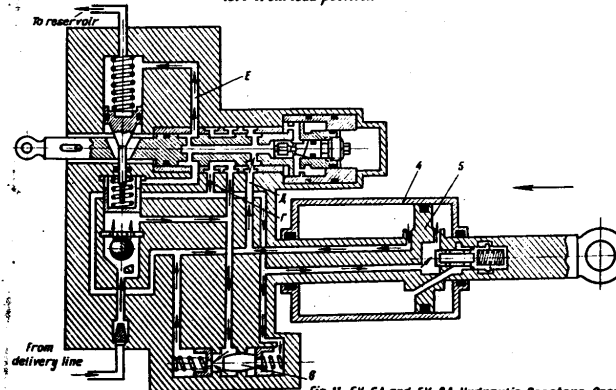


Diagram II. Slide valve in right-hand position. Operating rod moving to the right towards load position

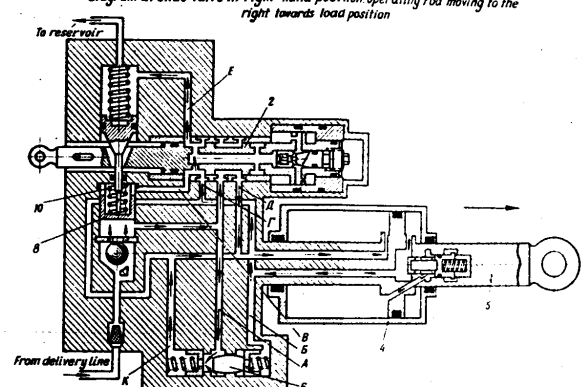


Diagram IV. No pressure in hydraulic system. Booster controlled manually. Operating rod moving to the right

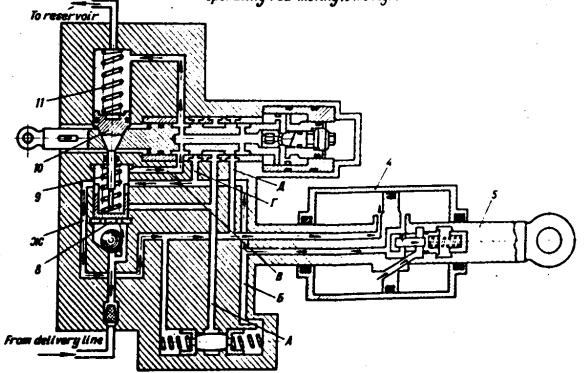


Fig.11. 6Y-5A and 6Y-8A Hydraulic Boosters. Operation Diagram (with Slide Valve in Different Positions)
For Reference See Fig. 8

14 - additional cross-feed valve

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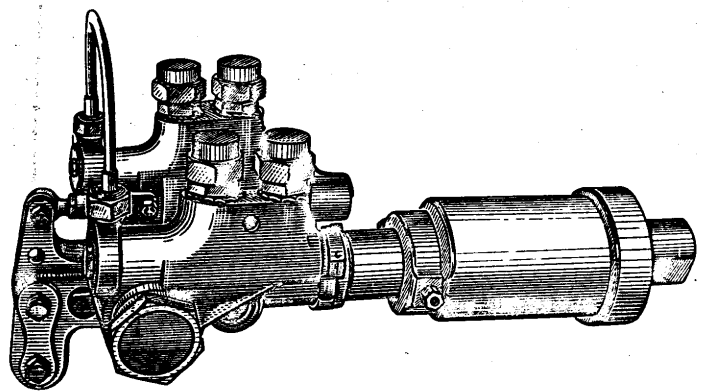


Fig.12. 6V-105 Hydraulic Booster. External View

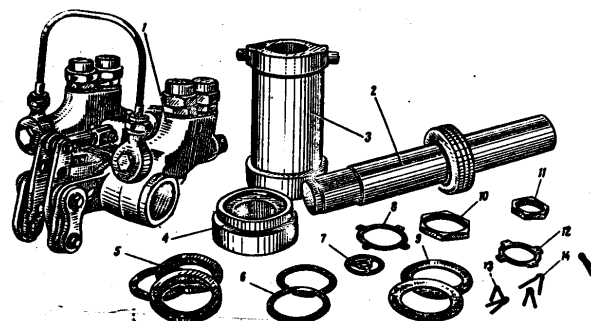


Fig. 14. Units and Parts of Hydraulic Booster 5Y-105
1 - head unit; 2 - piston unit; 3 - cylinder; 4 - cylinder bushing; 5 - cups; 6 and 9 - leather rings; 7 - gasket; 8 and 12 - locking rings; 10 - lock nut; 11 - nut; 13 and 14 - cotter pins.

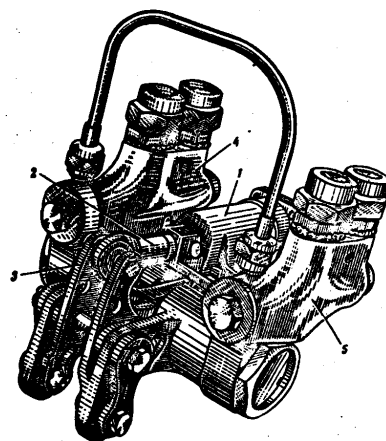


Fig. 15. Booster Head
 1—slide valve; 2—hinge; 3—best crank; 4—automatic delivery valve; 5—automatic drain valve.

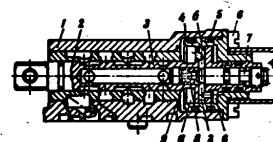


Fig. 16. Slide Valve
 1—slide valve body; 2—screw; 3—slide valve; 4—damper body; 5—sealing rings; 6—screws; 7—needle; 8—ball; 9—plug.

- 27 -

passage. The damping cavities are filled with fluid through a return valve mounted in the butt end of the needle and consisting of ball 8 and plug 9.

The damper adjustment consists in changing the clear opening of the channel connecting both damper cavities by turning the needle in or out. Upon completion of adjustment the needle position is fixed with a cotter pin passing through the slots on the end face of the slide valve and the holes in the needle.

The left end of the distributing slide valve terminates in a shank by means of which the slide valve unit is hinged to the bell crank unit.

Bell crank 1 (See Fig.13) linked with control surfaces by means of bolt 45 can rock around the axis of bolts 43 imparting reciprocal motion to the slide valve relative to the sleeve. The slide valve travel is made possible by the clearances between bushings 47 and head 26.

The rods running from the control stick or control pedals of the aircraft are connected by means of bolt 45 to the middle of the bell crank which ensures the aircraft rods and slide valve ratio of approximately 1:1 at the place of connection, i.e. a certain motion of the aircraft rod at the point of connection to the bell crank will cause the slide valve to move a double distance. This ensures a large overlap of the slide valve with a minimum idle travel of control system elements.

The slide valve moves only if there is pressure in the hydraulic system. If the pressure is absent, the stop fits tightly with its tapered section into the taper hole in the middle projection of the bell crank as a result of which the bell crank and consequently the slide valve are fixed in the neutral position relative to the head. In this case the effort from the control stick or control pedals will be transmitted through the bell crank, the stop and the head directly to the operating rod.

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The head unit has a complicated geometrical shape. Mounted in head 1 (Fig. 17) are stop 2 with plunger 3 and springs 4 and 11, plungers 6 and 9 and metering valve springs.

If the hydraulic system is cut off, stop 2 (See Fig. 17) moves under the action of spring 11 and locates bell crank 12 taking up the backlash (travel) of the slide valve. If there is no pressure in the hydraulic system, springs 4 and 11 shift plunger 3 in the upper position and ensure the connection of the cylinder cavities with each other and with the return line.

The metering valve (MK) consists of plungers 6 and 9 and springs 7. The operating fluid is supplied to the butt ends of plunger 9 from both cavities of the cylinder.

A difference of pressure in the cylinder cavities causes plunger 9 to move relative to the recess connected with the delivery line. As a result, the valve by-passes the hydraulic fluid into the corresponding cavity of the cylinder through the flats of plunger 9.

The difference of hydraulic fluid pressure in the cylinder cavities determines the length of the plunger travel, that is the size of passage for an additional supply of the fluid into the cylinder.

The cavities of the MK metering valve are closed at the faces with pipe connections 8. The joints between the head body and the pipe connections are sealed by rubber sealing rings inserted into the pipe connection grooves, and by aluminium washers.

Head 1 (See Fig. 13) is connected with the piston unit through a thread joint which is packed by aluminium washer 24 and rubber ring 22.

The piston unit is a power element which transmits the effort built up by the operating fluid pressure to the controlled units through a system of levers.

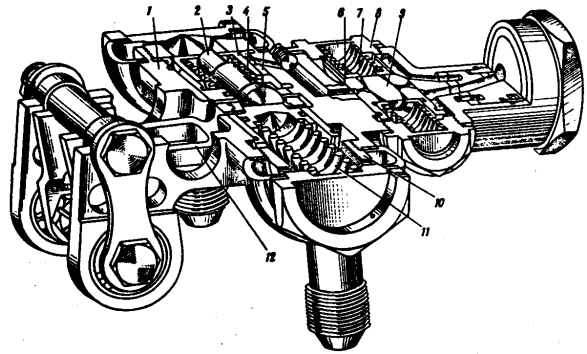


Fig. 17. Booster Head Longitudinal Section
1-head; 2-stop; 3-plunger; 4-spring; 5-sleeve; 6-plunger; 7-metering valve spring; 8-pipe connection; 11-spring; 12-bell crank.

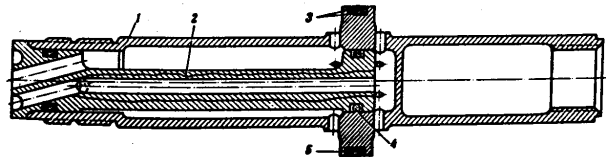


Fig. 18. Piston
1-piston; 2-distributor; 3-leather rings; 4 and 5-sealing rings.

Piston 1 (Fig.18) is a hollow rod with a bulged portion in the middle which is known as the piston head. The groove in the piston head accommodates rubber ring 5 and two leather rings 3 which separate the cylinder cavities.

The surfaces of the piston rods are machined and finished to very close tolerances. To improve the wear-resistant and anti-corrosive qualities of the rod, its surfaces are chrome-plated.

On the side of the head unit the piston cavity houses distributor 2 which permits each of the service collars of the slide valve to supply the hydraulic fluid or to return it from the corresponding cavity of the cylinder.

Distributor 2 is a duralumin rod with longitudinal and oblique ports.

The tightness of the distributor-to-piston joint is ensured by rubber sealing rings 4 (of a round section) which are inserted into special grooves on the external surface of the distributor.

Ear 62 (See Fig.13) is screwed into the other end of the piston. The ear serves to link the hydraulic boosters with the rods connected with the control units.

The piston unit is located in cylinder 11 which is a three-step bushing with two trunnions and the outer surface which serve for the installation of the hydraulic booster on the ball supports of the aircraft bracket.

The working surfaces of the cylinder are machined and finished to close tolerances. The cylinder bore of the smaller diameter is a guide for the operating rod.

The cylinder bore of the larger diameter receives cylinder nut 18 whose inner hole also serves as a guide for the operating rod.

The necessary tightness is ensured by two packing units inserted into the cylinder and nut grooves; one of the units consists of a gland, tongued cup 16 and leather ring 17, while

- 30 -

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the other unit includes round rubber ring 15 and two leather rings 14.

The hydraulic fluid is supplied to and returned from the hydraulic booster through flexible hoses which are connected with the head through an automatic delivery valve (AKH) and automatic drain valve (AKC).

The necessary tightness of the joints between the valves and the hydraulic booster head, as well as between the valves and pipe connections 29 and 40, is ensured by rubber sealing rings 39 and aluminium gaskets 38.

An additional cleaning of the hydraulic fluid is done by screen filters 56 mounted in pipe connections 50 of the automatic delivery valve.

The hydraulic fluid is supplied under pressure to plunger 28 through automatic delivery valve pipe connection 50 and pipe connection 29 and presses off the plunger together with stop 27, thus opening access to channel "a".

Through channel "a" the hydraulic fluid is fed to the middle recess of slide valve 5 and to metering valve 34.

When the slide valve shifts to the left, the hydraulic fluid through channel "a" and metering valve cavity "g" is delivered into channel "r" running to distributor 10 and through channel "r" to the right cavity of the cylinder.

The operating fluid is pressed out of the left cavity of the cylinder through the cavity between the distributor and the piston rod into channel "s", the recess of the metering valve and channel "a" to the slide valve, and further through the holes in the slide valve and channel "m" to the return line.

When the slide valve shifts to the right, the hydraulic fluid is forced into the left cavity of the cylinder and driven out of the right cavity. The cross feed of the cylinder cavities is ensured through channels "r" and "m", and connection of the cavity with the return line - through channel "s".

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The automatic delivery and drain valves are identical in construction and differ from each other by the presence of a spring in the drain valve, the presence of filters in the delivery valve pipe connections and by the slide valve design.

The automatic delivery and drain valves consist of bodies 1 (Figs 19 and 20), sleeves 2 with rubber seals around the outer diameter and slide valves 3.

The delivery and drain slide valves have three collars each; two of them serve to distribute the operating fluid, whereas the third one is the slide valve travel limiter. In the automatic drain valve, if there is no pressure supplied from the main hydraulic system, the slide valve is held by means of a spring in the extreme left position. In this case the hydraulic booster is connected with the return line of the main hydraulic system.

The threaded holes of the bodies receive plugs 8 and pipe connections 11. Swivel nipples 12 fitted on the pipe connections are joined by a pipe which supplies the operating fluid from the delivery line of the duplicate system for switching over the drain slide valve to the right position. In this position the hydraulic booster is connected with the return line of the duplicate system.

The pipe lines of the main and duplicate systems are connected to the pipe connections screwed into the bodies of the automatic delivery and drain valves.

Pipe connections 4 (See Fig. 19) of the automatic delivery valve carry screen filters 6 for an additional cleaning of the operating fluid.

Hydraulic booster EV-10M (Fig. 21) differs from the EV-10S modification by the shape of cross-feed pipe 48 (Fig. 22) and by the position of cylinder 11 relative to head 26.

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Hydraulic booster EY-10 (Fig.23) differs from hydraulic booster EY-10B in that it has no automatic delivery and drain valves.

The operating fluid is supplied to, and returned from, the hydraulic booster EY-10 through flexible hoses connected to the head by means of nipples 51 and 49 (Fig.24).

Hydraulic boosters EY-5A and EY-8A (Fig.25) differ from hydraulic boosters EY-10 by the presence of an additional cross-feed valve and a return valve in the fluid-feed pipe connection, and by the construction of the distributing rod slide valve which has no flats on the collars (to improve speed characteristics).

The additional cross-feed valve is located in a special recess of the piston and consists of plunger 18 (Figs 26 and 27), spring 17 and plug 16.

If there is no pressure in both cavities of the cylinder, the spring shifts plunger 18 into the extreme left position, and the cylinder cavities communicate with each other.

The delivery of the operating fluid under pressure into one of the cavities of the cylinder causes plunger 18 to move to the extreme right position and to disconnect the cylinder cavities.

Plug 16 is screwed into the piston opening and is locked with a spring ring.

Return valve ball 52 mounted in fluid-feed pipe connection 40 serves for damping the operating fluid forced out through throttle hole "a" in case of a sudden drop of pressure in the delivery line; this weakens the shock on the control stick and ensures a more smooth transition to the manual control during the hydraulic booster operation under load.

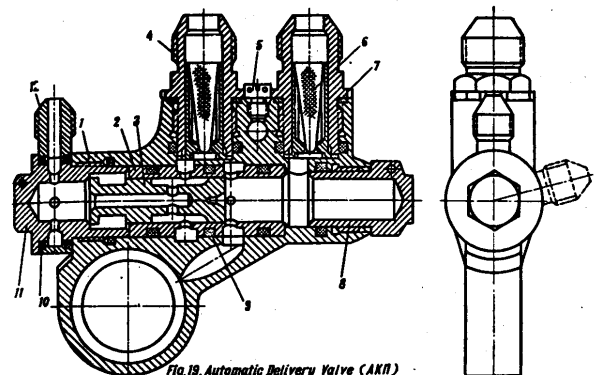


Fig.19. Automatic Delivery Valve (AKB)
1-body; 2-sleeve; 3-delivery slide valve; 4-pipe connection; 5-screw; 6-filter; 7-gasket;
8-plug; 9 and 10-sealing rings; 11-pipe connection; 12-swivel nipple.

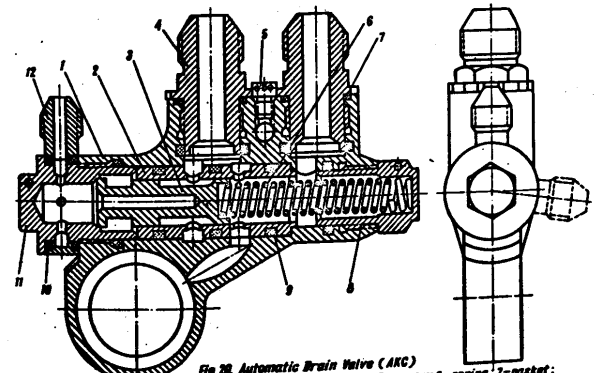


Fig.20. Automatic Drain Valve (AKD)
1-body; 2-sleeve; 3-delivery slide valve; 4-pipe connection; 5-screw; 6-spring; 7-gasket;
8-plug; 9 and 10-sealing rings; 11-pipe connection; 12-swivel nipple.

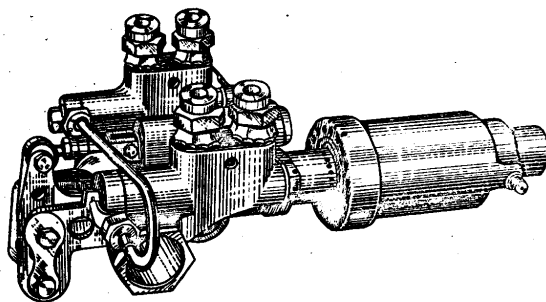


Fig. 21. 6Y-10M Booster. External View



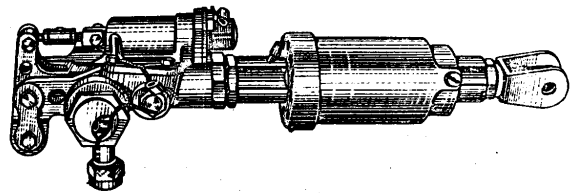
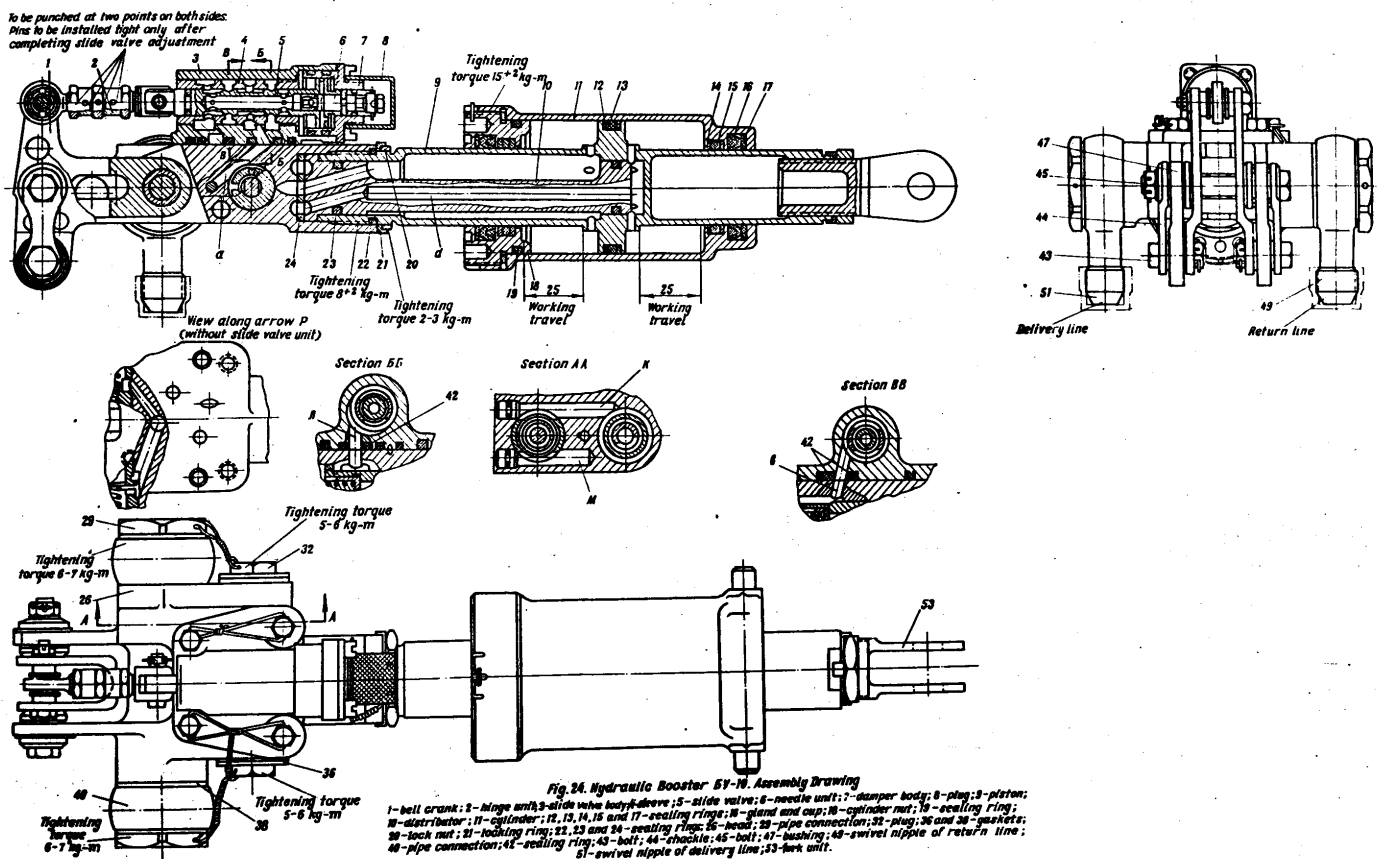


Fig. 23. 5Y-10 Booster. External View.



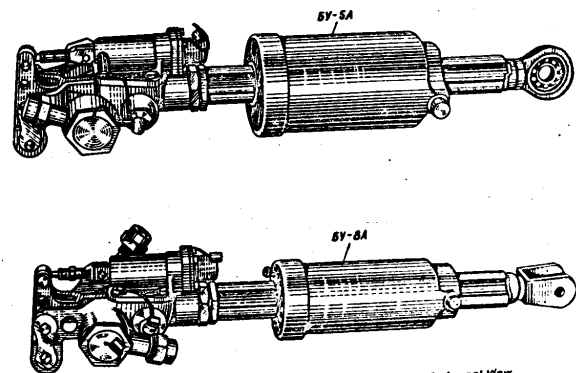
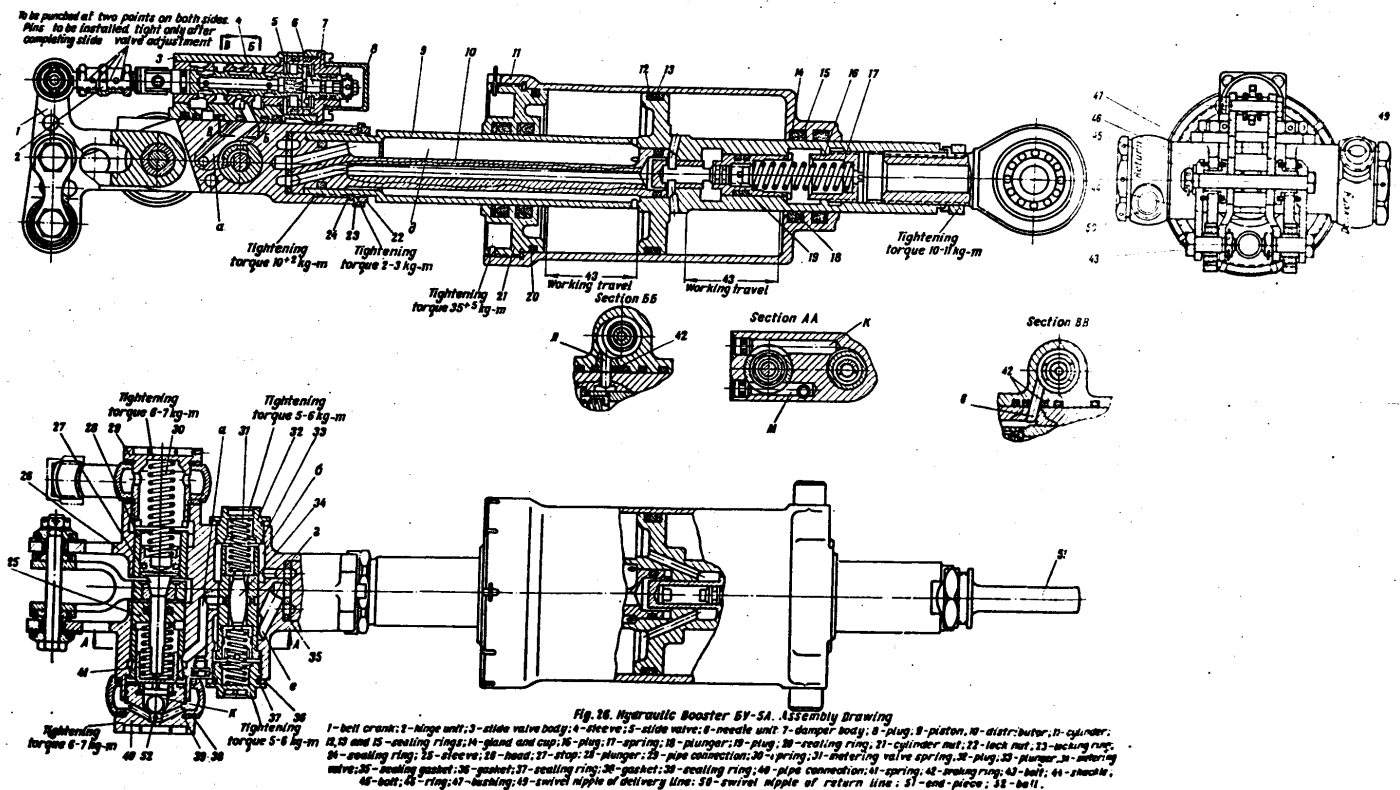
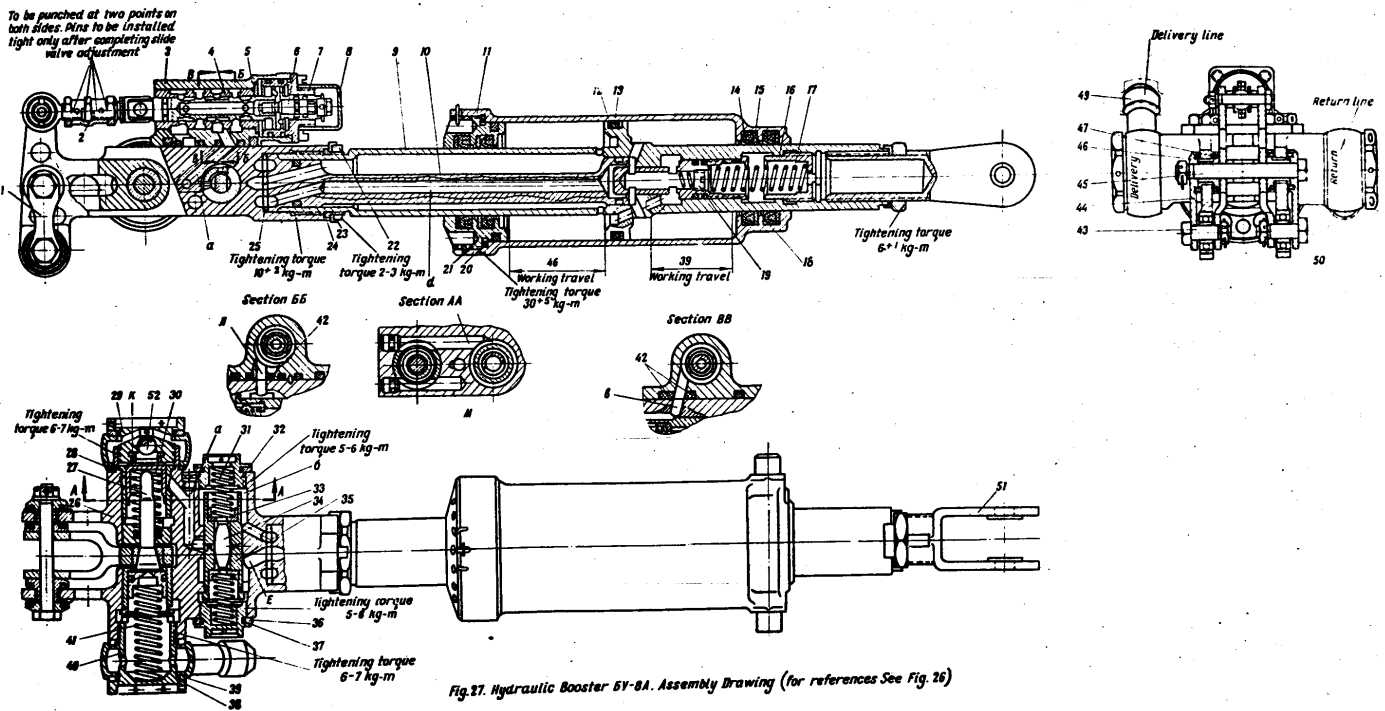


Fig. 25. Hydraulic Boosters 6V-5A and 6V-8A. External View





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IV. TYPE EV-13M HYDRAULIC BOOSTERS

Hydraulic boosters EV-13M, EV-14M and EV-14MC are various modifications of one and the same type. They differ from one another by certain design elements, power cylinder sizes, certain parameters and installation sizes.

The hydraulic boosters of this group are installed in aircraft in the irreversible control system: boosters EV-13M are installed in the aileron control line, boosters EV-14M - in the elevator control line and booster EV-14MC - in the stabilizer control line (the latter modification is mounted in aircraft fitted with controlled stabilizers).

Hydraulic boosters EV-13M and boosters EV-14M differ from each other by the sizes of cylinders; boosters EV-14MC differ from boosters EV-14M by the sizes of cylinders and by the travel of the main and duplicate slide valves.

A characteristic feature of the EV-13M hydraulic boosters is the presence of a duplicate distributing slide valve which ensures normal operation of the unit in case of jamming of the main slide valve, prevents a spontaneous creeping of the control stick and provides for aircraft control.

When the hydraulic booster operates from the duplicate slide valve, the aircraft control is somewhat different comparing with the hydraulic booster operation from the main slide valve, as in this case the effort required for shifting the duplicate slide valve should be greater. Besides, depending on the position of the jammed slide valve this effort is not equal when the slide valve moves to the right or to the left. The speed of motion of the operating rod may also be different in some cases and may be slightly less than during operation of the main slide valve.

1. Basic Specifications

Characteristics	EV-14M and EV-14MC	EV-13M
1	2	3
1. Type	Double-acting	
2. Operating principle	Hydro-mechanical	
3. Control system	Follow-up	
4. Maximum power built-up by hydraulic booster	4,600 $\pm$ 100 kg with P = 130 kg/cm ²	1,100 $\pm$ 50 kg with P = 130 kg/cm ²
5. Operating pressure in hydraulic system	85 $\pm$ 5 - 135 $\pm$ 5 kg/cm ²	
6. Operating fluid temperature	60 - 70°C (with short-time rise to +90°C for 10 min. after every 2 hours) 90 $\pm$ 1 mm	60 - 70°C (with short-time rise to +90°C for 10 min. after every 2 hours) 50 $\pm$ 2 mm
7. Working travel of operating rod	for EV-14M: 2 $\pm$ 0.05 mm; for EV-14MC: 1.5 $\pm$ 0.05 mm	2 $\pm$ 0.05 mm
8. Distributing slide valve travel (main and duplicate)	for EV-14M: 2 $\pm$ 0.05 mm; for EV-14MC: 1.5 $\pm$ 0.05 mm	2 $\pm$ 0.05 mm
9. Main and duplicate slide valve travel (at connection with aircraft bell crank)		

1	2	3
10. Dead zone of hydraulic booster along main slide valve travel and at connection with aircraft bell crank	Not more than 0.1 mm	
11. Speed of operating rod travel without load at pressure P = 120 - 140 kg/cm ²	Not less than 90 mm/sec.	Not less than 100 mm/sec.
12. Operating rod friction with hydraulic system cut off	Not more than 15 kg (a weight of 15 kg must shift the operating rod through the whole length of its travel within not more than 3 sec.)	
13. Main slide valve friction	Not more than 0.8 kg	
14. Effort required for shifting duplicate slide valve (effort required for overcoming spring resilience plus duplicate slide valve friction with main slide valve stopped)	Not less than 15 kg and not over 20 kg (a weight of 15 - 20 kg must shift the slide valve and cause the beginning of piston motion within not more than 3 sec. from extreme positions)	

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2. Operating Principle

All the modifications of EV-13M type are based on the same hydraulic operation diagram (See Fig. 28).

The distributing device is located in the head unit and consists of main slide valve 1, duplicate slide valve 2 and sleeve 3. The main slide valve is housed inside the duplicate slide valve which performs the role of a sleeve during normal operation of the main slide valve.

The duplicate slide valve remains motionless as long as the friction between the main and the duplicate valves does not exceed the resilience of spring 5 which is equal to 16 kg. If the main slide valve gets jammed, that is when the friction between the main and the duplicate slide valves exceeds the effort required for shifting the duplicate slide valve, the latter moves together with the main valve as one unit. Motion is imparted to the slide valve by control rod 13 through bell crank 12. The length of the slide valve travel is determined by the size of the clearance between bolt 14 and the head hole.

Hydraulic booster operation from the main slide valve.

The hydraulic fluid is supplied from the hydraulic system pump to the middle recess of slide valve 1 through a pipe connection and return ball valve 11 along channel A. When the main and the duplicate slide valves are in the neutral position, as shown in diagram I, the middle collars of the slide valves cover the channels which supply fluid to, and take it out of, the cylinder. In this case the operating rod remains motionless in spite of the fact that the hydraulic fluid is delivered under pressure to the distributing device of the hydraulic booster.

Then the slide valve moves to the left as shown in diagram II, the fluid from the middle recess is delivered into the left cavity of the cylinder along channel B through

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1	15. Hydraulic booster weight	EV-14M: 7.2 kg; EV-14MC: 7.6 kg	AMT-10
2	16. Operating fluid		
3		5 kg	AMT-10

the passage between the operating edge of the left middle collar of the main slide valve and the working edge of the inner grooves in the duplicate slide valve and through the permanent passage which is formed by the outer groove on the duplicate slide valve and the inner groove on the sleeve. The pressure of the working fluid causes the piston to move to the right. In its movement the piston forces the fluid from the right cavity into the return line along channel E through the permanent passage formed by the inner groove of the sleeve and the outer groove on the duplicate slide valve and through the newly created passage between the operating edge of the right middle collar of the main slide valve and the inner groove in the duplicate slide valve, and through the holes in the sleeve and the head.

When the slide valve moves to the right, as illustrated in diagram III, the hydraulic fluid from the middle recess flows into the right cavity of the cylinder along channel E through the newly created passage between the working edge of the right middle collar of the main slide valve and the working edge of the inner groove of the duplicate slide valve and through the permanent passage formed by the outer groove on the duplicate slide valve and the inner groove in the sleeve. The pressure of the hydraulic fluid causes the piston to move to the left and to force the fluid from the left cylinder into the reservoir through channel B.

Hydraulic booster operation from the duplicate slide valve is illustrated in Diagram IV. The main slide valve may become jammed in any point of its travel, including the extreme positions. The most typical positions of the jammed main slide valve are the extreme positions (right or left) and the middle position.

If the main slide valve gets jammed in the neutral position, the operation of the duplicate slide valve does not differ from the operation of the main slide valve except that

Diagram I. - am V. No pressure in hydraulic system. Booster controlled manually, operating rod moving to the left

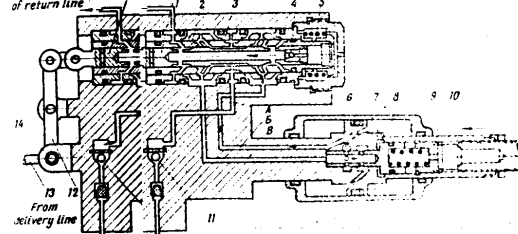
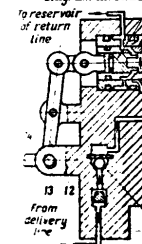


Diagram III. Slide



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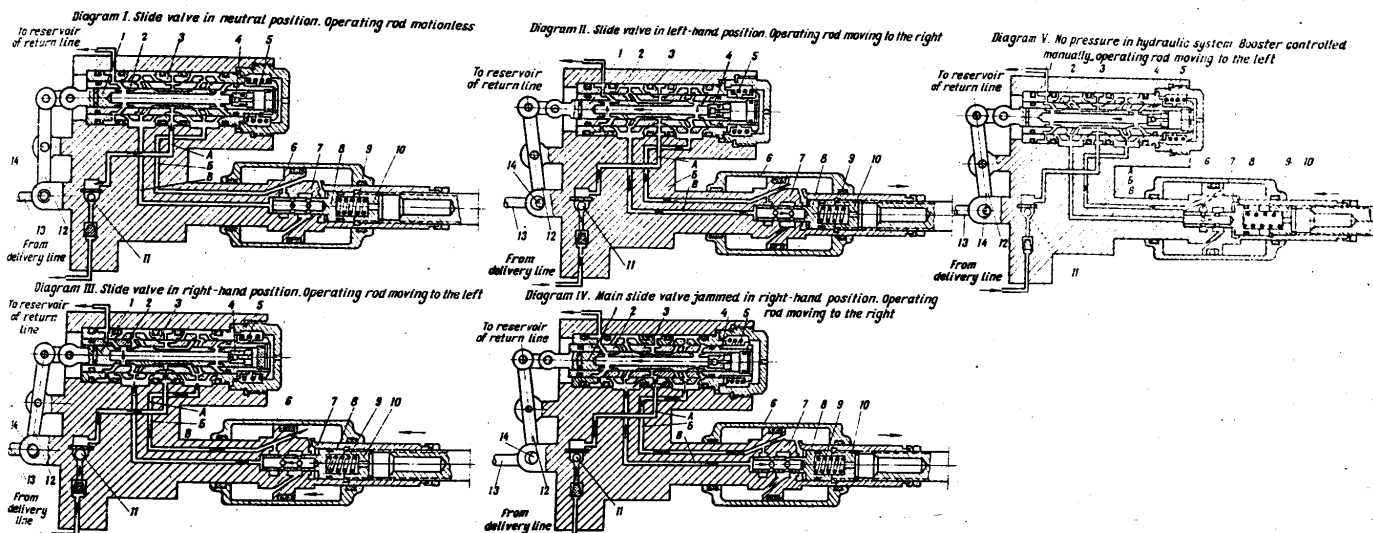


Fig. 28. Hydraulic Booster 5Y-13M. Operation Diagram
1 - main slide valve; 2 - duplicate slide valve; 3 - sleeve; 4 - damper; 5 - spring; 6 - cylinder; 7 - piston; 8 - cross-feed valve plunger; 9 - plug; 11 - ball valve; 12 - bell crank; 13 - control rod; 14 - bolt.

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the shifting of the duplicate valve requires a greater effort and the movement of the operating rod will be slightly slower.

In this case the slide valve travel in both directions will be symmetrical and both the delivery and drain channels will be covered symmetrically.

If the main slide valve gets jammed in one of the extreme positions, the hydraulic booster control will be somewhat different. To keep the booster piston immovable it is necessary to apply a certain effort to the control stick since the spring of the duplicate slide valve is compressed.

A deviation of the control stick in one direction is accompanied by the increase of the counteracting force. Contrary to that, when the control stick moves in the opposite direction, a certain effort must be applied to slow down its motion, as the spring of the duplicate slide valve tends to return to the initial position.

The speed of motion of the operating rod in one direction is determined by the size of the passage which is opened by the duplicate slide valve, whereas the speed of motion of the operating rod in the opposite direction is determined by the size of the passage opened by the main slide valve. These speeds therefore can be different.

By way of an example let us consider a case when the main slide valve is jammed in one of the extreme positions during hydraulic booster operation. If the slide valve gets jammed in the extreme right position, the supply of the hydraulic liquid into the working cavity of the cylinder and the return of the fluid from the other cavity will take place as shown in diagram III (see Fig. 25), and the piston will move to the left. The maximum speed of the piston motion in this case will be determined by the size of the passage uncovered by the main slide valve.

- 40 -

To stop the motion of the piston to the left it is sufficient to stop the motion of the control stick. The effort applied with this purpose to the control stick is transmitted through the rods and bell crank 12 (See Fig.28) to the main slide valve and consequently to the duplicate slide valve. As a result, the duplicate slide valve compressing spring 5 will move leftward till it covers the permanent passages formed by the outer grooves of the duplicate valve and the inner grooves of the sleeve. This will cut out the supply of the operating fluid to the working cavity of the cylinder and the return of the operating fluid from the idle cavity, and the piston motion will cease. Due to the fact that the spring tends to assume its initial position exerting an effort of 13 - 16 kg, it is necessary to apply an equal force to the stick in order to retain it in the stationary position.

During operation of the hydraulic booster from the main slide valve this effort is practically equal to zero.

To ensure rightward motion of the piston, the duplicate slide valve should be moved to the left till it uncovers a passage formed by the central recesses on the duplicate slide valve and the grooves in the sleeve, and a passage formed by the extreme recesses on the duplicate slide valve and the extreme grooves in the sleeve (See Diagram IV, Fig.28). Thus, the maximum speed of the piston motion to the right is determined by the size of the passages formed by the outer grooves on the duplicate slide valve and the inner grooves on the sleeve, whereas the speed of the leftward motion is determined by the size of the passages formed by the recesses on the main slide valve and the inner grooves on the duplicate slide valve.

With the passages opened to a full capacity the speed of piston motion in both directions is practically the same.

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If the main slide valve gets jammed in an intermediate position (between one of the extreme positions and the neutral position), maximum speeds of piston motion in one direction may be different from those in the other direction.

Should the main slide valve get jammed in one of the extreme positions, shifting the control stick in one direction will require an effort of 16 - 20 kg multiplied by the gear ratio, whereas shifting the control stick in the opposite direction will require a counteracting force of approximately the same value.

Hydraulic booster operation with manual control is illustrated in Diagram V (See Fig.28). The need of the manual control arises if there is no pressure in the hydraulic system. In this case the controlled elements are deflected by the physical effort of the pilot applied to the control stick and transmitted through a system of rods.

A change-over to the manual control due to the absence of operating pressure in the hydraulic booster is made possible by a cross-feed valve which ensures transfer of the hydraulic liquid from one cavity to the other depending on the piston motion in the cylinder.

Diagram V illustrates operation of the cross-feed valve. When the pressure in the cylinder cavities drops to $5 \pm 0.5 \text{ kg/cm}^2$, cross-feed valve plunger 8 under the action of spring 9 will move to the extreme left position. As the piston moves to the right, part of the operating fluid will be forced out into the reservoir along channel 5 through the distributing slide valve, and the remaining amount of liquid will freely flow from the right cavity of the cylinder into the left one. When the piston moves leftward, the operating fluid will flow along the same channels from the left cavity to the right cavity.

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As pressure is fed into one of the cylinder cavities, the cross-feed plunger, overcoming the spring resilience will shift to the extreme right position, thus disconnecting the cylinder cavities and ensuring normal operation of the hydraulic booster under operating pressure.

3. Description of Design

The EV-13M hydraulic booster is shown in Fig.29 (external view) and Fig.30 (assembly drawing).

Hydraulic boosters EV-14M (Fig.31) are connected to the aircraft controlled units by means of ear 15 (Fig.32) instead of a fork in the case of hydraulic boosters EV-13M. The same purpose in hydraulic boosters EV-14M (Fig.33) is served by end-piece 15 (Fig.34) which is different by its design from either the fork or the ear of the hydraulic boosters mentioned above.

The type EV-13M hydraulic boosters differ from the EV-10 type boosters by the design of the head unit. All other units and parts are identical with the corresponding units and parts of hydraulic boosters EV-10. Therefore, the description of the booster EV-13M construction is confined only to that of the head unit.

The head of the EV-13M hydraulic booster serves to join the piston with the aircraft control lever system and to accommodate the distributing device.

The head of the booster houses sleeve 2 (Fig.35) with distributing slide valves: duplicate slide valve 3 and main slide valve 4. Sleeve 2 is a hollow bushing with a thrust shoulder at the right end. The outer surface of the sleeve is provided with four annular recesses separated by five grooves of rectangular section which accommodate sealing rings.

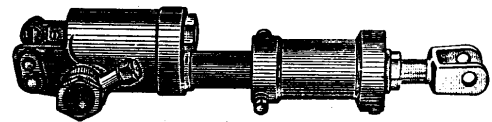
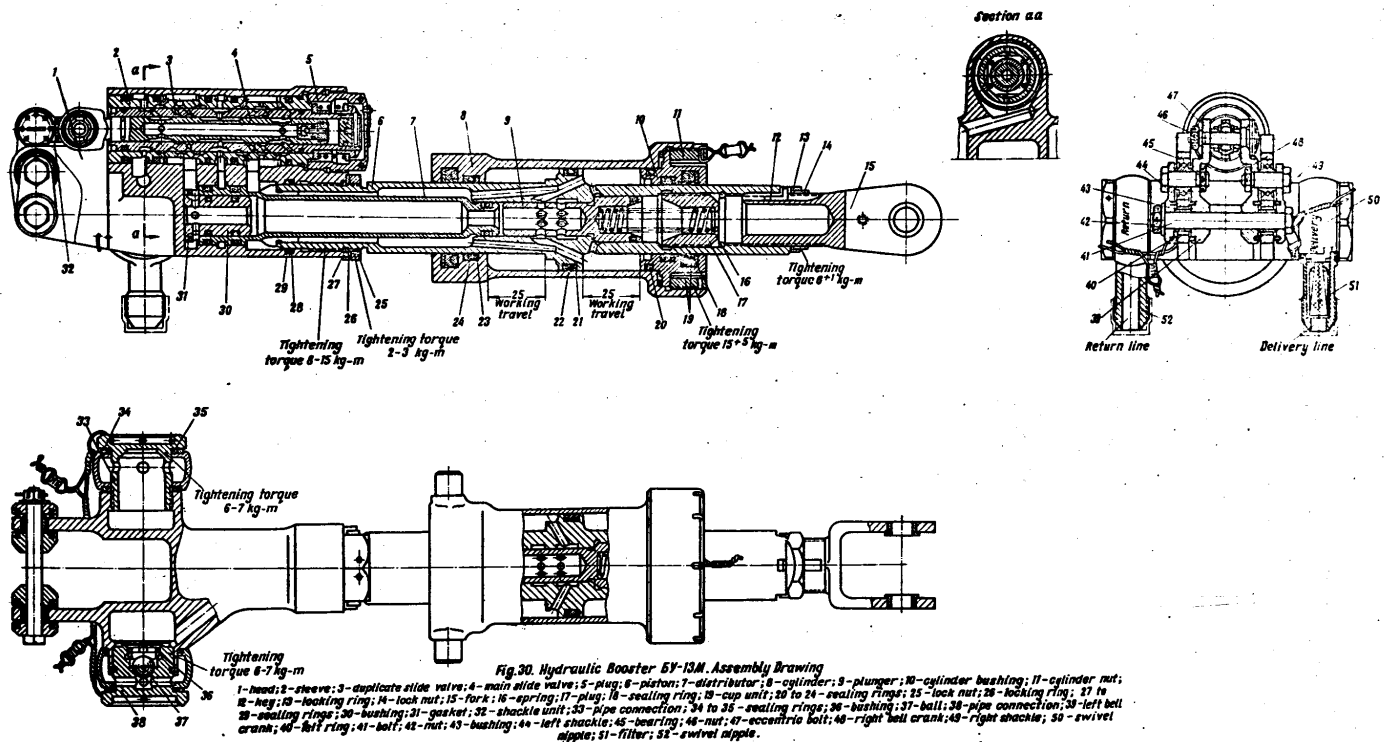


Fig.29. EV-13M Booster. External View



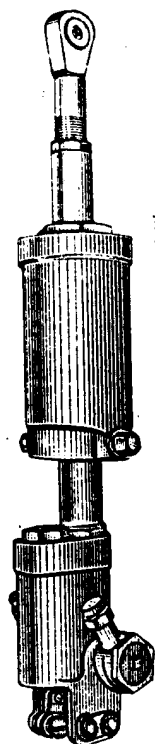


Fig. 31. 6Y-14M Booster. External View

Section aa

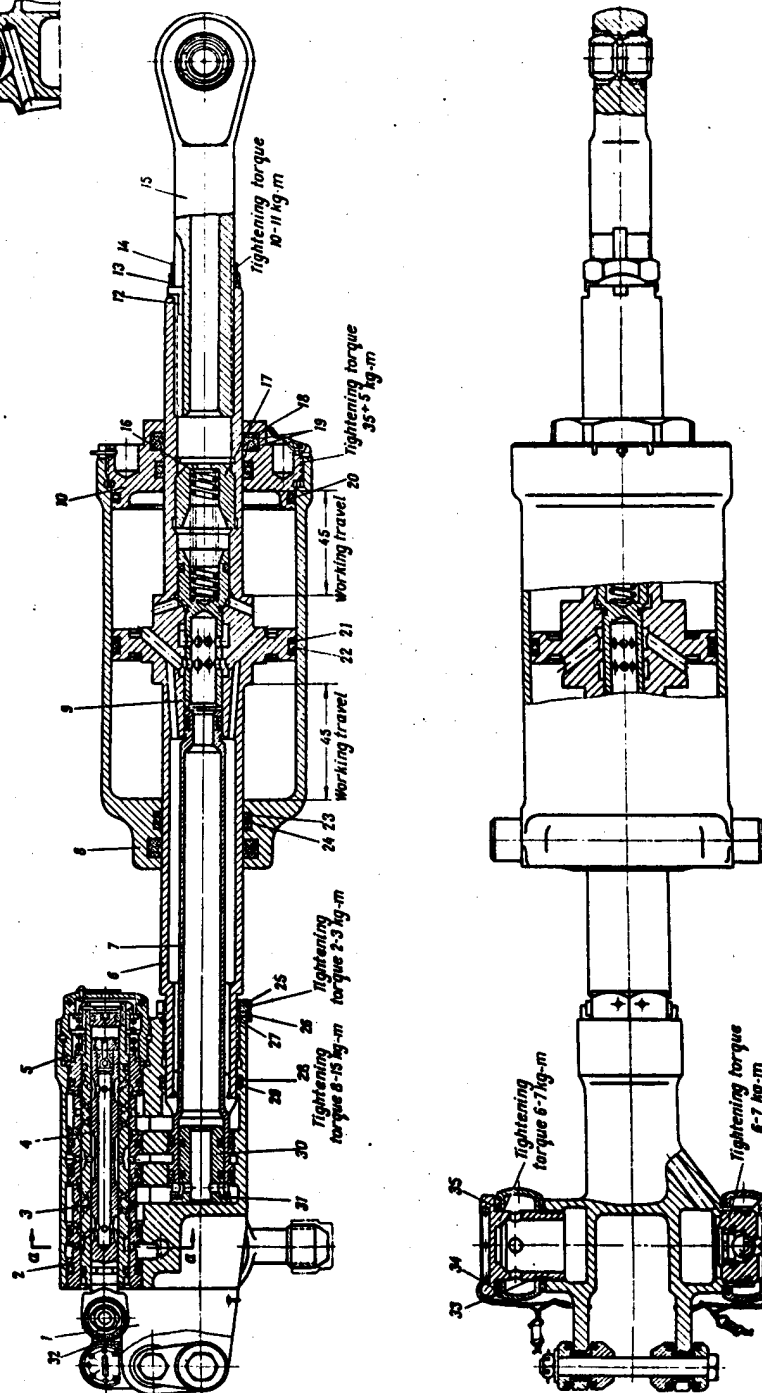
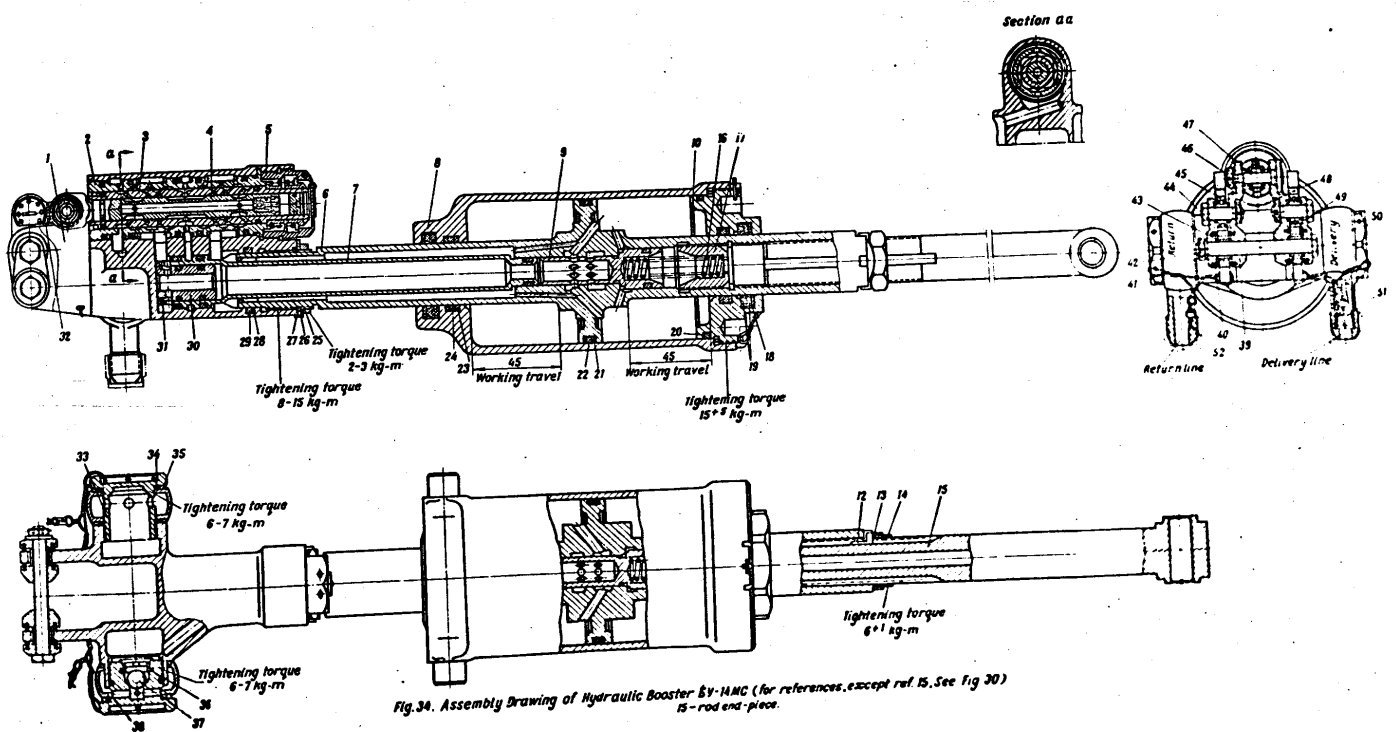


Fig. 32. Hydraulic Booster 5V-14M. Assembly Drawing (for references, except ref. 15, see Fig. 30)



Fig.33. 6Y-14MC Booster. External View



- 43 -

The operating fluid is supplied under pressure to the second recess on the right side through the pipe connection of the delivery line and flows through the radial holes in the sleeve to the middle recesses of the main or duplicate slide valves for distribution. The extreme left recess together with the radial and oblique passages in the sleeve and in the duplicate and main slide valves forms permanent passages through which the operating fluid is returned into the tank. Each of the two other recesses (the first and the third one on the right side) is connected by radial and oblique borings with three inner annular grooves. They serve to communicate the operating cavities of the cylinder either with the delivery line or the return line depending on the position of the slide valve.

The inner cavity of the sleeve houses duplicate slide valve 3 which is also a hollow bushing. The outer surface of the duplicate slide valve is provided with four angle grooves and three shaped recesses.

During operation of the hydraulic booster from the duplicate slide valve, the two extreme shaped recesses connect the return line of the slide valve with the drain line through the oblique borings. The middle shaped recess, depending on the position of the slide valve, connects the right or left cavities of the cylinder with the delivery line.

The inner surface of the duplicate slide valve has four annular recesses which are connected through the oblique passages with the four angle grooves on the outer surface of the slide valve. These annular recesses, holes and grooves form the inlet and outlet passages for the operating fluid during hydraulic booster operation from the main slide valve.

The outer surface of the duplicate slide valve also has two annular grooves of a rectangular section for sealing rings.

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The right end of the duplicate slide valve mounts bushing 9, spring 10 and nut 12 fixed by stop ring 14. Spring 10 holds the duplicate slide valve in the neutral position.

When the main slide valve gets jammed, the pilot, overcoming the force of the spring, shifts the duplicate valve. The travel of the duplicate slide valve is determined by the clearance between bushings 43 (See Fig. 30) and the holes in the head.

Main slide valve 4 (See Fig. 35) which is a hollow rod with an ear at the left end is housed inside the duplicate slide valve.

The outer surface of the main slide valve has three recesses which form operating collars. Two middle collars of the main slide valve ensure the regulation of the fluid supply into one cylinder cavity and its return from the other cavity by covering and uncovering the ports in the duplicate slide valve and, consequently, the channels leading to both cylinder cavities. The inner edges of the middle collars control the delivery of the fluid into the cylinder, and the outer edges - the return of the fluid from the cylinder.

The operating surfaces of the main and duplicate slide valve collars are machined and finished to very close tolerances so that they may cover in the neutral position and open in a shifted position the fluid inlet and outlet passages simultaneously and symmetrically.

The hole on the right side of the main slide valve houses a damper which is essentially a return valve. The purpose of the damper is to eliminate a possible hydraulic disbalance of the slide valve by permitting the operating fluid to flow through the clearances between the ball and the seat in the damper body.

The main slide valve is hinged with bell crank 39 (See Fig. 30) which imparts to it reciprocal motion. The travel

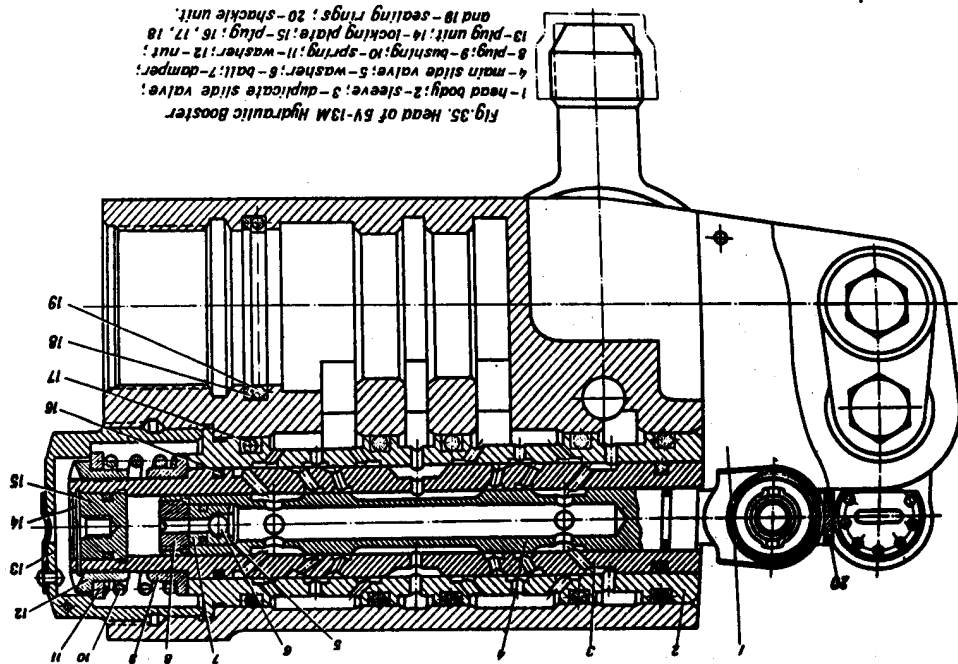


Fig. 35. Head of 6V-13M Hydraulic Booster
1-head body; 2-sleeve; 3-duplicate slide valve;
4-main slide valve; 5-washer; 6-ball; 7-damper;
8-plug; 9-bushing; 10-spring; 11-washer; 12-nut;
13-plug unit; 14-locking plate; 15-plug; 16, 17, 18
and 19-sealing rings; 20-shackle unit.

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of the main slide valve is determined by the size of the clearance between bushing 43 and its hole in head 1.

Head 1 is connected with piston 6 with the help of a thread. The joint is packed by rubber rings with leather gaskets.

V. TYPE EV-1Y HYDRAULIC BOOSTERS

The given type includes hydraulic boosters EV-1Y (See Fig.1), EV-1M (Fig.36) and B1-1M-PB (Fig.37). These boosters are various modifications of the EV-1Y type and differ from one another by certain design elements and installation sizes.

Hydraulic boosters EV-1 are widely used in various aircraft. Hydraulic boosters EV-1Y, a little modernized, are interchangeable with hydraulic boosters EV-1 and differ from the latter by the following features:

1. The threaded joint of the sealing nut with the cylinder has been substituted by a lock joint of the run breech block type which allows to remove the cylinder without disassembling the whole unit.
2. The head-to-rod joint has been altered;
3. The sealing cup design has been changed and one packing unit has been divided in two, which reduces the operating rod friction and improves the tightness of the operating rod-to-cylinder sliding joint;
4. The role of the guide bushing in the hydraulic booster EV-1 is performed by the booster head whereas the EV-1Y type is provided with a special floating bushing which improves the tightness of the joint, and prevents the possibility of the distributing end-piece being jammed during assembly and operation under great loads as well as in the extreme positions of the piston.

Hydraulic boosters EV-1Y, series A, differ from hydraulic boosters EV-1Y of earlier series by the following features:

1. Improved anti-corrosive coating.
2. Smaller clearances in sliding joints.
3. Replacement of one set of tongued cups in the sliding joint of the operating rod with the cylinder and the cylinder bushing by a set with a sealing ring of the round section.
4. Presence of leather rings protecting the rubber seals in sliding joints.

Hydraulic boosters EV-1V are installed on MiG-17 aircraft in the reversible system of ailerons control.

Hydraulic boosters EV-1M and EV-1M-PB differ from EV-1V boosters, series A, by the following features:

1. Reinforced trunnions by which the hydraulic booster is attached to the aircraft bracket.
2. Different construction of the end-piece which is connected with the rods running to the aircraft control surfaces.

Hydraulic booster EV-1M-PB is employed in the elevator control system, since the travel of elevators is not equal relative to the neutral position, this hydraulic booster differs from the EV-1M modification by a symmetric location of the flat on the metering valve.

Hydraulic boosters EV-1M are installed in the irreversible aileron control system, whereas EV-1M-PB boosters are used in the irreversible elevator control system.

Hydraulic boosters EV-1V are more widely used than other boosters of this type, therefore the construction and operation of EV-1V boosters will be treated in greater detail in the subsequent paragraph.

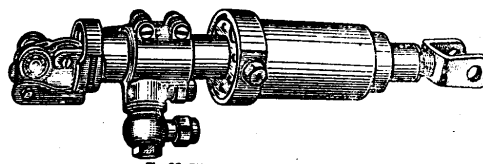


Fig. 36. EV-1M Booster. External View

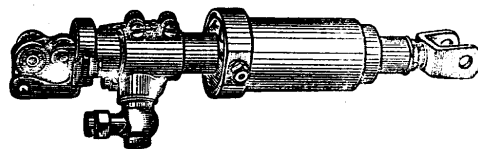


Fig. 37. EV-1M-PB Booster. External View

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1. Basic Specifications

Characteristics	EV-1Y	EV-1M and EV-1M-B
1. Type	2	3
2. Operating principle	Double-acting Hydro-mechanical Follow-up	
3. Control system	Not less than 1,100 kg with $P = 80 \text{ kg/cm}^2$	
4. Maximum power	750 kg with $P = 60 \text{ kg/cm}^2$ $60 \pm 5 \text{ kg/cm}^2$	
5. Operating pressure in hydraulic system	$56 \pm 1 \text{ mm}$ 52 mm	
6. Maximum travel of operating rod	(26 mm on either side of the neutral position)	
7. Working travel of operating rod	52 mm (26 mm on either side of the neutral position for EV-1M boosters; not more than 19 mm on the side of the fork and not more than 33 mm on the side of the head for EV-1M-FB boosters)	
8. Maximum travel of distributing rod	$0.8 - 0.9 \text{ mm}$ $1.2 - 1.3 \text{ mm}$	

I. Basic Specifications

Characteristics	EY-1V	EY-1M and EY-1M-PB
1	2	3
1. Type	Double-acting	
2. Operating principle	Hydro-mechanical	
3. Control system	Follow-up	
4. Maximum power	Not less than 750 kg with $P = 60 \text{ kg/cm}^2$ $60 \pm 5 \text{ kg/cm}^2$	Not less than 1,100 kg with $P = 80 \text{ kg/cm}^2$
5. Operating pressure in hydraulic system	$56 \pm 1 \text{ mm}$	$60 - 80 \text{ kg/cm}^2$
6. Maximum travel of operating rod	52 mm (26 mm on either side of the neutral position)	$56 \pm 1 \text{ mm}$
7. Working travel of operating rod		52 mm (26 mm on either side of the neutral position for EY-1M boosters; not more than 19 mm on the side of the fork and not more than 33 mm on the side of the head for EY-1M-PB boosters)
8. Maximum travel of distributing rod	0.8 - 0.9 mm	1.2 - 1.3 mm

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1	2	3
9. Dead zone 10. Speed of operating rod travel under load 11. Operating rod friction with hydraulic system out in 12. Friction losses in distributing rod 13. Hydraulic booster weight 14. Operating fluid 15. Guaranteed service life of hydraulic booster 16. Guaranteed storage period of hydraulic booster and spare parts as slushed by the Manufacturer	0.2 - 0.6 mm Not less than 70 mm/sec. Not more than 12 kg Not more than 0.55 kg Not more than 3.5 kg AM-10 300 hours ¹ during 4 - 5 years 2 years	Not more than 0.1 mm Not more than 0.65 kg Not more than 3.65 kg AM-10 300 hours ¹ during 4 - 5 years 2 years

¹ The guaranteed service life of the hydraulic booster includes 3 years of booster operation on aircraft, the remaining time being distributed between transportation and storage in depots of the Manufacturer and the Customer.

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2. Operating Principle

The hydraulic diagram of operation of this type boosters differs from the diagram described above (See Section II) by the system of cross-feeding both cavities of the cylinder with the help of floating plungers of the metering valve and the vacuum damper of the one-way slide valve.

The diagram of hydraulic booster EV-1V operation is illustrated in Fig.38.

Cylinder 1 accommodates operating rod 4 which is provided with channels supplying the operating fluid to the cylinder and returning it from the cylinder.

Distributing rod 2 which is housed in the inner cavity of the operating rod serves for the distribution of the operating fluid. It is essentially a slide valve with four collars. The operating fluid is supplied to the recess between the middle collars of the slide valve from a pump which builds up a pressure up to 60 kg/cm².

When the distributing rod is in the neutral position, the middle collars of the slide valve overlap the cylinder supply and return channels "a" and "b" by 0.2 - 0.3 mm, thus locking the fluid in both cavities of the cylinder.

When the distributing rod moves in one or the other direction, the channels become uncovered, which results in the hydraulic fluid being forced into one of the cavities of the cylinder and out of the other into the return line. Under the hydraulic fluid pressure the operating rod moves along the axis of the cylinder and deflects the controlled units through leverage.

Both EV-1V and EV-10 hydraulic boosters allow to overcome heavy loads originating from the surfaces of the controlled units by a minor effort applied to the control stick or control pedals.

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Metering needle 6 located inside the piston of the operating rod parallel to its axis ensures a smooth and uniform travel of the operating rod under load. As the piston moves from the neutral to the load position, additional quantities of the operating fluid are supplied into the operating cavity through the flats of the metering needle (Diagram II, slide valve shifted to the right) directly from the delivery line by-passing the distributing device; when the piston moves to the neutral position (Diagram III), the operating fluid is supplied to the drain cavity of the cylinder directly from the delivery line through the flats of the metering needle and damps the piston motion.

The hydraulic booster is provided with an emergency cross-feed system ensuring a normal manual control if there is no pressure in the hydraulic system. This purpose is served by plungers 3 by-passing the operating fluid from one cylinder cavity to the other (Diagram IV) when the manual control is used.

Under normal operating conditions of the hydraulic booster the liquid is supplied to the recess between the plungers as a result of which the plungers occupy the extreme position and cover the channels which connect both cavities of the cylinder.

The cylinder is sealed tight by rubber seals.

Diagram I illustrates the neutral position of the slide valve; the middle collars of the valve cover channels "a" and "g" and the operating rod is motionless as both cylinder cavities are cut off from the delivery and return lines.

Diagram II illustrates operation of the hydraulic booster when the operating rod moves to the right towards load position (for aileron deflection). To deflect the ailerons, operating rod 4 must move to the right or to the left of its middle position in the cylinder.

Diagram I. Slide valve in neutral position. Operating rod (piston) motion from middle position

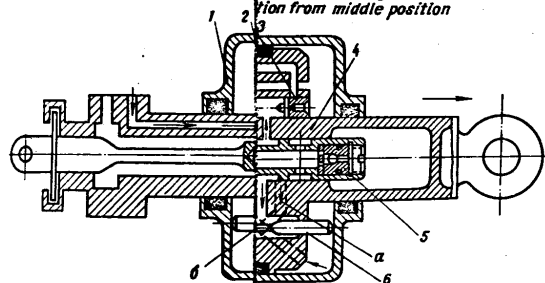
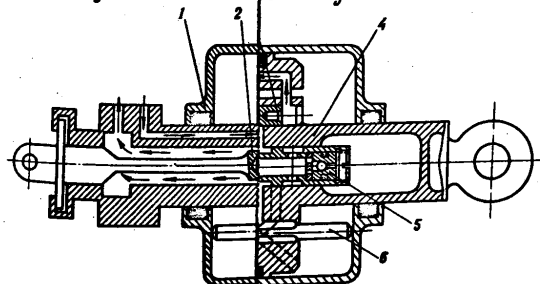


Diagram III. Slide valve in left-hand position. Booster controlled system. Fluid being moving to the left from load left cavity



inside the piston of the cylinder ensures a smooth and quiet operation under load. As the piston moves to the right position, additional fluid is supplied into the cylinder of the metering needle (to the right) directly to the distributing device; in the neutral position (Diagram III), the drain cavity of the metering needle line through the flats of the piston motion. The piston is provided with an emergency manual control if there is a failure in the system. This purpose is achieved by operating fluid from the metering needle (Diagram IV) when the

operation of the hydraulic booster is controlled manually. The recess between the plungers occupy the space which connect both the metering needle and the rubber seals. In the neutral position of the valve cover the operating rod is motionless and the fluid is forced from the delivery

operation of the rod moves to the right (deflection). To deflect the rod to the right or to the left of the cylinder.

Diagram I. Slide valve in neutral position. Operating rod (piston) motionless

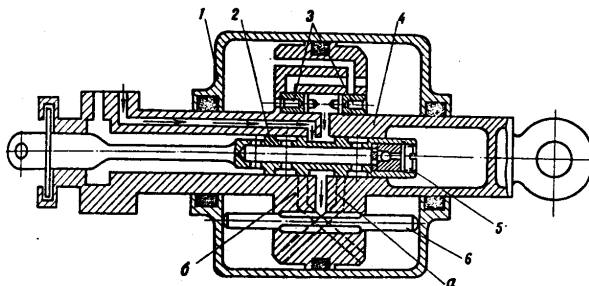


Diagram II. Slide valve in right-hand position. Operating rod (piston) moving to the right towards load position from middle position

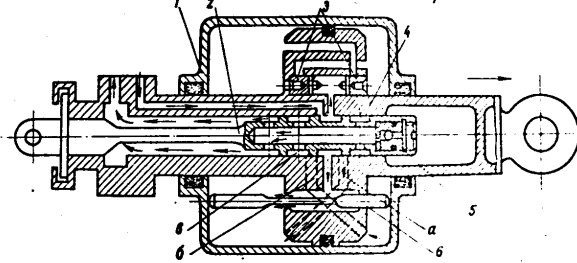


Diagram III. Slide valve in left-hand position. Operating rod (piston) moving to the left from load position to neutral position

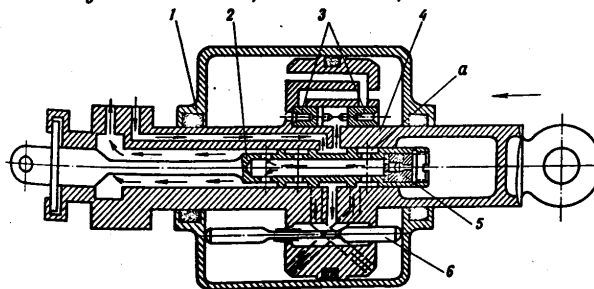


Diagram IV. No pressure in hydraulic system. Booster controlled manually. Operating rod (piston) moving to the right; fluid being forced from right cavity to left cavity

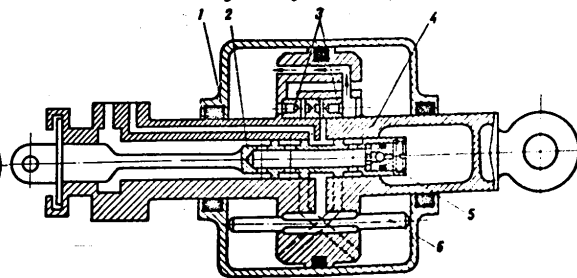


Fig. 38. 6Y-1Y Hydraulic Booster. Operation Diagram
1 - cylinder; 2 - slide valve; 3 - by-pass plungers; 4 - operating rod; 5 - slide valve damper; 6 - metering needle.

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In the case under consideration operating rod 2 moves to the right, as a result of which channels "a" and "g" will be uncovered on the left side and the operating fluid will be forced into the left cavity of the cylinder through the uncovered passage into channel "g". Simultaneously, the operating fluid will be supplied into the left cavity of the cylinder directly from the delivery line through the flats of the metering needle. Under the action of the operating fluid the operating rod will start moving to the right relative to the cylinder deflecting the ailerons from their neutral position. At the same time the hydraulic fluid in the right cavity of the cylinder will be forced out into the return line along channel "g" through the uncovered passage of recess "g".

D i a g r a m III illustrates operation of the hydraulic booster when the operating rod returns from the load position (the ailerons return to the neutral position). In this case the distributing rod (slide valve) is shifted leftward as a result of which channels "a" and "g" will be partially uncovered on the right side and the operating fluid will start coming into the right cavity of the cylinder along channel "g" through the uncovered passage.

Under the action of the hydraulic fluid and aerodynamic forces on the ailerons the operating rod moves to the left to its middle position forcing the fluid into the left cavity of the cylinder along channel "a" to the return line.

Simultaneously, the operating fluid is supplied from the delivery line to the left cavity of the cylinder opposite the moving piston through the flats of the metering needle, thus ensuring a more smooth and uniform motion of the piston. When the piston reaches the middle position relative to the cylinder, the fluid supply through the metering needle stops.

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When the operating rod moves to the left of its neutral position, additional quantities of the hydraulic liquid will be supplied to the right cavity of the cylinder.

D i a g r a m

IV illustrates the manual control. The need of the mechanical control may arise if there is no pressure in the hydraulic system. In this case the ailerons will be deflected by the pilot's physical effort applied to the control stick and transmitted through a system of levers.

During this, operating rod 4 functions as one of the rods in the control system.

To ensure a possibility of the mechanical control, the hydraulic boosters incorporate two pairs of by-pass plungers which disconnect the cylinder cavities if there is pressure in the delivery line. Should there be no pressure in the hydraulic system, the pilot's effort is transmitted to operating rod 4. The pressure that is built up in the right or left cylinder cavity (depending on the direction of the effort) moves the corresponding plungers. The other plungers located at the opposite side of the piston head are motionless.

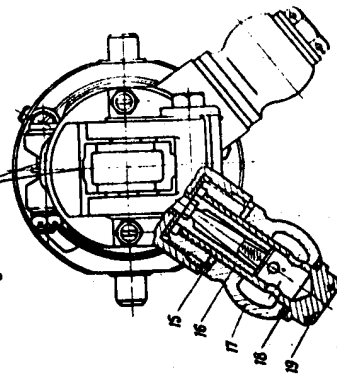
The moving plungers open the channels connecting the cylinder cavities and ensure a free motion of the operating rod.

3. Description of Design

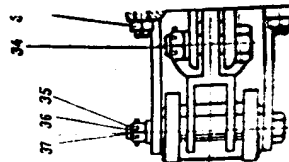
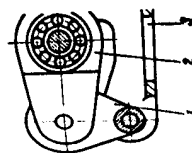
The H-2 hydraulic booster incorporates cylinder 23 (Fig. 2), and operating rod (piston) 12 housing distributing and (inlet valve) 11 with a system of inlet and outlet plungers, and parts which connect the unit with control rods and control hydraulic system.

Cylinder 2 (Fig. 2) is a split-walled bushing with a bottom plate and two annuli 4. The annuli serve for attaching the hydraulic booster to the aircraft, bushing 2

Notches on end-piece and head unit to be aligned during final assembly



Tightening torque 8-12 kg



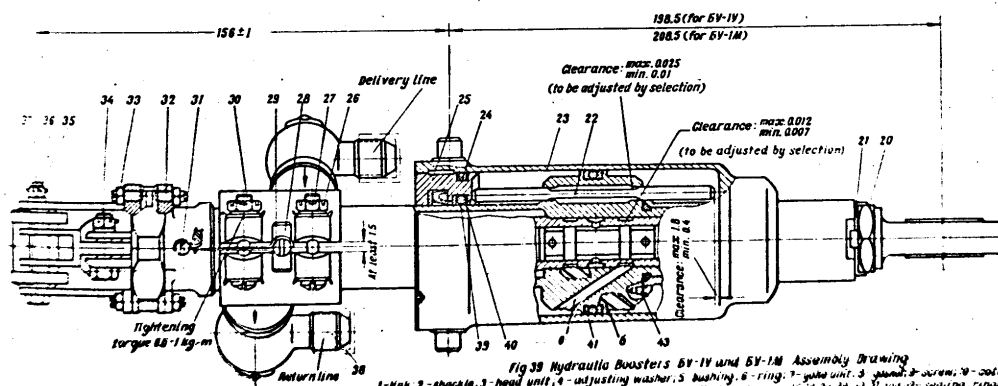
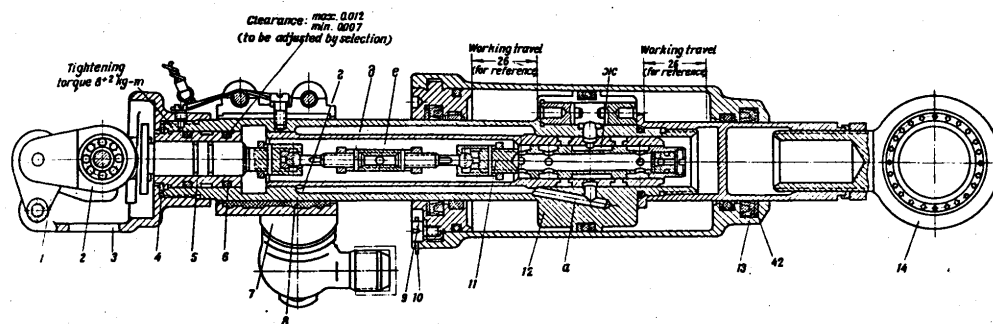


Fig 38 Hydraulic Boosters BV-IV and BV-IM Assembly Drawing

1 - knob, 2 - shackle, 3 - head unit, 4 - adjusting washer, 5 - bushing, 6 - ring, 7 - gate unit, 8 - pin (15 x 16), 9 - slide valve unit, 10 - piston unit, 11 - seal unit, 12 - end-piece, 13 - nut, 14 - end-piece, 15 - 16, 17 - valve, 18 - seal, 19 - seal, 20 - seal, 21 - seal, 22 - seal, 23 - seal, 24 - seal, 25 - seal, 26 - seal, 27 - seal, 28 - seal, 29 - seal, 30 - seal, 31 - seal, 32 - seal, 33 - seal, 34 - seal, 35 - seal, 36 - seal.

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is an end-face cover of the cylinder and is connected with the latter by means of two projections on the bushing which fit into the corresponding slots on the cylinder. The bushing-to-cylinder joint is locked by means of screws 3.

The cylinder-to-bushing joint is sealed tight by rubber ring 24 (Fig. 39).

The end plate and the bushing of the cylinder are provided with holes for the rods of piston unit 12. The necessary tightness is ensured by sets of rubber collars 13 (seal unit) with leather rings 42, round rubber rings 39 and two leather rings 40 inserted into the recess of the cylinder and the bushing.

The purpose of the leather rings is to make the rubber seals more wear-resistant.

The cylinder and the cylinder bushing are manufactured of alloy steel 12XH3A. To improve the wear-resistant quality and to prevent scores and seizing, the friction surfaces are subjected to case-hardening. To protect the outer surfaces from corrosion, the surfaces are cadmium-plated.

The piston serves as operating rod by means of which the effort of the hydraulic fluid is transmitted to the ailerons through a system of levers. This unit consists of piston 1 (Fig. 41) with press-fitted sleeve 2, two pairs of plungers 3, rod 4 and a number of small parts. The piston unit incorporates a distributing mechanism.

Piston 1 is a hollow rod carrying a threaded piston head. The latter has two channels to supply the operating fluid from the delivery line to the middle recess of distributing slide valve 11 (See Fig. 39).

The operating fluid is delivered through special channels to a calibrated hole which accommodates metering needle 22.

Two other holes house two cross-feed plungers 3 each (See Fig. 41). The cavity between the plungers also communicates with the delivery line, due to which the plungers are in the

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extreme positions covering the channels which interconnect the cylinder cavities. The circular groove on the piston head accommodates rubber ring 7 and two leather rings 6 which divide the cylinder in two cavities.

Pressed-in sleeve 2 (See Fig. 41) makes one unit with the piston. Circular recess "A" between the piston rod and the sleeve serves for delivery of the operating fluid from the delivery line to the distributing slide valve.

The inner cavity of the sleeve is made stepped. The cavity of the smaller diameter receives the distributing slide valve, while cavity "B" serves for the return of the fluid from the hydraulic booster.

The external surface of the cavity of a smaller diameter has three circular grooves. The middle groove connects cavity "B" (Fig. 39) with fluid supply circular cavity "A" through the radial holes and channel "a". The extreme right groove serves to connect the left cavity of the cylinder through channel "a" with a pair of radial ports lined up with the right middle collar of the slide valve. The extreme left groove serves to connect the right cavity of the cylinder through channel "a" with a pair of radial ports lined up with the left middle collar of the slide valve. To ensure the required overlap of the slide valve pair, the width of their ports and their mutual arrangement are within 0.02 - 0.03 mm.

The outer surface of the piston is threaded on one side to receive hydraulic booster lead 3 (Fig. 39); the two holes made here in the piston serve for connection with the delivery and return lines. The other end of the piston is connected with rod 4 (See Fig. 41) through a thread joint.

The rod is a hollow cylinder with a recess for lightening purpose; the female thread at both ends of the rod serves for connecting the rod with end-piece 14 (See Fig. 39) and the piston. The sealing between the piston and the rod is effected by means of rubber ring 5 (See Fig. 41).

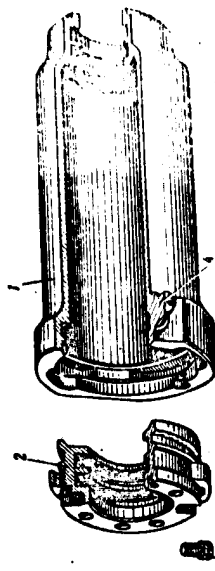


Fig. 40. Cylinder and Bushing of 54-11 4, 3
1 - cylinder, 2 - bushing, 3 - screws

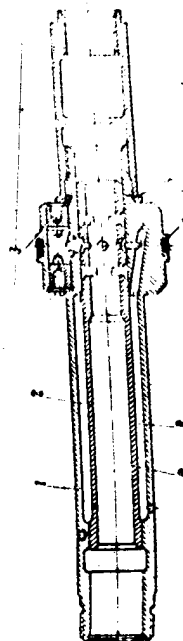


Fig. 41. Piston Unit (Operating Rod)
1 - piston, 2 - sleeve, 3 - plungers, 4 - rod, 5 - ring

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To provide for the locking of the threaded joint, the collar of the rod has 18 slots, while the end face surface of the piston head is provided with a threaded hole. The joint is locked by means of screw 43 (See Fig.39).

The sleeve of the piston unit accommodates slide valve mit 11, which regulates the delivery of the operating fluid and the return of the fluid from the cylinder cavities.

The slide valve (distributing rod) consists of slide valve 9 (Fig.42), a hinge joint and end-piece 1.

Slide valve 9 is a hollow rod with four collars and a number of radial holes for the return of the operating fluid from the hydraulic booster to the return line.

To ensure the necessary overlap of the ports (within 0.2 - 0.3 mm) the mutual arrangement and the length of middle collars are made within ± 0.02 mm. The edges of the middle collars must not be deformed or rounded so as not to disturb the necessary overlap.

In view of the fact that the overlap value is very small, the tightness in the slide valve-to-sleeve joint is ensured by a careful surfacing of the friction surfaces and by matching the slide valves and sleeves to one another so that the diametral clearance between them is within 0.007 - 0.012 mm.

The hole at the right end of the slide valve houses a damper which is essentially a return valve. The purpose of the damper is to eliminate a possible disbalance in the slide valve pair by filling the closed space in the hydraulic booster rod with the operating fluid.

The damper consists of body 12, steel ball 11, threaded plug 13 and washer 10.

The slide valve is thread-joined with the hinge which consists of turn-buckle 6, extensions 7 with spherical heads, nuts 8, spherical seats 4, two lock nuts 2 and pins 3. Spherical seats 4 take the thrust of the heads of extensions 7. The extensions are connected with slide valve 9 and end-piece 1.

End-piece 1 is a guide rod with a shaped flange. The end-piece has a fork with holes for bolt 34 (See Fig.39) by means of which the slide valve unit is connected with control bell crank through shackle 2.

The other end of the end-piece is provided with a thread to receive the nut of the hinge joint.

This construction of the distributing rod makes it possible to adjust the slide valve position in the sleeve and compensates for axial misalignment of the seats for the slide valve and the end-piece.

The head unit serves for the connection of the distributing rod with control levers and allows to adjust the slide valve unit position relative to the distributing ports of the piston sleeve.

The head unit includes head 1 (Fig.43) and bushing 2.

The head has a complicated shape and is essentially a bushing with a hollow flange and a bracket at one end and a female thread at the other.

Two stops 3 are screwed into the faces of the head flange from both sides limit axial motion of the flange of the end-piece linked with the distributing slide valve.

The head unit is connected with the rod of piston 12 (See Fig.39) by means of a thread. Adjusting washer 4 is selected and installed between the end faces of the head and the piston to ensure a definite position of the head relative to the piston and to keep the hydraulic booster size within the rated limits.

The position of the head relative to the piston rod is fixed by lock screw 31. Upon the completion of adjustment of the distributing rod travel (slide valve unit), nuts 33 of the stops are stop-soldered.

The yoke unit serves for the delivery of the operating fluid to the hydraulic booster from the delivery line and for

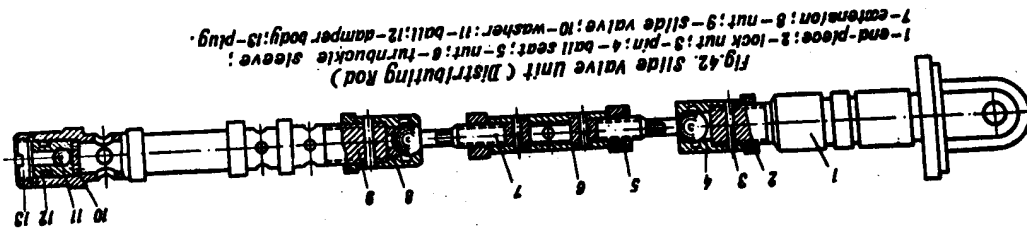
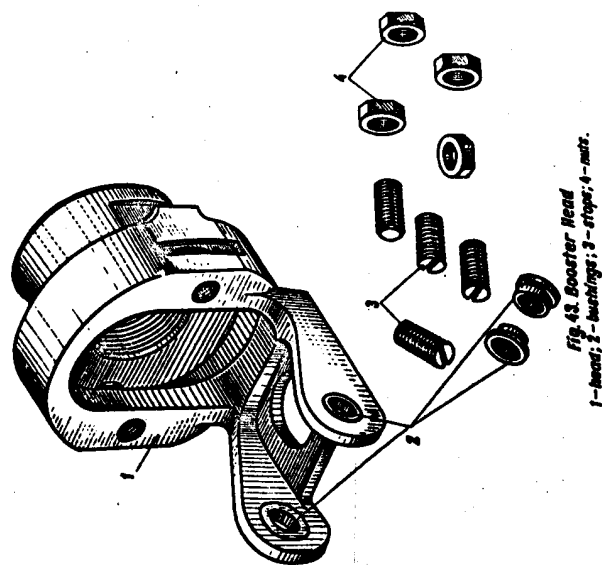


Fig.42. Slide Valve Unit (Distributing Rod)
1 - end-piece; 2 - lock nut; 3 - pin; 4 - ball seat; 5 - nut; 6 - turnbuckle sleeve;
7 - extension; 8 - nut; 9 - slide valve; 10 - washer; 11 - ball; 12 - damper body; 13 - plug.



the return of the fluid to the return line.

The yoke unit consists of yoke 1 (Fig. 44), sleeves 3, bushings 2 and pins 4. Yoke 1 is a bushing which has a slot throughout its whole length and two bosses. The outer surface has two pairs of lugs with holes for coupling bolts 27 (See Fig. 39), located near the slot.

The holes in the yoke bosses which line up with the holes in the piston rod during the installation of the yoke serve for the delivery of the operating fluid to the hydraulic booster and for the return of the fluid from the booster. The bosses receive thin-walled bushings 2 (See Fig. 44) and threaded steel sleeves 3 which are locked in place with pins 4. Bushings 2 inserted in the yoke holes serve to keep off rubber from bulging into the fluid lines and outlet holes.

The yoke is fitted on the piston rod and is tightened with coupling bolts. The joint between the piston rod and the yoke is made tight by rubber gasket 5 (See Fig. 39).

The assembly arrows made on the yoke bosses indicate the places of connection of the delivery and return lines.

Screen filter 16 arranged inside the connection is

serves to additionally clean the hydraulic fluid.

The hydraulic booster is connected with the hydraulic control rods by means of link 2, shackle 1 and end-plate 14.

The link is hinged with the hydraulic boosting beam by means of bolt 35. Shackle 1 which is fastened to the distributing rod of the slide valve with bolt 34 is connected together with link 1 to the valve stem of the valve with leverage.

End-plate 14 is connected to the rod of the hydraulic unit (operating rod) and is locked in position by means of lock nut 20 and lock 23.

The end-plate points the hydraulic boosting beam to the aircraft control unit leverage.

Hydraulic boosters EV-1M and EV-1M-PB differ from the EV-1V type boosters by the fork unit, more sturdy trunnions of the cylinders, wider service collars and by the distributing rod slide valve whose collars are made with out flats so as to improve speed characteristics.

Besides, the design of the needle of hydraulic boosters EV-1M-PB has been altered; the flat of the needle is made asymmetric. This fact calls for attention during assembly of the EV-1M-PB unit, so that the extended cylindrical end of the needle mounted in the piston is located on the side of the head.

The swivel nipple of the EV-1M-PB hydraulic booster is shifted through 190° relative to the nipple of hydraulic boosters EV-1V and EV-1M.

VI. INSTALLATION AND MAINTENANCE INSTRUCTIONS

1. Delivery Set

The hydraulic boosters shipped off by the Manufacturing Plant to the Consumers are fully assembled, coated with corrosion-preventive compound and packed in special cases in paraffine paper.

Alongside with the hydraulic booster, the case contains:
(a) technical papers (Hydraulic Booster Service Log, Delivery List);

(b) set of spare parts for one hydraulic booster;
(c) set of erecting tools.

Besides, each 10 hydraulic boosters sent to the Consumer are provided with a group set of spare parts.

2. Installation of Hydraulic Booster on Bracket and Checking Standard of Installation

Correct operation of a hydraulic booster depends to a great extent on proper installation of the booster on the air-

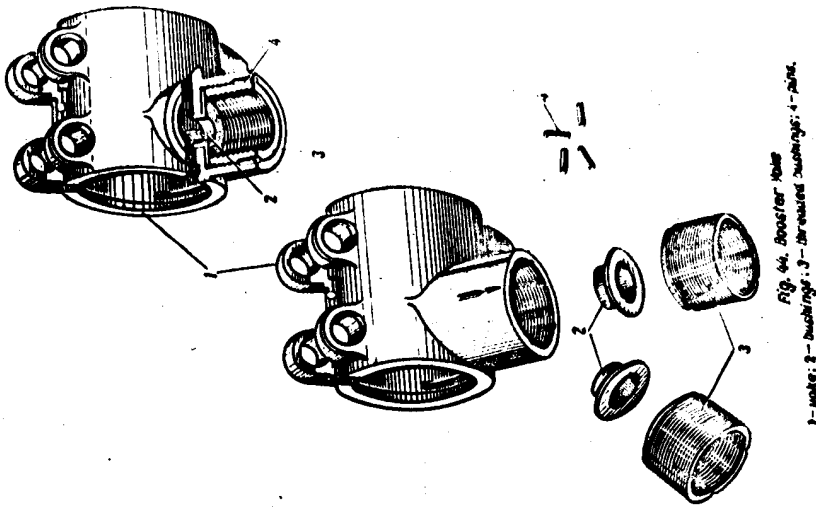


Fig. 44. Booster Valve
1 - guide; 2 - bushing; 3 - threaded bushings; 4 - nut.

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craft. Misalignment, jamming and excessive friction in hinge joints of the reversible system may be the cause of abnormal operation, for instance, of the control stick or pedal creeping.

In order to ensure a proper installation of the hydraulic booster and to check the installation for quality, it is good practice to mount the hydraulic booster on a special bracket and then to install it together with the bracket on aircraft. The bracket must be a sufficiently rigid panel on which the hydraulic booster is installed together with the reversible system (See Fig.5).

It is not advisable to mount the hydraulic booster on a bracket consisting of two parts as this may result in scores on the operating rod and cause the aircraft control stick creeping due to misalignment in the reversible system. The design of the bracket, however, which is accepted in helicopters does not meet these requirements as the hydraulic booster together with the front bell crank is mounted on one bracket, while the rear bell crank linked with the reverse rods is mounted on another bracket.

Prior to installing a hydraulic booster on a bracket remove the anti-corrosive compound from the booster in compliance with appropriate instructions.

After this the hydraulic booster of the EV-1Y type should be checked manually for ease of the operating rod travel. Besides, it is necessary to check the booster head and all the bolts for presence of seals and locks.

All the parts of the booster mounting bracket must meet the following requirements:

- (a) all hinge joints must be parallel with the axis of the booster bell crank;
- (b) the panel must be sufficiently rigid ensuring a reliable installation of the hydraulic booster and of all the units of the reversible system;

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(c) planes of rocking of all hinge joint elements must be aligned with the plane of rocking of the hydraulic booster. When installing the hydraulic booster and all associated parts on the bracket, it is necessary to comply with the following requirements:

(a) the connection of all ears with forks of all hinge joints must be free of jamming and rubbing throughout the whole range of the operating rod travel;

(b) tightening of hinge joint bolts must not increase friction of the mating parts. The bolts of all hinges must enter the holes under a manual effort in any position and throughout the whole range of travel of the operating rod.

Upon the completion of travel of the operating rod, of erection throughout the installation check the standard of erection throughout the whole range of travel. For this purpose:

(a) Make sure that there is no seizure or play between the ears and forks in all hinge joints;

(b) check to see that the booster head is positioned correctly relative to control levers;

(c) check to see that the notches on the distributing rod and the slide valve body or head are in line, which

relates to a correct installation of the distributing rod;

(d) check all joints for correct locking;

(e) check the standard of erection by reproducing the

service movements which are characteristic of the aircraft (helicopter). If the booster is meant for a permanent control system where the control stick or pedal creeping is highly undesirable, the check should include a test of the control stick or pedals for possible creeping.

If the hydraulic booster is mounted in the periodic control system, such checking is not necessary. For example, the helicopter collective pitch control consists in a periodic shifting of the lever on the control sector in accordance with the desirable change of the collective pitch.

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The term "creeping" is applied to the motion of the control stick or control pedals which continues after the cessation of light jerks applied to the control stick or pedals to shift them with the booster hydraulic system switched on. The force which must be applied in order to stop this motion of the control stick or pedals must not exceed the maximum value set for the given aircraft. For example, the Specifications for checking the standard of erection of EV-11 hydraulic booster contain the following paragraph: "In view of the damping force of the hydraulic booster slide valve, the control stick after being shifted to the left is allowed to continue its motion if it can be stopped by an effort up to 260 gr. If the control stick is shifted to the right, the effort must not exceed 100 gr."

The creeping in all cases is caused by a disturbed relation between two forces: the force of friction in the mechanical control linkage from the control stick or pedals to the hydraulic booster and the total force of friction in hinge joints of the hydraulic booster reversible system and hydraulic booster slide valve.

The creeping may be completely absent only if the total force of friction in the reversible system hinge joints and the force of the slide valve friction are less than the total force of friction in the control linkage between the control stick or pedals and the hydraulic booster. This relation is as a rule taken into account by the designers of the aircraft (helicopter) control system.

Not rarely, the creeping of the control stick or pedals is confused with spontaneous motion of these elements to either side. These defects are absolutely different from each other by their nature and consequences.

The creeping causes an excessive strain on the pilot and results in his fatigue. As safety is concerned, however, the creeping is absolutely harmless.

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On the other hand, spontaneous motion of the control stick or pedals to one or the other side is due to the presence of a hard foreign object in the clearances which determine movement of the slide valve relative to the operating rod. Spontaneous motion of the control stick or pedals results from incorrect operation of the hydraulic booster and often has undesirable consequences.

In hydraulic boosters EV-10 foreign objects are likely to get into the clearance between the head bracket and bushings 47 (See Fig. 13), in hydraulic boosters EV-13M - the clearance between the head bracket and bushings 43 (See Fig. 30). These clearances are protected by felt glands. In hydraulic boosters EV-1Y the vulnerable place is the slot in head 3 (See Fig. 39) between stops 32 where the end-piece flange is located. The head of the hydraulic booster EV-1Y is protected by a special cover. In service special care should be exercised towards the condition of this cover; during the installation of these boosters it is necessary to make sure that the cover is securely fastened on the booster head.

In the case of hydraulic boosters EV-10 it is necessary to prevent the possibility of foreign particles getting inside the inner cavity and beyond the gland and to make sure that the felt gland is installed correctly.

Before flight be sure to check the hydraulic booster for operation by shifting the control stick or control pedals three or five times through the whole length of their travel with the hydraulic booster in operation.

The check for creeping is carried out on a special installation whose kinematic diagram and hydraulic system fully correspond to those of the aircraft with the load out off and spring feel mechanism inoperative.

The control stick or pedals of the testing installation must be statically balanced. In those cases when the check

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is carried out on aircraft it is necessary to take into account the disbalance of the control stick.

The friction in the linkage from the control stick to the hydraulic booster should be adjusted so that the total value of the friction force is not less than the minimum possible total value of friction in the aircraft taking into account the friction resulting from sealing and sagging of the rods.

In those cases when the creeping is out of the permissible limits it is necessary to check the force of the distributing rod friction. If the slide valve travel is difficult, remove the slide valve, wash with clean non-ethylated gasoline all service surfaces of the slide valve and the damper and wipe with chamolis and dry the inner cavity of the sleeve till gasoline is completely evaporated.

After that moisten all service surfaces of the slide valve and the sleeve in the hydraulic fluid, install the slide valve back into the sleeve and check the hydraulic booster for proper operation.

Note: When carrying out the above procedures, exercise particular care towards the condition of the locks of turnbuckle joints and distributing rod travel adjusting screws; take special care to protect the service collars of the slide valve from scratches, scratches and scores.

3. Installation of Hydraulic Booster on Aircraft

1. When mounting the hydraulic booster installed on the bracket, ensure a clearance not more than 0.05 mm between the bracket lugs and the bearing plate.

2. Do not install bolts (in joints) with interference; the bolts must be inserted manually.

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9. Check the control stick for creeping with the pump in operation. In case of creeping, check the hinge joints of the bracket for misalignment or rubbing, the operating rod for correct position relative to the control lever, and whether friction of the hydraulic booster distributing rod can be felt by hand.

Rubbing or misalignment should be eliminated by adjusting clearances in the joints and by dismounting the hydraulic booster on the aircraft; friction of the slide valve is eliminated by washing and cleaning the slide valve and its damper as well as the hydraulic booster sleeve.

10. When carrying out scheduled maintenance of the hydraulic booster or when replacing the booster by a new one, remove the booster in the following sequence:

- (a) turn off pressure in the booster hydraulic system and disconnect the delivery and return hoses;
- (b) disconnect the hydraulic booster from the control rod and reinstall the bolt connecting the rods with the hydraulic booster; remove the bolt connecting the operating rod with the aircraft control rods;
- (c) unscrew the nuts fastening the bearing races in the cylinder trunnions;
- (d) remove the hydraulic booster from the bracket.

4. Operation and Maintenance

Maintenance of the hydraulic booster during its service consists in supervision of the booster operation, inspections of seals, checking of operating fluid and the whole hydraulic system for cleanliness, as well as in the performance of scheduled maintenance procedures.

During operation of the hydraulic booster the surface of the operating rod, the slide valve and the stop may get covered with a film of fluid which gradually gathers into

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3. When connecting flexible hoses pay attention to the arrows on pipe connections. The supply hose should be connected to the pipe connection on which the arrow pointer is directed towards the booster inside, whereas the return hose should be connected to the pipe connection on which the arrow points away from the booster. Connecting and disconnecting the operating fluid supply hoses, prevent the hydraulic booster from turning around the operating rod and from being unlocked.

4. The control system should be adjusted in such a way as to provide for a neutral position of the controlled units and control elements within the tolerance limits for the given aircraft when the booster operating rod is in the neutral position. The maximum deviation of the controlled units must be limited not by the hydraulic booster, but by special stops in the system of rods.

5. Check the entire control system for smooth motion with controlled units completely disconnected; seizure is not permissible.

6. When installing the spring feel mechanism, see that the ear of the mechanism fits smoothly into the lever pin and does not cause tension of the mechanism springs in the neutral position.

7. Switch on the pump and check the controlled units for deflection through full angles; the control stick is allowed to be short of, or beyond, the neutral position within the tolerance limits set for the given aircraft.

8. Check the position of the spring feel mechanism. Disconnect the spring feel mechanism, set the controlled units in the neutral position and, with the controlled units and the hydraulic system under pressure, connect the spring feel mechanism to the control lever without tightening the mechanism springs.

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drops; such formation of drops may be tolerated.

Permissible leakage of hydraulic fluid during booster operation:

(a) with the hydraulic system ON, for boosters EV-10 and EV-13M: up to 4 cm³/hour, which amounts to two drops per minute; and for boosters EV-14: up to 5 cm³/hour, which amounts to 3 - 4 drops per minute. The hydraulic booster should be checked for leakage when the booster is operated from the aircraft pump or from a ground source; before the check the hydraulic booster should be wiped clean;

(b) with the hydraulic booster out of operation and the hydraulic system OFF, 5 drops per hour.

When checking the booster for tightness, it is necessary to bear in mind that after a prolonged period of hydraulic booster inoperation (more than 10 - 15 days), the rubber cups and rings lose their elasticity and packing quality, though they ensure a complete tightness of the booster during its operation. To restore elasticity of the booster during booster operation under pressure for 10 - 15 minutes, using for this purpose a special installation or timing this procedure with the scheduled run of the engine.

When filling the reservoir with fluid, do not under any circumstances mix AMT-10 fluid with any other oils. The vessels (tanks, funnels, hoses) used for filling the hydraulic fluid, must be absolutely clean. Before the procedure the vessels should be washed with gasoline and dried in the air, and the operating fluid must be filtered through silk cloth.

It is not allowed to lubricate the internal parts of a hydraulic booster operating on AMT-10 oil with any other lubricants.

The temperature of the operating fluid in the hydraulic system must be not more than +60°C.

Prior to connecting the hydraulic booster to the hydraulic system, prime the entire system from a ground truck. Proposed as follows:

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(a) connect the delivery and return hoses to each other;

(b) pump the hydraulic fluid filled into the system in the course of 5 - 8 minutes with the hydraulic booster disconnected cock open;

(c) switch over the automatic valve 10 times with the hydraulic booster disconnected cock closed, after which drain the fluid from the system.

Upon the completion of the above procedures, connect the hoses to the hydraulic booster, prime fresh fluid and shift the control stick several times through the whole length of its travel with the ground pump connected or the engine running.

Disassembly of the hydraulic booster should be performed only if it is absolutely necessary: in case of a leaky or disadjusted hydraulic booster, excessive friction in the distributing or operating rods; before disassembly make sure that there is no misalignment or seizure in the lever system. The hydraulic booster is disassembled on a clean table covered with paraffine paper. It is allowed to assemble only one booster on the work table in order to prevent confusion of booster parts.

In case of a prolonged interval in operation of hydraulic boosters installed on aircraft it is necessary to test the hydraulic boosters in not less than every 10 days from a ground truck in the course of 5 - 10 minutes. When dealing with hydraulic boosters EV-13M, check the operation of both the main and the duplicate slide valves.

To check operation of the hydraulic booster from the duplicate slide valve, it is necessary to jam the main slide valve with the help of a special device in the following way: fit limiter 3 (Fig. 45) on the journal of the main slide valve and, shifting the cover aside, open the hole in the plug. After that fit the clamp with special slots on the head and nut of the main slide valve bolt and turn in clamp screw 2

so that its end enters the plug hole. After that slightly tighten up the screw and clamp the limiter installed before between the end faces of the main and duplicate slide valves. This will ensure operation of the hydraulic booster valve. the duplicate slide valve.

The set of tools includes one limiter and a clamp with a screw for each hydraulic booster EY-13M. This limiter and the clamp serve at the same time for checking operation of hydraulic boosters EY-14M and EY-14MC.

VII. HYDRAULIC BOOSTER AND AIRCRAFT CONTROL SYSTEM TROUBLE SHOOTING

Trouble	Cause	Remedy
1. Creeping of control pedals or control stick (with spring feel mechanism switched off), unstable position of control stick in neutral (with spring feel mechanism switched off)	(a) Incorrect installation of hydraulic booster on bracket (b) Excessive friction of distributing rod (c) Insufficient preliminary tension of springs of spring feel mechanism. Each of the above causes may bring about an increase of the total friction in the hinges of the reversible system	Check: (a) Clearances between lugs and forks in hinge joints of reversible system at all positions of operating rod (b) Bolts in hinge joints for over-tightening (c) Hydraulic booster-to-bell crank connection (d) Preliminary tension of springs of spring feel mechanism (e) Distributing rod of hydraulic

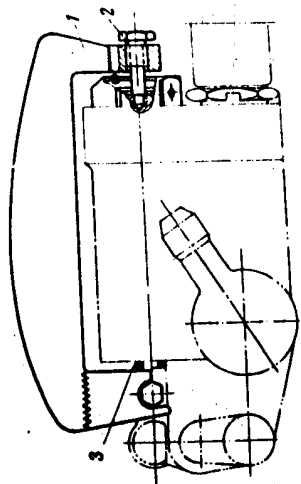


Fig. 45. Fixture for Checking Operation of EY-13M Hydraulic Booster Duplicate Slide Valve
1-clamp; 2-screw; 3-limiter.

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Trouble	Cause	Remedy
	and the friction of the slide valve as compared with the total friction of the linkage between the control stick and the hydraulic booster, with a resultant disbalance between the friction forces which is required for normal operation	booster for easy motion. If the distributing rod moves with difficulty, wash the rod and sleeve (the seat under the slide valve) with clean gasoline and wipe with clean cloth or chamois. After that, coat the distributing rod and sleeve with operating fluid and assemble them in accordance with the prescribed procedure. When performing these operations, see that the looking devices of joints are intact and protect the service collars of the slide valve from scratches. Connect the control linkage and check operation of the hydraulic booster on the ground

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Trouble	Cause	Remedy
2. Binding of control stick in extreme positions	Bindrance in control system between controlled surfaces and hydraulic booster (a) Foreign objects in clearances which determine slide valve travel: clearance between end-piece of flange and booster head faces in hydraulic booster EV-10; clearance between bushing of coupling bolt and hole in head bracket, also clearance between taper surface of stop and bell crank bushing in hydraulic booster EV-10 (b) Distributing device of hydraulic booster out of adjustment	Eliminate bindrance in control system by rod adjustment In case of hydraulic boosters EV-10, remove cover from booster head and thoroughly clean cover and head of dirt and foreign objects. In case of hydraulic boosters EV-10, remove coupling bolt, back out both shackles through some angle and clean clearances between bushing and hole in head bracket. If there are no foreign objects, check operation of hydraulic booster under pressure. If fault still persists, remove hydraulic booster and check distributing rod for correct adjustment (a) Check all joints by sections. Check connection of
3. Difficult motion of control stick or pedals in one direction	Excessive friction in hinges of rods and control	(a) Excessive friction in hinges of rods and control
4. Difficult motion of control stick (or pedals)	Excessive friction in hinges of rods and control	(a) Check all joints by sections. Check connection of

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Trouble	Cause	Remedy
at some sections of their travel	levers at section from control stick (or pedals) to hydraulic booster slide valve (b) Overtightened bolts of hinges of hydraulic booster and control bell cranks See Item 1. Backlash in rods and levers of control system 5. Jerks of control stick (or pedals) at some sections of their travel; same during change of direction of piston motion 6. Distributing rod moves with difficulty	control bell crank with hydraulic booster head (b) Bolts must enter smoothly in place under manual effort or under light impact of wooden mallet See Items 1 and 2. Disconnect hydraulic booster and check for backlash; eliminate backlash if any
	(a) Contamination of working surfaces of piston and cylinder (b) Excessive friction of rubber cure	(a) Disassemble cylinder (or piston) and run, remove cylinder Wash piston and cylinder in non-oxidizing solution, dry, coat with operating fluid and reassemble in position (b) Replace rubber

Trouble	Cause	Remedy
7. Hydraulic booster fails to ensure maximum effort	(a) Heavy drop of pressure in supply line (b) Excessive back-pressure in return line (c) Worn sealing ring on piston	(a) Wash or replace felt filters in supply line (b) Check pressure at booster outlet, for which purpose disconnect return hose from hydraulic booster. Place vessel under pipe connection, increase pressure in hydraulic booster and check its operation with spring feel mechanism disconnected. If hydraulic booster operates normally, it indicates that pressure in return line was too high because of some defect (clogging or clamping of pipes, incorrect connection of pipes) (c) Replace sealing ring on piston
8. Leakage through pipe connections	(a) Loose thread joints (b) Leaky rubber seals	(a) Check pipe connections or cylinder nut for proper tightening

Trouble	Cause	Remedy
9. Leakage of hydraulic fluid around piston rod, slide valve or stop (in excess of limits indicated in paragraph "Operation and Maintenance")	Wear of sealing cups or rubber rings	moment. If fault persists, replace gaskets (b) Check rubber sealing rings Replace set of cups or sealing rings on slide valve
10. Control stick stops in position other than neutral	Spring feel mechanism out of adjustment	Disconnect spring feel mechanism, set controlled elements in neutral position with free control stick (or pedals) and increase pressure (from auxiliary unit) in hydraulic system, after which connect spring feel mechanism to control lever without tensioning spring; eliminate spring tension if any

VIII. SCHEDULED MAINTENANCE

1. Scheduled Maintenance

of Hydraulic Boosters EV-10 and EV-13M

A. Pre-Flight and Post-Flight Inspections:

(a) Check operation of the hydraulic booster with the hydraulic system disconnected and connected by shifting the control stick two or three times in both directions throughout the whole operating range; in case of hydraulic boosters EV-10 and EV-10M repeat the same by engaging and disengaging the duplicate system. The control stick must move without jerks and creeping.

(b) When attending to hydraulic boosters EV-13M, check operation of the duplicate slide valve after every 10 flying hours but not less than once a fortnight.

(c) Examine attachment of the hydraulic booster to the bracket and check the rods and pipe lines for reliable connection to the hydraulic booster.

B. After Every 25 Flying Hours or after Every 3 Months if Aircraft Is Not Flown:

(a) Carry out scheduled maintenance within the scope of pre-flight and post-flight inspections.

(b) Make sure that there is no increase of longitudinal play in hinge joints of the distributing rod. With this purpose, holding up the upper part of the bell crank, move the slide valve along its axis on the side of the cap (with the cap removed). If the joint is sound, motion of the slide valve is practically imperceptible. Should the clearance be too large, send the unit to a repair shop.

(c) Remove the filters mounted in fluid supply pipe connections, examine them and wash in clean non-ethylated gasoline.

In case of clogged filters, remove the hydraulic booster from the aircraft and thoroughly wipe it with a cloth slightly moistened in non-ethylated gasoline. Disassemble the distributing device, for which purpose, in case of EV-10 boosters, disconnect the hinge unit from the slide valve, remove the plug from the damper unit, remove the slide valve body and take out the slide valve. When dealing with EV-10B and EV-10M hydraulic boosters, disassemble in addition automatic delivery and drain valves. In boosters EV-13M remove the main and duplicate slide valves, take off rubber rings, wash them thoroughly with clean non-ethylated gasoline, dry and wipe with chamol. Examine the middle collars of the slide valve; the collars must be free of notches and scratches. Hinge joints must ensure free turning of parts.

If no defects are evidenced, coat the washed parts with the operating fluid and reinstall them in position. If inner cavities of slide valve units are contaminated, send the hydraulic booster to a repair shop for complete disassembly of the head unit and the power cylinder.

- Notes: 1. Disassembly and assembly of hydraulic boosters should be carried out in closed premises on a clean table covered with paraffine paper.
2. If the plane (helicopter) is not flown, pump the fluid through the hydraulic booster using the aircraft pump or a ground installation after every 10 days, for which purpose build up working pressure and shift the control stick not less than 5 - 10 times throughout the entire operating range.

C. After Every 100 Flying Hours or after Every 6 Months if Aircraft Is Not Flown:

- (a) Carry out the operations prescribed for 25 hour maintenance.

- (b) Drain the operating fluid from the hydraulic reservoir and all the hydraulic system, disconnect pipes and hoses.
- (c) Thoroughly wash the hydraulic reservoir and pipe lines with filtered operating fluid and fill all the hydraulic system with fresh filtered fluid. Prior to usage, in all cases, the hydraulic fluid must be filtered through silk cloth.

2. Scheduled Maintenance of Hydraulic Boosters EV-1Y

A. Pre-Flight Inspection

1. Build up working pressure in the hydraulic system and shift the control stick in various directions several times. The control stick must move smoothly, without creeping and jerks. Check visually the control stick for uniform speed of return from the extreme right and left positions to the middle position.
2. Check operation of the cross-feed system, for which purpose disconnect the booster hydraulic system from the cockpit by means of a cock and repeat movements of the control stick in the same directions. The control stick must move smoothly without excessive effort. Faulty operation of the cross-feed system with the hydraulic booster disconnected will be evidenced by a difficult motion of the control stick.

Note: If the cross-feed system cannot be set right after several motions of the control stick, dismantle the hydraulic booster and send it to a repair shop.

B. Post-Flight Inspection

1. Turn the main filter of the hydraulic system 3 or 5 times.
2. Check to see that there is no leakage of the operating fluid from under the hydraulic booster panel.

C. After Every 25 ±5 Flying Hours:

1. Carry out pre-flight and post-flight inspections.
2. Examine attachment of the hydraulic booster and the control linkage.
3. Check the hydraulic booster head cover for condition (in case of boosters EV-1, EV-1Y, EV-1M, EV-1M-PB). If any faults are evidenced, remove the cover and check it thoroughly for cleanliness, after which reinstall the cover in position.

4. Remove the silk and felt filters installed in the hydraulic system and the screen filters installed in the fluid supply pipe connections; examine the filters and wash them in clean non-ethylated gasoline.

If the screen filters installed in the fluid delivery pipe connection are contaminated, remove the hydraulic booster from the aircraft, take out the distributing slide valve unit (also the duplicate slide valve in case of boosters EV-13M, EV-14M and EV-14MC), examine and, if contamination is evidenced, carry out complete stripping and washing of the hydraulic booster; drain operating fluid from the hydraulic tank and the hydraulic system, wash the hydraulic system with operating fluid and fill in fresh fluid.

Upon completing assembly, check the hydraulic booster for:

1. external tightness;
2. suction force which should be not less than 400 grams for boosters EV-1, EV-1Y, EV-1M, EV-1M-PB and not less than 250 cm³ for EV-13M and EV-14M (EV-5, EV-8, EV-13, and EV-14 have no suction);
3. uniform speed of motion of the operating rod in either direction;

4. inner leakage with the piston in neutral position (not more than 1100 cm³/min. for boosters EV-1, EV-1Y, EV-1M, EV-1M-PB, and not more than 600 cm³/min. for other boosters).

Notes: 1. Disassembly and assembly of hydraulic boosters should be carried out in closed premises on a clean table covered with organic glass, textolite or vinyplast.

2. If the aircraft is not flown, pump the fluid through the hydraulic boosters from the aircraft pump or a ground installation after every 10 days, for which purpose build up working pressure and shift the control stick not less than 5 - 10 times throughout the entire operating range.

3. In case of hydraulic boosters EV-13M, EV-14M and EV-14MC check operation of the duplicate slide valve after every 10 flying hours (or after every 10 days if the plane is not flown). With this purpose jam the main slide valve by means of a clamp and yoke and keep on moving the control stick to extreme position within a period of 2 - 3 minutes. The effort required to deflect the control stick in one direction may be greater than in the other direction.

4. When priming the hydraulic system, do not under any circumstances mix the used operating fluid with the fresh fluid.

Replacement of the operating fluid in hydraulic booster systems should be carried out in periods prescribed by appropriate instructions.

IX. CORROSION-PREVENTIVE TREATMENT AND STORAGE OF HYDRAULIC BOOSTERS

Hydraulic boosters are subjected to corrosion-preventive treatment when they are removed from aircraft before their service life has expired, when they are returned to the Manufacturer for some reason or other and also when they are to be placed for long-term storage in a depot.

Subject to corrosion-preventive treatment are both the inner cavities and external surfaces of the hydraulic booster. Before applying the corrosion-preventive compound, check the hydraulic booster by external inspection for corrosion and damaged anti-corrosive coating.

1. Corrosion-Preventive Treatment of Inner Cavities

1. Fill the inner cavity of the hydraulic booster with clean operating fluid AMT-10; the procedure should be performed on a special stand used for corrosion-preventive treatment.

2. Fit caps on the return and delivery pipe connections.

Note: The operating fluid AMT-10 which is used for filling the inner cavities of hydraulic boosters should be brought to the stand in the Manufacturer's vessels provided with seals.

2. External Corrosion-Preventive Treatment

3. Corrosion-preventive treatment of hydraulic boosters should be carried out in a special room not later than 4 hours after the booster is removed from the aircraft.

4. Wipe carefully the hydraulic booster with a clean cloth moistened in gasoline E-70 and dry it in the air till the smell of gasoline is no longer felt.

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5. When dealing with hydraulic boosters EV-1V and EV-1W, move the cylinder to the extreme position, towards the end-piece. In hydraulic boosters EV-10E and EV-10M it is necessary to mask the paint-coated pipes to protect them from the corrosion-preventive compound. If pipes are not masked, the corrosion-preventive compound must be removed from them after completing the treatment.

6. Apply the corrosion-preventive compound in two layers. The first layer should be applied by placing hydraulic boosters in petrolatum heated to a temperature of 70 - 80°C and keeping them there in the course of:

- 3 - 5 minutes for boosters EV-1V, EV-1W, EV-1M-EV-1N;
- 3 - 5 minutes for boosters EV-10, EV-10E, EV-10M;
- 4 - 6 minutes for EV-5A, EV-8A, EV-13N, EV-14M and EV-14MC boosters

Having cooled the hydraulic booster to 25 - 35°C, apply the second layer of anti-corrosive compound. With this purpose dip the booster into another bath with petrolatum at 55 - 65°C and immediately remove it from the bath.

7. Let the excess compound trickle down, after which place the hydraulic booster on paraffine paper and allow the petrolatum coating to cool down. This done, remove the masking or the corrosion-preventive compound from the pipes of hydraulic boosters EV-10E and EV-10M.

Note: When applying the corrosion-preventive compound, keep inner cavities of the boosters well protected against petrolatum and gasoline.

8. Wrap the hydraulic boosters in 2 layers of paraffine paper and place them in cardboard cases.

9. Paraffine the cases by placing them into a compound consisting of 80 per cent of paraffine and 20 per cent of ceresine heated to a temperature of 70 - 80°C.

10. Pack the hydraulic boosters in wooden boxes.

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Note: All the boxes and packing materials for hydraulic boosters must comply with appropriate Specifications.

During the preparation for corrosion-preventive treatment and in the process of the treatment do not take boosters by unprotected hands. Use should be made of clean knitted or cotton gloves, or paraffine paper.

3. Removal from Storage

11. Unpack the hydraulic booster and remove the wrapping paper.

12. Remove the corrosion-preventive compound by placing the boosters in aircraft or transformer oil heated to a temperature of 70 - 80°C and keeping them in the oil in the course of:

- 7 - 12 minutes for boosters EV-1V;
- 9 - 14 minutes for boosters EV-10E, EV-10M and EV-10;
- 10 - 15 minutes for boosters EV-5, EV-8, EV-13N, EV-14M and EV-14MC.

Wipe the booster with a piece of clean cloth moistened with gasoline E-70. Wash the booster head in fluid AMT-10 heated to a temperature of 70 - 80°C in the course of 0.5 - 1 minute.

13. Wipe the hydraulic booster dry with a piece of soft clean cloth.

Note: Do not remove the corrosion-preventive compound from the cylinder bushing-to-cylinder joint surfaces of hydraulic boosters EV-1V and EV-1W. When processing the boosters, keep the booster inner cavity well protected against gasoline and petrolatum.

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14. Drain AMT-10 fluid from the inner cavity of the booster.

4. Storage of Hydraulic Boosters in Depots

15. Hydraulic boosters should be kept in Air Force depots and in the Consumer's store rooms in the Manufacturer's packaging.

16. The store premises must be heated and well ventilated. Relative humidity of the air in the store room must be not over 70 per cent. A short time rise of humidity up to 80 per cent may be tolerated.

The air temperature must be within the range of $+10^{\circ}$ to $+35^{\circ}\text{C}$. Sharp fluctuations of the ambient temperature and air humidity are not permissible.

17. The store premises must be separated from production shops and protected against the penetration of various vapours and gases which are likely to cause corrosion. Storage of chemicals, acids, alkalis and storage batteries in one room with hydraulic boosters is not permissible.

18. The windows of the store room must be provided with louvers or screens to protect hydraulic boosters from the action of direct sun rays.

19. Storage of hydraulic boosters in depots must be supervised by a representative of the Inspection Department; he must see to it that materials are subjected to laboratory analyses and that anti-corrosion treatment and storage requirements are duly complied with.

Note: Petrolatum may be substituted by aircraft oils WD-20 or MK-22 (State Standard FOCT 1013-49) mixed with 6 - 10 per cent of ceresine and heated to a temperature of $60 - 70^{\circ}\text{C}$.

All the materials used for corrosion-preventive treatment must have a Certificate and a laboratory test record certifying that they are fit for use.

APPENDICES

APPENDIX 1

Specification of type EV-10 hydraulic booster units

No.	Description	EV-10 booster unit	EV-10B and EV-10M booster units	EV-5A booster unit	EV-8A booster unit
1	2	3	4	5	6
1	Piston unit	Д02-0	Д02-0	П02-0	К02-0
2	Head unit	Г03-0	Д03-0	П03-0	К03-0
3	Hydraulic booster pipe unit	-	Д04-0 in EV-10B Д05-0 in EV-10M	-	-
4	Bar (unit)	-	-	П04-0	-
5	Fork unit	-	-	-	К04
6	Head sub-unit	Д06-0	Д06-0	П06-0	Д06-0
7	Automatic delivery valve	-	Д07-0	-	-
8	Automatic drain valve	-	Д08-0	-	-
9	Plunger unit	-	-	П07	П07-0
10	Pipe connection unit	-	-	П08	П08-0
11	Bell crank unit	Д09-0	Д09-0	П09-0	Д09-0
12	Slide valve unit	Д10-0	Д10-0	П10-0	П10-0
13	Ring unit	Д11-0	Д11-0	Д11-0	Д11-0
14	Filter	Д12-0	Д12-0	Д12-0	Д12-0
15	Body sub-unit	Д13-0	Д13-0	Д13-0	Д13-0

1	2	3	4	5	6
16	Slide valve unit	Д14-0	Д14-0	П14-0	П14-0
17	End piece unit	Д15-0	Д15-0	Д15-0	Д15-0
18	Needle unit	Д16-0	Д16-0	Д16-0	Д16-0
19	Seal unit	БУ-1063Б	БУ-1063Б	БУ-1063Б	БУ-1063Б
20	Cylinder nut wrench	Д20-0	Д20-0	П20-0	Д20-0
21	Assembly cones	Д21-0	Д21-0	Д21-0	Д21-0
22	Hydraulic booster assembly	Г00-0	Д00-0	П00-0	К00-0
23	Hydraulic booster overall drawing	Г01-0	Д01-0	П01-0	К01-0

Specification of type БУ-10 hydraulic booster parts

No.	Description	БУ-10, БУ-10Б and БУ-10М booster parts	БУ-5А booster part	БУ-8А booster part
1	2	3	4	5
1	Locking	Д00-1		
2	Locknut	Д00-2	Д00-1	Д00-1
3	Cylinder	Д00-3	Д00-2	Д00-2
4	Sealing ring	Д00-4	П00-3	К00-3
5	Cylinder nut	Д00-5	П00-4 П00-5	Д00-4 К00-5

1	2	3	4	5
6	Sealing ring	Д00-6	П00-6	П00-6
7	Sealing ring	Д00-7	П00-7	-
8	Sealing ring	Д00-8	Д00-8	Д00-8
9	Sealing ring	Д00-9	Д00-9	Д00-9
10	Sealing washer	Д00-10	Д00-10	Д00-10
11	Plug	Д00-11	Д00-11	Д00-11
12	Fork	Г00-12	-	-
13	Piston	Д02-15	П02-15	К02-15
14	Distributor	Д02-16	П02-16	-
15	Sealing ring	Д02-17	П02-17	Д02-17
16	Spring	-	П02-18	П02-18
17	Plug	-	П02-19	П02-19
18	Shackle	Д03-20	Д03-20	Д03-20
19	Sealing ring	Д03-21	Д03-21	Д03-21
20	Nut	Д03-22	Д03-22	Д03-22
21	Washer	Д03-23	Д03-23	Д03-23
22	Bolt	Д03-24	Д03-24	Д03-24
23	Castle nut	Д03-25	Д03-25	Д03-25
24	Washer	Д03-26	Д03-26	Д03-26
25	Washer	Д03-27	Д03-27	Д03-27
26	Bolt	Д03-28	Д03-28	Д03-28
27	Bushing	Д03-29	Д03-29	Д03-29
28	Bolt	Д03-30	Д03-30	Д03-30

1	2	3	4	5
29	Bushing	Д03-31	Д03-31	Д03-31
30	Bolt	Д03-32	Д03-32	Д03-32
31	Nut	Д03-33	Д03-33	Д03-33
32	Bolt	Д03-34	Д03-34	Д03-34
33	Metering valve plunger	Д03-35	Д03-35	Д03-35
34	Plunger	Д03-36	Д03-36	Д03-36
35	Gasket	Д03-37	Д03-37	Д03-37
36	Metering valve spring	Д03-38	Д03-38	Д03-38
37	Plug	Д03-39	Д03-39	Д03-39
38	Stop	Д03-40	Д03-40	Д03-40
39	Pipe connection	Д03-41	-	-
40	Gasket	Д03-42	Д03-42	Д03-42
41	Plunger	Д03-43	Д03-43	Д03-43
42	Spring	Д03-44	Д03-44	Д03-44
43	Felt ring	Д03-45	Д03-45	Д03-45
44	Sealing ring	Д03-46	Д03-46	Д03-46
45	Pipe connection	Д03-47	Д03-47	Д03-47
46	Spring	Д03-48	Д03-48	Д03-48
47	Swivel nipple	Г03-50	Г03-50	Г03-50
48	Swivel nipple	Г03-51	Г03-51	Г03-51
49	Pipe for hydraulic booster BY-10B	Д04-55	-	-
50	Pipe for hydraulic booster BY-10M	Д04-56	-	-

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1	2	3	4	5
51	Bar	-	П04-57	-
52	Fork	-	-	К04-58
53	Bushing	-	-	К04-59
54	Head	Д06-60	П06-60	Д06-60
55	Sleeve	Д06-61	Д06-61	Д06-61
56	Sleeve	Д06-62	Д06-62	Д06-62
57	Technological plug	Д06-63	Д06-63	Д06-63
58	Sealing ring	Д06-64	Д06-64	Д06-64
59	Sleeve	Д06-65	Д06-65	Д06-65
60	Technological plug	Д06-66	Д06-66	Д06-66
61	Sealing ring	Д06-67	Д06-67	Д06-67
62	Delivery slide valve	Д07-70	-	-
63	Pipe connection	Д07-71	-	-
64	Swivel nipple	Д07-72	-	-
65	Pipe connection	Д07-73	-	-
66	Screw	Д07-74	-	-
67	Gasket	Д07-75	-	-
68	Plug	Д07-76	-	-
69	Sleeve	Д07-77	-	-
70	Delivery valve body	Д07-78	-	-
71	Plunger	-	П07-79	-
72	Technological plug	-	П07-80	-
73	Sealing ring	-	П07-81	-

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1	2	3	4	5
74	Drain valve body	Д08-82	-	-
75	Plug	Д08-83	-	-
76	Spring	Д08-84	-	-
77	Drain slide valve	Д08-85	-	-
78	Pipe connection	-	П08-87	П08-87
79	Plug	-	П08-88	П08-88
80	Bell crank	Д09-90	П09-90	Д09-90
81	Bushing	Д09-91	Д09-91	Д09-91
82	Plug	Д10-94	Д10-94	Д10-94
83	Nut	Д11-97	Д11-97	Д11-97
84	Coupling	Д11-98	Д11-98	Д11-98
85	Nut	Д11-99	Д11-99	Д11-99
86	Fork	Д11-100	Д11-100	Д11-100
87	Filter frame	Д12-104	Д12-104	Д12-104
88	Slide valve body	Д13-107	Д13-107	Д13-107
89	Sleeve	Д13-108	Д13-108	Д13-108
90	Damper body	Д13-109	Д13-109	Д13-109
91	Pin	Д13-110	Д13-110	Д13-110
92	Screw	Д13-111	Д13-111	Д13-111
93	Slide valve	Д14-115	П14-115	П14-115
94	Sealing ring	Д14-116	Д14-116	Д14-116
95	End-piece	Д15-120	Д15-120	Д15-120
96	Needle	Д16-123	Д16-123	Д16-123

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1	2	3	4	5
97	Sealing ring	Д16-124	Д16-124	Д16-124
98	Plug	Д16-125	Д16-125	Д16-125
	<u>Erecting tools</u>			
99	Wrench body	Д20-130	П20-130	Д20-130
100	Pin	Д20-131	П20-131	Д20-131
101	Assembly cone	Д21-133	Д21-133	Д21-133
102	Cap for adjusting slide valve	Д00-135	Д20-135	Д20-135
	<u>Parts used for other assemblies</u>			
103	Ring	ЕУ-1047А	ЕУ-1047А	ЕУ-1047А
104	Locknut	ЕУ-1057А	ЕУ-1057А	ЕУ-1057
105	Sealing ring	ЕУ-1059Б	ЕУ-1059Б	ЕУ-1059Б
106	Gland	ЕУ-1061У	ЕУ-1061У	ЕУ-1061У
107	Sealing ring	ЕУ-1062	ЕУ-1062Б	ЕУ-1062Б
108	Screen	ЕУ-1072	ЕУ-1072	ЕУ-1072
109	Locking	ЕУ-1092	-	-
110	Locking	-	ЕУ-193	ЕУ-193
111	Rod assembly cone	ЕУ-1098	ЕУ-1098	ЕУ-1098
112	Bolt	ЕУ-1099	ЕУ-1099	ЕУ-1099
113	Pin	ЕУ-1524А	ЕУ-1524А	ЕУ-1524А
114	Spring ring	-	406-022А	406-022А

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1	2	3	4	5
	<u>Standard parts</u>			
115	Cotter pin	<u>1x1.5</u> State Standard FOCT 397-54	<u>1x1.5</u> State Standard FOCT 397-54	-
116	Cotter pin	<u>1.5x12</u> State Standard FOCT 397-54	<u>1.5x12</u> State Standard FOCT 397-54	-
117	Cotter pin	<u>1.5x20</u> State Standard FOCT 397-54	<u>1.5x20</u> State Standard FOCT 397-54	-
118	Union nut	1045A-5-4	-	-
119	Nipple	1046A-5-4	-	-
120	Ball bearing	98077	98077	98077
121	Hinge bearing	MM-5	MM-5	MM-5
122	Ball	<u>1/8"</u> State Standard FOCT 3722-54	<u>1/8"</u> State Standard FOCT 3722-54	<u>1/8"</u> State Standard FOCT 3722-54
123	Ball	-	<u>3/8"</u> State Standard FOCT 3722-54	<u>3/8"</u> State Standard FOCT 3722-54
124	Ball bearing	-	State Standard FOCT 3722-54 FM3 No.1202	State Standard FOCT 3722-54

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1	2	3	4	5
125	Locking wire, dia. 1 mm	-	<u>1 = 700</u> State Standard FOCT 792-41	<u>1 = 700</u> State Standard FOCT 792-41
126	Cotter pin	-	<u>2x12</u> State Standard FOCT 397-54	<u>2x12</u> State Standard FOCT 397-54
APPENDIX 2				

Specification of type EV-18M hydraulic booster units

No.	Description	EV-18M booster unit	EV-14M booster unit	EV-14MC booster unit
1	2	3	4	5
1	Distributor bushing	EV-13030M	EV-13030M	EV-13030M
2	Distributor unit	EV-13040M1	EV-14040M1	EV-14040M1
3	Piston unit	EV-13050M1	EV-14050M1	EV-14050M1
4	Pork unit	EV-13070M	-	-
5	Bar unit	-	EV-5600M	-
6	End-piece unit	-	-	EV-5600MC
7	Damper unit	EV-13080M	EV-13080M	EV-13080M
8	Plunger unit	EV-14090M1	EV-14090M1	EV-14090M1
9	Plug unit	EV-13230M	EV-13230M	EV-13230M

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1	2	3	4	5
10	Head unit	EY-36200	EY-36200	EY-36200
11	Duplicate slide valve unit	EY-36010	EY-36010	EY-36010
12	Slide valve unit	EY-38020	EY-36020	EY-36020
13	Sleeve unit	EY-36030	EY-36030	EY-36030
14	Head sub-unit	EY-36070	EY-36070	EY-36070
15	Plug unit	EY-36080	EY-36080	EY-36080
16	Pipe connection unit	EY-8700B	EY-8700B	EY-8700B
17	Shaokle unit	EY-5360D	EY-5360D	EY-5360D
18	Filter unit	D12-0	D12-0	D12-0
19	Assembly cones	D21-0	D21-0	D21-0
20	Seal unit	EY-1063B	EY-1063B	EY-1063B
21	Hydraulic booster assembly drawing	EY-13000M2	EY-14000M1	EY-14000MC
22	Hydraulic booster overall drawing	EY-13100M	EY-14100M	EY-14100MC

Specification of type EY-13M hydraulic booster parts

No.	Description	EY-13M booster part	EY-14M booster part	EY-14MC booster part
1	2	3	4	5
1	Distributor	EY-13017M1	EY-14017M1	EY-14017M1
2	Piston	EY-13018M1	EY-14018M1	EY-14018M1

1	2	3	4	5
3	Plug	EY-13022M	EY-13022M	EY-13022M
4	Washer	EY-13029M	EY-13029M	EY-13029M
5	Sealing ring	EY-13031M	EY-13031M	EY-13031M
6	Bushing	EY-13032M	EY-13032M	EY-13032M
7	Washer, adjusting	EY-13033M	EY-13033M	EY-13033M
8	Felt ring	EY-13034M	EY-13034M	EY-13034M
9	Sealing ring	EY-13035M	EY-13035M	EY-13035M
10	Gasket	EY-13036M	EY-13036M	EY-13036M
11	Gasket	EY-13037M	EY-13037M	EY-13037M
12	Locking	EY-13038M	EY-13038M	EY-13038M
13	Gasket	EY-13043M	EY-13043M	EY-13043M
14	Gasket	EY-13044M	EY-13044M	EY-13044M
15	Gasket	EY-13051M	EY-14051M	EY-14051M
16	Fork	EY-13071M	-	-
17	Castle nut	EY-13201M	EY-13201M	EY-13201M
18	Washer	EY-13202M	EY-13202M	EY-13202M
19	Cover	EY-13231M	EY-13231M	EY-13231M
20	Cylinder bushing	EY-13250A	EY-14250	EY-14250
21	Cylinder nut	EY-13251A	-	-
22	Cylinder	EY-13254A	EY-14254	EY-14254C
23	Swivel nipple	EY-13411	EY-13411	EY-13411
24	Swivel nipple	EY-13415	EY-13415	EY-13415
25	Key	EY-14002	EY-14002	EY-14002

1	2	3	4	5
26	Locking	EY-14008	EY-14008	EY-14008
27	Plunger	EY-14021	EY-14021	EY-14021
28	Spring ring	EY-14023	EY-14023	EY-14023
29	Pipe connection	EY-14025	EY-14025	EY-14025
30	Washer, adjusting	EY-14027	EY-14027	EY-14027
31	Sealing ring	EY-14029	EY-14029	EY-14029
32	Distance bushing	-	EY-14856	-
33	Head body	EY-36001A	EY-36001A	EY-36001A
34	Duplicate slide valve	EY-36002	EY-36002	EY-36002C
35	Slide valve	EY-36003	EY-36003	EY-36003
36	Sleeve	EY-36004	EY-36004	EY-36004
37	Bushing	EY-36005	EY-36005	EY-36005
38	Slide valve spring	EY-36006	EY-36006	EY-36006
39	Plug	EY-36007	EY-36007	EY-36007
40	Nut	EY-36008	EY-36008	EY-36008
41	Washer	EY-36009	EY-36009	EY-36009
42	Bell crank, right	EY-36012	EY-36012	EY-36012
43	Bell crank, left	EY-36013	EY-36013	EY-36013
44	Bushing	EY-36014	EY-36014	EY-36014C
45	Bolt	EY-36015	EY-36015	EY-36015
46	Shackle	EY-36016	EY-36016	EY-36016
47	Plug	EY-36019	EY-36019	EY-36019
48	Eccentric bolt	EY-36021	EY-36021	EY-36021
49	Gasket	EY-36071	EY-36071	EY-36071

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1	2	3	4	5
50	Locknut	D00-2	D00-2	D00-2
51	Sealing ring	D00-4	-	-
52	Sealing ring	D00-8	D00-8	D00-8
53	Sealing ring	D00-9	D00-9	D00-9
54	Nut	D00-22A	D00-22A	D00-22A
55	Washer	D03-23A	D03-23A	D03-23A
56	Washer	D03-26	D03-26	D03-26
57	Bolt	D03-32	D03-32	D03-32
58	Nut	D03-33	D03-33	D03-33
59	Gasket	D03-42	D03-42	D03-42
60	Filter frame	D12-104	D12-104	D12-104
61	Sealing ring	D14-116	D14-116	D14-116
62	Locknut	-	D00-2	D00-2
63	Sealing ring	-	D00-4	D00-4
64	Sealing ring	-	D00-7	D00-7
65	Sealing ring	D07-81	D07-81	D07-81
66	Plug	D08-88	D08-88	D08-88
67	Pipe connection	EY-8743B	EY-8743B	EY-8743B
68	Ear	-	EY-5645M	-
69	Rod end-piece	-	-	EY-5645C
70	Bolt	EY-5102D	EY-5102D	EY-5102D
71	Bushing	EY-5103D	EY-5103D	EY-5103D
72	Castle nut	EY-5106D	EY-5106D	EY-5106D

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1	2	3	4	5
73	Damper body	EV-5322D	EV-5322D	EV-5322D
74	Washer	EV-5323D	EV-5323D	EV-5323D
75	Plug	EV-5338D	EV-5338D	EV-5338D
76	Shackle	EV-5361D	EV-5361D	EV-5361D
77	Locknut	EV-1057	-	-
78	Seal	EV-1059	EV-1059	EV-1059
79	Gland	EV-1061	EV-1061	EV-1061
80	Filter screen	EV-1072	EV-1072	EV-1072
81	Pin	EV-1490	EV-1490	EV-1490
82	Bushing	EV-1628	-	-
83	Spring	ANP1-941D	ANP1-941D	ANP1-941D
84	Plug	EV-4055	EV-4055	EV-4055
85	Spring ring	406-022	406-022	406-022
86	Sealing ring	PA1-686A	-	-
87	Sealing ring	EV15-434	EV15-434	EV15-434
88	Sealing ring	EV15-445A	EV15-445A	EV15-445A
89	Sealing ring	PC1-1142	PC1-1142	PC1-1142
90	Sealing ring	PC1-1382	PC1-1382	PC1-1382
91	Sealing ring	EV-21265	EV-21265	EV-21265
<u>Erecting tools</u>				
92	Rod assembly cone	EV-1098	EV-1098	EV-1098
93	Bolt	EV-1099	EV-1099	EV-1099

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1	2	3	4	5
94	Assembly cone	A21-133	A21-133	A21-133
95	Cylinder nut wrench	EV-13045	-	-
96	Insert for wrench	-	EV-14045	EV-14045
97	Clamp	EV13M-001	EV13M-001	EV13M-001
98	Screw	EV13-002	EV13-002	EV13-002
99	Limiter	EV13M-003	EV13M-003	EV13M-003
<u>Standard parts</u>				
100	Ball bearing	980077	980077	980077
101	Hinge bearing III-12	-	State Standard	-
102	Hinge bearing III-6	State Standard	FOCT 3635-54	State Standard
		FOCT 3636-54	State Standard	FOCT 3636-54
		dia. 3/8"	FOCT 3636-54	dia. 3/8"
103	Ball	State Standard	FOCT 3722-54	State Standard
		FOCT 3722-54	FOCT 3722-54	FOCT 3722-54
		dia. 3.969	dia. 3.969	dia. 3.969
104	Ball	State Standard	State Standard	State Standard
		FOCT 3722-54	FOCT 3722-54	FOCT 3722-54
		State Standard	State Standard	State Standard
105	Cotter pin	FOCT 397-54	FOCT 397-54	FOCT 397-54
		1x12	1x12	1x12

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1	2	3	4	5
106	Cotter pin	State Standard ГОСТ 397-54 1.5x12	State Standard ГОСТ 397-54 1.5x12	State Standard ГОСТ 397-54 1.5x12
107	Cotter pin	State Standard ГОСТ 397-54 1.5x20	State Standard ГОСТ 397-54 1.5x20	State Standard ГОСТ 397-54 1.5x20
108	Cotter pin	-	State Standard ГОСТ 397-54 2x12	State Standard ГОСТ 397-54 2x12
109	Rivet	2008A50-1.6-4	2008A50-1.6-4	2008A50-1.6-4
110	Seal	H9301	H9301	H9301
111	Locking wire, dia.1 mm	1 = 700 mm	1 = 700 mm	1 = 700 mm

Specification of type EV-1V hydraulic booster units

APPENDIX 3

No.	Description	EV-1V booster part	EV-1M booster part	EV-1M-PB booster part
1	2	3	4	5
1	Piston sub-unit	EV-1100E	EV-1100E	EV-1100E
2	Piston unit	EV-1200E	EV-1200E	EV-1200E

No.	Description	EV-1V booster part	EV-1M booster part	EV-1M-PB booster part
1	2	3	4	5
2	Yoke unit	EV-1400V	EV-1400V	EV-1400V
4	Slide valve unit	EV-1500B	EV-1500M	EV-1500M
5	Slide valve sub-unit	EV-1566B	EV-1566M	EV-1566M
6	End piece unit	EV-1510B	EV-1510B	EV-1510B
7	End piece unit	EV-1600V	-	-
8	Fork unit	-	EV-1600M	EV-1600M
9	Shackle unit	EV-1700	EV-1700	EV-1700
10	Head unit	EV-1800	EV-1800	EV-1800
11	Link unit	EV-1900	EV-1900	EV-1900
12	Seal unit	EV-1063E	EV-1063E	EV-1063E
13	Filter	EV-1070	EV-1070	EV-1070
14	Cylinder bushing wrench	EV-1082V	EV-1082V	EV-1082V
15	Assembly cones	EV-1096	EV-1096	EV-1096
16	Hydraulic booster assembly drawing	EV-1001A	EV-1001M	EV-1001M-PB
17	Hydraulic booster overall drawing	EV-1002A	EV-1002M	EV-1002M-PB

Specification of type EV-1V hydraulic booster parts

No.	Part	EV-1V booster part	EV-1V EV-1M-PB boosters part	Note
1	2	3	4	5
1	Piston	EV-1101B	EV-1101B	
2	Sleeve	EV-1102B	EV-1102B	
3	Rod	EV-1205B	EV-1205B	
4	Plunger	EV-1206B	EV-1206B	
5	Sealing ring	EV-1286M	EV-1286M	
6	Cylinder	EV-1010B	EV-1010M	
7	Yoke	EV-1413V	EV-1413V	
8	Swivel nipple	EV-1414A	EV-1414A	
9	Pipe connection	EV-1415A	EV-1415A	
10	Sealing ring	EV-1416	EV-1416	
11	Bushing	EV-1488	EV-1488	
12	Threaded bushing	EV-1489	EV-1489	
13	Pin	EV-1490	EV-1490	
14	Bushing	EV-1511B	EV-1511B	
15	Slide valve	EV-1518B	EV-1518M	
16	Nut	EV-1519	EV-1519	
17	Nut	EV-1520	EV-1520	
18	Shank	EV-1521A	EV-1521A	
19	Sleeve	EV-1523A	EV-1523A	
20	Pin	EV-1524A	EV-1524A	

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1	2	3	4	5
21	End-piece	EV-1525A	EV-1525A	
22	Damper body	EV-1567B	EV-1567B	
23	Washer	EV-1568B	EV-1568B	
24	Plug	EV-1569B	EV-1569B	
25	Ball seat	EV-1578	EV-1578	
26	Locking nut	EV-1579	EV-1579	
27	End-piece	EV-1626V	-	
28	Fork	-	EV-1627	
29	Bushing	-	EV-1628	
30	Shackle	EV-1729	EV-1729	
31	Head	EV-1831B	EV-1831B	
32	Bushing	EV-1833	EV-1833	
33	Link	EV-1936	EV-1936	
34	Bolt	EV-1040	EV-1040	
35	Bolt	EV-1041	EV-1041	
36	Castle nut	EV-1042	EV-1042	
37	Washer	EV-1043	EV-1043	
38	Stop	EV-1044	EV-1044	
39	Nut	EV-1045	EV-1045	
40	Ring	EV-1047A	EV-1047A	
41	Coupling bolt	EV-1048	EV-1048	
42	Spherical nut	EV-1049	EV-1049	
43	Plug	EV-1050	EV-1050	

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1	2	3	4	5
44	Sealing gasket	EY-1051A	EY-1051A	
45	Adjusting washer	EY-1054A	EY-1054A	
46	Locknut	EY-1057A	EY-1057A	
47	Cylinder bushing	EY-1058E	EY-1058E	
48	Seal	EY-1059E	EY-1059E	
49	Gland	EY-1061Y	EY-1061Y	
50	Sealing ring	EY-1062E	EY-1062E	
51	Needle (for EY-1MBP and EY-1064H-PB)	EY-1064	EY-1064	
52	Filter frame	EY-1071	EY-1071	
53	Filter screen	EY-1072	EY-1072	
54	Lock	EY-1076	EY-1076	
55	Screw	EY-1077	EY-1077	
56	Screw	EY-1079	EY-1079	
57	Screw	EY-1080A	EY-1080A	
58	Sealing ring	EY-1087	EY-1087	
59	Bushing	EY-1091A	EY-1091A	
60	Screw	EY-1009	EY-1009	
61	Lock	EY-1092	EY-1093	
<u>Breasting tools</u>				
62	Wrench body	EY-1083V	EY-1083V	
63	Pin	EY-1084Y	EY-1084Y	
64	Sleeve cleaning rod	EY-1095	EY-1095	

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1	2	3	4	5
65	Piston assembly cone	EY-1097	EY-1097	
66	Rod assembly cone	EY-1098	EY-1098	
67	Bolt	EY-1099	EY-1099	
<u>Borrowed parts</u>				
68	Sealing ring	Д00-6	Д00-6	From EY-10
69	Sealing ring	Д00-7	Д00-7	The same
70	Sealing ring	Д02-17	Д02-17	From EY-10
71	Sealing ring	Д00-8	Д00-8	From EY-10
72	Sealing ring	Д16-124	Д16-124	From EY-10
<u>Standard parts and parts used for other assemblies</u>				
73	Ball bearing No.1006	State Standard ГОСТ 5720-51	State Standard ГОСТ 5720-51	
74	Ball bearing No.6	State Standard ГОСТ 6121-39	State Standard ГОСТ 6121-39	
75	Ball	3/16"	3/16"	
76	Cotter pin	State Standard ГОСТ 397-54	State Standard ГОСТ 397-54	
		1.5x15	1.5x15	
77	Cotter pin	State Standard ГОСТ 397-54	State Standard ГОСТ 397-54	
		1.5x20	1.5x20	
78	Locking wire, dia.1 mm	1 = 400 mm	1 = 400 mm	

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APPENDIX 4

Individual set of spare parts for type EV-10 hydraulic boosters

No.	Part	Description	Quantity per one booster			
			EV-5A	EV-8A	EV-10A	EV-10B and EV-10H
1	2	3	4	5	6	7
1	Д12-0	Filter (unit)	1	1	1	1
2	П20-0	Wrench for cylinder nut	1	-	-	-
3	Д20-0	Wrench for cylinder	-	-	1 for	1 for
4	Д21-0	Assembly cone	1 for	1 for	10 units	10 units
5	Д00-1	Locking device	2 units	2 units	1 for	1 for
6	П00-4	Sealing ring	2	2	10 units	10 units
7	Д00-4	Sealing ring	3	2	2	2
8	Д00-6	Sealing ring	-	-	2	2
9	П00-6	Sealing ring	-	-	2	2
10	П00-7	Sealing ring	4	4	-	-
11	Д00-7	Sealing ring	3	-	-	-
12	Д00-8	Sealing ring	-	-	4	4
13	Д00-9	Sealing ring	1	1	3	3
14	Д00-10	Sealing washer	6	6	6	9
			2	2	2	2

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1	2	3	4	5	6	7
15	Д00-17	Sealing ring	-	2	2	2
16	П00-17	Sealing ring	2	-	-	-
17	Д03-21	Sealing ring	3	3	3	3
18	Д03-37	Gasket	4	4	4	4
19	Д03-42	Gasket	9	9	9	9
20	Д03-45	Felt ring	8	8	8	8
21	Д03-46	Sealing ring	4	4	2	2
22	Д07-45	Gasket	-	-	-	8
23	Д14-116	Sealing ring	6 (2 pcs of each diameter)	6 (2 pcs of each diameter)	6 (2 pcs of each diameter)	6 (2 pieces of each diameter)
24	Д16-24	Sealing ring	2	2	2	2
25	Д00-135	Cap for adjusting slide valve	1 for	1 for	1 for	1 for
			2 units	2 units	2 units	2 units
26	EV-1047A	Ring	2	2	4	4
27	EV-1062B	Sealing ring	-	3	3	3
28	EV-1063B	Sealing ring	4	4	4	4
29	EV-1092	Lock	-	-	3	3
30	EV-1093	Lock	3	3	-	-
31	406-022	Spring ring	1	1	-	-
32		Cotter pin	3	3	3	3

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State Standard
ГОСТ 397-54

1	2	3	4	5	6	7
33	1.5x12 State Standard ГОСТ 397-54	Cotter pin	4	4	4	4
34	1.5x20 State Standard ГОСТ 397-54	Cotter pin	6	6	6	6
35	2x12 State Standard ГОСТ 397-54	Cotter pin	2	2	2	2
36	-	Wire dia. 1 mm	1=700 mm	1=700 mm	1 = 700mm	1 = 700 mm

APPENDIX 5

Individual set of spare parts for type EV-13M hydraulic boosters

No.	Designa- tion	Description	Quantity	Note
1	2	3	4	5
1	EV-13034M	Felt ring	8	
2	EV-13036M	Gasket	8	
3	EV-13037M	Gasket	2	
4	EV-13038M	Lock	2	

1	2	3	4	5
5	EV-13043M	Gasket	2	
6	EV-13044M	Gasket	4	
7	EV-13045M	Wrench for cylinder nut	1	For EV-13M
8	EV-13051M	Gasket	2	Ditto
9	EV-14002	Key	3	
10	EV-14003	Lock	2	
11	EV-14029M	Sealing ring	2	
12	EV-14045	Insert for wrench	1	For EV-14M and EV-14MC
13	EV-14051M	Gasket	2	Ditto
14	EV-36071M	Gasket	1	
15	EV15-434	Sealing ring	5	
16	EV15-445	Sealing ring	2	
17	A12-0	Filter (unit)	1	
18	A21-0	Assembly cones	1	For 2 units
19	A00-4	Sealing ring	1	For EV-13M
20	A00-8	Sealing ring	2	
21	A00-9	Sealing ring	5	
22	A00-42	Gasket	9	
23	A14-116	Sealing ring	3	
24	A0-124	Sealing ring	2	
25	A0-4	Sealing ring	1	For EV-14 and EV-14MC

1	2	3	4	5
26	Π00-7	Sealing ring	3	For EV-14 and EV-14MC
27	Π07-81	Sealing ring	2	
28	EV-1063B	Sealing unit	2	
29	EV-5178B	Sealing ring	2	
30	PA1-686A	Sealing ring	3	
31	EV-21265	Sealing ring	2	
32	ΠC1-1142	Sealing ring	2	
33	1.5x12	Cotter pin	4	
34	1.5x20	Cotter pin	6	
35	2x12	Cotter pin	2	For EV-14M and EV-14MC
36	EV-13M-001	Clamp	1	
37	EV-13-002	Screw	1	
38	EV-23M-003	Limiter	1	

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APPENDIX 6

Individual set of spare parts for type EV-1V hydraulic boosters

No.	Designation	Description	Quantity	Note
1	2	3	4	5
1	EV-1009	Screw	1	
2	EV-1045	Nut	2	

1	2	3	4	5
3	EV-1044	Stop	2	
4	EV-1047A	Ring	1	
5	EV-1062B	Sealing ring	2	
6	EV-1063B	Sealing unit	2	
7	EV-1070	Filter	1	
8	EV-1082V	Wrench for cylinder bushing	1	
9	EV-1087	Sealing ring	5	
10	EV-1092	Lock	2	For EV-1V
11	EV-1093	Lock	-	For EV-1M and EV-1M-PB
12	EV-1095	Sleeve cleaning rod with spare strip	1	
13	EV-1096	Assembly cones for seals	1 for 4 boosters	
14	EV-1416	Sealing ring	5	
15	Π00-6	Sealing ring	1	
16	Π00-7	Sealing ring	2	
17	Π00-8	Sealing ring	2	
18	Π02-17	Sealing ring	1	
19	State Standard ГОСТ 397-54 1.5x20	Cotter pin	1	

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1	2	3	4	5
20	State Standard <u>GOST 397-54</u> 1.5x15	Cotter pin	1	
21	dia. 1 mm	Locking wire	1 = 400 mm	

APPENDIX 7

List of erecting tools

No.	Description	EV-1V EV-1M EM-1M-PB	EV-5A	EV-8A	EV-10 EV-10B EV-10M	EV-13M	EV-14M EV-14MC
1	2	3	4	5	6	7	8
1	Wrench for cylinder nut	EV-1082V	Д20-0	Д20-0	Д20-0	EV-13045	-
2	Insert for wrench	-	-	-	-	-	EV-14045
3	Assembly cone	EV-1096	Д21-0	Д21-0	Д21-0	Д21-0	Д21-0
4	Cap for adjusting slide valve	-	Д00-135	Д00-135	Д00-135	-	-
5	Rod for sleeve cleaning	EV-1095	-	-	-	-	-

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1	2	3	4	5	6	7	8
6	Clamp	-	-	-	-	EV13M-001	EV13M-001
7	Screw	-	-	-	-	EV13-002	EV13-002
8	Limiter	-	-	-	-	EV13M-003	EV13M-003

APPENDIX 8

Table of clearances and fits.Boosters EV-10, EV-10B and EV-10M

Designation	Description	Size and fit	Maximum deviation, mm		Clearance(+) and interference(-) without coating, mm	Clearance(+) and interference(-) with coating, mm
			upper	lower		
1	2	3	4	5	6	7
Д00-3	Cylinder	56A	+0.03	0	+0.175	
Д02-15	Piston	56M	-0.095	-0.145	+0.095	
Д00-8	Cylinder	32A	+0.027	0		+0.077
Д02-15	Piston	32M	-0.025	-0.05		+0.025
Д00-3	Cylinder	56A	+0.03	0	+0.062	
Д00-5	Cylinder nut	56D	-0.012	-0.032	+0.012	
Д00-5	Cylinder nut	32A	+0.027	0		+0.077
						+0.025

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1	2	3	4	5	6	7
Д02-15	Piston	32X	-0.025	-0.05		+0.077 +0.025
Д02-15	Piston	22A ₃	+0.045	0	+0.175	
Д02-16	Distributor	22Ш ₃	-0.06	-0.13	+0.06	
Д03-20	Shackle	8A	+0.016	0	+0.031	
Д03-24	Bolt	8Д	-0.005	-0.015	+0.005	
Д03-20	Shackle	7A	+0.016	0	+0.031	
Д03-32	Bolt	7Д	-0.005	-0.015	+0.005	
ШМ-5	Hinge bearing	5	0	-0.01	+0.008	
Д03-28	Bolt	5C	0	-0.008	-0.01	
Д03-31	Bushing	8A	+0.016	0	+0.031	
Д03-24	Bolt	8Д	-0.005	-0.015	+0.005	
980077	Ball bearing	7	0	-0.01	+0.015	
Д03-32	Bolt	7Д	-0.005	-0.015	-0.005	
Д06-60	Head	28A	+0.023	0	+0.063 +0.02	+0.053 -0.004
Д02-15	Piston	28X	-0.02	-0.04	+0.063 +0.02	+0.053 -0.004
Д06-60	Head	28A	+0.023	0	+0.153 +0.06	
Д02-16	Distributor	28Ш ₃	-0.06	-0.13		
Д06-60	Head	11.7A ₃	+0.035	0	+0.847 +0.8	+0.841 +0.786
Д03-31	Bushing	10.9C	0	-0.012		

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1	2	3	4	5	6	7
Д06-60	Head	23A	+0.023	0	-0.015 -0.03 (to be adjusted by selection)	
Д06-61	Sleeve	23ПД	+0.039	+0.035		
Д06-60	Head	18A	+0.019	0	-0.015 -0.025 (to be adjusted by selec- tion)	
Д06-62	Sleeve	18ПД	+0.032	+0.02		
Д06-60	Head	23A	+0.023	0	-0.015 -0.03 (to be adjusted by selection)	
Д06-65	Sleeve	23ПД	+0.039	+0.025		
Д06-60	Head	19T	0	-0.023	+0.009 -0.023	
980077	Ball bearing	19	0	-0.009		
Д06-60	Head	5A ₃	+0.025	0	+0.047 +0.01	
Д13-110	Pin	5X	-0.01	-0.022		
Д06-61	Sleeve	19A	+0.023	0	+0.010 +0.015 (to be adjusted by selec- tion)	
Д03-43	Plunger	19Д	-0.008	-0.022		
Д06-61	Sleeve	8A	+0.016	0	+0.076 +0.035	+0.07 +0.021
Д03-40	Stop	8Ш	-0.035	-0.06		

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1	2	3	4	5	6	7
Δ06-65	Sleeve	18A	+0.019	0	+0.074	+0.068
Δ03-40	Stop	18A	-0.03	-0.055	+0.03	+0.016
Δ06-62	Sleeve	14A	+0.019	0	+0.089	
Δ03-86	Plunger	14X ₃	-0.02	-0.07	+0.02	
Δ06-62	Sleeve	7.5A	+0.016	0	+0.012	
Δ03-35	Valve metering plunger	7.5A	-0.005	-0.015	+0.007 (to be adjusted by selection)	
Δ09-90	Bell crank	8A	+0.016	0	+0.031	
Δ03-24	Bolt	8A	-0.005	-0.015	+0.005	
Δ09-90	Bell crank	5H	+0.004	-0.009	+0.012	
Δ03-28	Bolt	5C	0	-0.008	-0.009	
Δ09-90	Bell crank	7A	+0.016	0	+0.031	
Δ03-32	Bolt	7A	-0.005	-0.015	+0.005	
Δ09-90	Bell crank	15A	+0.019	0	-0.034	
Δ09-91	Pushing	15H _p	+0.034	+0.022	-0.003	
Δ13-107	Slide valve body	18A	+0.019	0	-0.015	
Δ13-108	Sleeve	18H ₁	+0.032	+0.02	-0.025 (to be adjusted by selection)	
Δ11-100	Fork	5H	+0.004	-0.009	+0.007	
Δ03-34	Bolt	5H2a	+0.009	-0.003	-0.018	

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1	2	3	4	5	6	7
Δ13-107	Slide valve body	26A	+0.023	0	+0.005 -0.003 (to be adjusted by selection)	
Δ13-109	Damper body	26H	+0.017	+0.002		
Δ13-107	Slide valve body	5CH _p	-0.03	-0.045	-0.008 -0.035	
Δ13-110	Pin	5X	-0.01	-0.022		
Δ13-108	Sleeve	11A	+0.019	0	+0.010 +0.007 (to be adjusted by selection)	
Δ14-115	Slide valve	11C	0	-0.012		
Δ13-109	Damper body	20A	+0.023	0	+0.027 +0.020 (to be adjusted by selection)	
Δ14-115	Slide valve	20A	-0.008	-0.022		
Δ13-109	Damper body	11A	+0.019	0	+0.049 +0.02	
Δ14-115	Slide valve	11	-0.02	-0.03		
Δ14-115	Slide valve	5A	+0.013	0	+0.006 +0.003 (to be adjusted by selection)	
Δ03-34	Bolt	5H2a	+0.009	-0.003		
Δ14-115	Slide valve	6A	+0.013	0	+0.025 +0.004	
Δ16-123	Needle	6A	-0.004	-0.012		
Δ13-120	End-piece	14F	0	-0.019	+0.008 -0.019	
Δ11-5	Hinge bearing	14	0	-0.008		

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1	2	3	4	5	6	7
PO3-51	Swivel nipple (for booster EY-10)	11A ₃	+0.035	0	+0.105 +0.02	
D12-104	Filter frame (for booster EY-10)	11X ₃	-0.02	-0.07		
DO7-78	Pipe connection (for boosters EY-10B, EY-10M)	11A ₃	+0.035	0	+0.105 +0.02	
D12-104	Filter frame (for boosters EY-10B, EY-10M)	11X ₃	-0.02	-0.07		
DO7-77	Sleeve (for boosters EY-10B, EY-10M)	13A	+0.019	0	+0.012 +0.008 (to be adjust- ed by selec- tion)	
DO7-70	Delivery slide valve (for boosters EY-10B, EY-10M)	13П	+0.006	-0.006		

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1	2	3	4	5	6	7
DO7-77	Sleeve (for boosters EY-10B, EY-10M)	13A	+0.019	0	+0.012 +0.008 (to be adjusted by selection)	
DO8-85	Drain slide valve (for boosters EY-10B, EY-10M)	13П	+0.006	-0.006		
DO7-78	Delivery valve body (for boosters EY-10B, EY-10M)	20.5A ₃	+0.045	0	+0.130 +0.025	
DO7-71	Pipe connection	20.5X ₃	-0.025	-0.085		
DO7-78	Delivery valve body (for boosters EY-10B, EY-10M)	16A ₃	+0.035	0	+0.105 +0.02	
DO7-73	Pipe connection (for boosters EY-10B, EY-10M)	16X ₃	+0.02	+0.07		
DO7-78	Delivery valve body (for boosters EY-10B, EY-10M)	16A ₃	+0.035	0	+0.105 +0.02	

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1	2	3	4	5	6	7
Д07-76	Plug (for boosters BY-10Б, BY-10М)	16X ₃	-0.02	+0.07	+0.105 +0.02	
Д07-78	Delivery valve body (for boosters BY-10М)	20.5A ₃	+0.045	0		
Д07-77	Sleeve (for boosters BY-10Б, BY-10М)	20.5X	-0.02	-0.04	+0.085 +0.02	
Д08-82	Drain valve body (for boosters BY-10Б, BY-10М)	20.5A ₃	+0.045	0		
Д07-71	Pipe connection (for boosters BY-10Б, BY-10М)	20.5X ₃	-0.025	-0.085	+0.130 +0.025	
Д08-82	Drain valve body (for boosters BY-10Б, BY-10М)	16A ₃	+0.035	0	+0.105 +0.02	

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1	2	3	4	5	6	7
Д07-73	Pipe connection (for boosters BY-10Б, BY-10М)	19X ₃	-0.02	-0.07	+0.105 +0.02	
Д08-82	Drain valve body (for boosters BY-10Б, BY-10М)	20.5A ₃	+0.045	0		
Д07-77	Sleeve (for boosters BY-10Б, BY-10М)	20.5X	-0.02	-0.04	+0.085 +0.02	
Д08-82	Drain valve body (for boosters BY-10Б, BY-10М)	16A ₃	+0.035	0		
Д08-83	Plug (for boosters BY-10Б, BY-10М)	16X ₃	-0.02	-0.07	+0.105 +0.02	

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APPENDIX 9

Table of clearances and fits. Boosters EV-5A and EV-8A

Designation	Description	Size and fit	Maximum deviation, mm		Clearance (+) and interference (-) without coating, mm	Clearance (+) and interference (-) with coating, mm
			upper	lower		
1	2	3	4	5	6	7
Π00-8	Cylinder	75A	+0.03	0		+0.175
Π02-15	Piston	75Π	-0.095	-0.145		+0.095
K00-8	Cylinder	56A	+0.03	0		+0.175
K02-15	Piston	56Π	-0.095	-0.145		+0.095
Π00-8	Cylinder	32A	+0.027	0		+0.077
Π02-15	Piston	32X	-0.025	-0.050		+0.025
K00-8	Cylinder	32A	+0.027	0		+0.077
K02-15	Piston	32X	-0.025	-0.050		+0.025
Π00-8	Cylinder	75A	+0.03	0		+0.05
Π00-5	Cylinder nut	75C	0	-0.02		0
K00-8	Cylinder	56A	+0.03	0		+0.062
K00-5	Cylinder nut	56Д	+0.012	-0.032		+0.012
Π00-5	Cylinder nut	32A	+0.027	0		+0.077
Π02-15	Piston	32X	-0.025	-0.050		+0.025
K00-5	Cylinder nut	32A	+0.027	0		+0.077
K02-15	Piston	32X	-0.025	-0.050		+0.025

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1	2	3	4	5	6	7
Π02-15	Piston	23A ₃	+0.045	0	+0.175	
Π02-16	Distributor	23Π ₃	-0.06	-0.13	+0.06	
K02-15	Piston	23A ₃	+0.045	0	+0.175	
Π02-16	Distributor	23Π ₃	-0.06	-0.13	+0.06	
K02-15	Piston	23A ₃	+0.045	0	+0.175	
Π02-16	Distributor	23Π ₃	-0.06	-0.13	+0.06	
Д03-20	Shackle	8A	+0.016	0	+0.031	
Д03-24	Bolt	8Д	-0.005	-0.015	+0.005	
Д03-20	Shackle	7A	+0.016	0	+0.031	
Д03-32	Bolt	7Д	-0.005	-0.015	+0.005	
ММ-5	Hinge bearing	5	0	-0.01	+0.008	
Д03-28	Bolt	5C	0	-0.008	-0.01	
Π03-31	Bushing	8A	+0.016	0	+0.031	
Д03-24	Bolt	8Д	-0.005	-0.015	+0.005	
Д03-31	Bushing	8A	+0.016	0	+0.031	
Д03-24	Bolt	8Д	-0.005	-0.015	+0.005	
980077	Ball bearing	7	0	-0.01	+0.015	
Д03-32	Bolt	7Д	-0.005	-0.015	-0.005	
Π06-60	Head	28A	+0.023	0	+0.063	+0.053
Π02-15	Piston	28X	-0.02	-0.04	+0.02	-0.004
Д06-60	Head	28A	+0.023	0	+0.063	+0.053
K02-15	Piston	28X	-0.02	-0.04	+0.02	-0.004

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1	2	3	4	5	6	7
Π06-60	Head	28A	+0.023	0	+0.153	
Π02-16	Distributor	28Π ₃	-0.06	-0.13	+0.006	
Π06-60	Head	28A	+0.023	0	+0.153	
Π02-16	Distributor	28Π ₃	-0.06	-0.13	+0.016	
Π06-60	Head	11.7A ₃	+0.035	0	+0.947	+0.941
Π03-31	Bushing	10.8e	0	-0.012	+0.9	+0.886
Π06-60	Head	11.7A ₂	+0.035	0	+0.847	+0.841
Π03-31	Bushing	10.9C	0	-0.012	+0.8	+0.786
Π06-60	Head ^{1/}	23A	+0.023	0	-0.015	
Π06-61	Sleeve ^{1/}	23ΠΠ	+0.039	+0.025	-0.03 (to be adjusted by selection)	
Π06-60	Head	23A	+0.023	0	-0.015	
Π06-61	Sleeve	23ΠΠ	+0.039	+0.025	-0.03 (to be adjusted by selection)	
Π06-60	Head	18A	+0.019	0	-0.015	
Π06-62	Sleeve	18ΠΠ	+0.032	+0.02	-0.025 (to be adjusted by selection)	

^{1/} Permissible deviation in individual part +0.2 mm.

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1	2	3	4	5	6	7
Π06-60	Head	18A	+0.019	0	-0.015	
Π06-62	Sleeve	18ΠΠ	+0.032	+0.02	-0.025 (to be adjusted by selection)	
Π06-60	Head	23A	+0.023	0	-0.015	
Π06-65	Sleeve	23ΠΠ	+0.039	+0.025	-0.03 (to be adjusted by selection)	
Π06-60	Head ^{1/}	23A	+0.023	0	-0.015	
Π06-65	Sleeve ^{1/}	23ΠΠ	+0.039	+0.025	-0.03 (to be adjusted by selection)	
Π06-60	Head	19T	0	-0.023	+0.009	
980077	Ball bearing	19	0	-0.009	-0.023	
Π06-60	Head	19T	0	-0.023	+0.009	
980077	Ball bearing	19	0	-0.009	-0.023	
Π06-60	Head	5A ₃	+0.025	0	+0.047	
Π13-110	Pin	5X	-0.01	-0.022	+0.01	
Π06-60	Head	5A ₃	+0.025	0	+0.047	
Π13-110	Pin	5X	-0.01	-0.022	+0.01	
Π06-61	Sleeve ^{2/}	19A	+0.023	0		+0.015
Π06-43	Piunger ^{2/}	19Π	-0.008	-0.022		+0.010 (to be adjusted by selection)

^{1/} Permissible deviation in individual part +0.2 mm.

^{2/} Permissible deviation in individual part +0.1 mm.

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1	2	3	4	5	6	7
Д06-61	Sleeve	8A	+0.016	0	+0.076	+0.07
Д03-40	Stop	8M	-0.035	-0.06	+0.035	+0.021
Д06-65	Sleeve	18A	+0.019	0	+0.074	+0.068
Д03-40	Stop	18M	-0.03	-0.055	+0.03	+0.016
Д06-62	Sleeve	14A	+0.019	0		+0.089
Д03-36	Plunger	14X ₃	-0.02	-0.07		+0.02
Д06-62	Sleeve ^{1/}	7.5A	+0.016	0		+0.012
Д03-35	Metering valve plunger ^{1/}	7.5M	-0.005	-0.005		+0.007 (to be adjusted by selection)
П09-90	Bell crank	8A	+0.016	0	+0.031	
Д03-24	Bolt	8M	-0.005	-0.015	+0.005	
Д03-90	Bell crank	8A	+0.016	0	+0.031	
Д03-24	Bolt	8M	-0.005	-0.015	+0.005	
П09-90	Bell crank	5H	+0.004	-0.009	+0.012	
Д03-28	Bolt	5C	0	-0.008	-0.009	
Д03-90	Bell crank	5H	+0.004	-0.009	+0.012	
Д03-28	Bolt	5C	0	-0.008	-0.009	
П09-90	Bell crank	7A	-0.016	0	+0.031	
Д03-32	Bolt	7M	-0.005	-0.015	+0.005	

1/ Permissible deviation in individual part +0.1 mm.

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1	2	3	4	5	6	7
Д03-90	Bell crank	7A	+0.016	0	+0.031	
Д03-32	Bolt	7M	-0.005	-0.015	+0.005	
П09-90	Bell crank	15A	+0.019	0	-0.034	
Д03-91	Bushing	15M _p	+0.034	+0.022	-0.003	
Д03-9	Bell crank	15A	+0.019	0	-0.034	
Д03-91	Bushing	15M _p	+0.034	+0.022	-0.003	
Д13-107	Slide valve body ^{1/}	18A	+0.019	0	-0.015	
Д13-108	Sleeve ^{1/}	18M _p	+0.032	+0.02	-0.025 (to be adjusted by selection)	
Д11-100	Fork	5H	+0.004	+0.009	+0.007	
Д03-34	Bolt	5M _{2a}	+0.009	-0.003	-0.018	
Д13-107	Slide valve body ^{2/}	26A	+0.023	0	+0.005	
Д13-109	Damper body ^{2/}	26H	+0.017	+0.002	-0.003 (to be adjusted by selection)	
Д13-107	Slide valve body	5C _{mp}	-0.03	-0.045	-0.008	
Д13-110	Pin	5X	-0.01	-0.022	-0.035	

1/ Permissible deviation in individual part +0.2 mm.

2/ Permissible deviation in individual part +0.1 mm.

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1	2	3	4	5	6	7
Д13-108	Sleeve ^{1/}	11A	+0.019	0		+0.010
П14-115	Slide valve ^{1/}	11C	0	-0.012		+0.007 (to be adjusted by selection)
Д13-109	Damper body ^{1/}	20A	+0.023	0		+0.040
П14-115	Slide valve ^{1/}	20Д	-0.008	-0.022		+0.020 (To be adjusted by selection)
Д13-109	Damper body ^{1/}	11A	+0.019	0		+0.049
П14-115	Slide valve ^{1/}	11	-0.02	-0.03		+0.020
П14-115	Slide valve	5A	+0.013	0	+0.006	
Д03-34	Bolt	5П2a	+0.009	-0.003	+0.003 (to be adjusted by selection)	
П14-115	Slide valve	6A	+0.013	0	+0.025	
Д16-123	Needle	6Д	-0.004	-0.012	+0.004	
Д15-120	End piece	14T	0	-0.019	+0.008	
ДМ-5	Hinge bearing	14	0	-0.008	-0.019	
ГО3-51	Swivel nipple	11A ₃	+0.035	0	+0.105	
Д12-104	Filter frame	11A	-0.02	-0.07	+0.02	
П02-15	Piston ^{2/}	14A	++0.019	0	+0.012	
П07-79	Plunger ^{2/}	14C	0	-0.012	+0.018 (to be adjusted by selection)	

1/ Permissible deviation in individual part +0.05 mm.

2/ Permissible deviation in individual part +0.1 mm.

1	2	3	4	5	6	7
К02-15	Piston ^{1/}	14A	+0.019	0	+0.012	
П07-79	Plunger ^{1/}	14C	0	-0.012	+0.018 (to be adjusted by selection)	
П02-15	Piston ^{1/}	19A	+0.023	0	+0.063	
П07-79	Plunger ^{1/}	19X	-0.02	-0.04	+0.020	
К02-15	Piston ^{1/}	19A	+0.023	0	+0.063	
П07-79	Plunger ^{1/}	19X	-0.02	-0.04	+0.020	
П02-15	Piston ^{1/}	21.6A ₃	+0.045	0	+0.13	
П02-19	Plug ^{1/}	21.6X ₃	-0.025	-0.085	+0.025	
К02-15	Piston ^{1/}	21.6A ₃	+0.045	0	+0.13	
П02-19	Plug ^{1/}	21.6X ₃	-0.025	-0.085	+0.025	
П04-0	Lug	35H	+0.007	-0.02	+0.018	
П13	Ball bearing	35	0	-0.011	-0.02	
No. 1202						
К04-58	Fork	11A	+0.019	0		
К04-59	Bushing	11П ₁₃	+0.075	+0.040		

1/ Permissible deviation in individual part +0.1 mm.

APPENDIX 1C

Table of clearances and fits Boosters EY-13M, EY-14 and EY-14MC

Designation	Description	Size and fit	Maximum deviation, mm		Clearance (+) and interference (-) without coating, mm	Clearance (+) and interference (-) with coating, mm	Note
			upper	lower			
1	2	3	4	5	6	7	8
EY-36001A	Head body	dia.30A	+0.023	0	+0.063 +0.02		
EY-36004	Sleeve	dia.30X	-0.02	-0.04			
EY-36001A	Head body	dia.34A	+0.027	0	+0.077 +0.025	+0.061 +0.001	
EY-36007	Plug	dia.34X	-0.025	-0.050			
EY-36007A	Head body	dia.26A	+0.023	0	+0.118		
EY-14017H1	Distributors	dia.26M	-0.06	-0.095	+0.06		
EY-36001A	Head body	dia.28A	+0.023	0	+0.063 +0.02	+0.047 -0.004	
EY-14018H1	Piston	dia.28X	-0.02	-0.04			
EY-36001A	Head body	dia.19T	0	-0.023	+0.009 -0.023		
980077	Ball bearing	dia.19	0	-0.009			
EY-36004	Sleeve	dia.19A	+0.023	0	+0.007 +0.010 (to be adjusted by selection)		
EY-36002	Duplicate slide valve	dia.19C	0	-0.014			

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1	2	3	4	5	6	7	8
EY-36002	Duplicate slide valve	dia.11A	+0.019	0	+0.010 +0.07 (to be adjusted by selection)		
EY-36003	Slide valve	dia.11	0	-0.012			
EY-36002	Duplicate slide valve	dia.16M ₃	-0.045	-0.105	+0.140 +0.045		
EY-36005	Bushing	dia.16A ₃	+0.036	0			
EY-14017M1	Distributor	dia.22M ₃	-0.06	-0.13	+0.175 +0.06		
EY-14018M1	Piston	dia.22A ₃	+0.045	0			
EY-14017M1	Distributor	dia.15X ₃	-0.02	-0.07	+0.089 +0.02		
EY-14018M1	Piston	dia.15A	+0.019	0			
EY-14018M1	Piston	dia.75M	-0.095	-0.145		+0.175 +0.095	
EY-14254	Cylinder	dia.75A	+0.03	0			
EY-14018M1	Piston	dia.32X	-0.025	-0.050		+0.077 +0.025	
EY-14254	Cylinder	dia.32A	+0.027	0			
EY-14250	Cylinder nut	dia.32A	+0.027	0		+0.077 +0.025	
EY-14018M1	Piston	dia.32X	-0.025	-0.050			
EY-14250	Cylinder nut	dia.75C	0	-0.02		+0.050 0	
EY-14254	Cylinder	dia.75A	+0.03	0			
EY-36021	Eccentric bolt	dia.8A	-0.005	-0.015	+0.003 -0.005 (to be adjusted by selection)		
EY-36012	Ball crank, right	dia.8A	+0.016	0			

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1	2	3	4	5	6	7	8
EY-86021	Eccentric belt	dia. 4H	-0.004	-0.012	+0.016		
EY-86018	Bell crank, left	dia. 4H	+0.004	-0.009	-0.005		
AOB-82	Bolt	dia. 7H	-0.005	-0.015	+0.019		
EY-86012	Bell crank, right	dia. 7H	+0.004	-0.012	-0.007		
AOB-82	Bolt	dia. 7H	-0.005	-0.015	+0.019		
EY-86018	Bell crank, left	dia. 7H	+0.004	-0.012	-0.007		
AOB-82	Bolt	dia. 7H	-0.005	-0.015	+0.019		
EY-86016	Shackle	dia. 7H	+0.004	-0.012	-0.007		
EY-86021	Eccentric belt	dia. 6C	0	-0.008	+0.017		
MM-6	Bearing hinge	dia. 6H2a	+0.009	-0.009	-0.009		
EY-86015	Bolt	dia. 8	0	-0.01	+0.026		
EY-86016	Shackle	dia. 8A	+0.016	0	0		
EY-86015	Bolt	dia. 8	0	-0.01	+0.026		
EY-86012	Bell crank, right	dia. 8A	+0.016	0	0		
EY-86015	Bolt	dia. 8	0	-0.01	+0.026		
EY-86018	Bell crank, left	dia. 8A	+0.016	0	0		
AOB-82	Bolt	dia. 7H	-0.005	-0.015	+0.015		
980077	Ball bearing	dia. 7	0	-0.01	-0.005		
MM-6	Hinge bearing	dia. 14B	0	-0.012	+0.012		
EY-86008	Slide valve	dia. 14T	0	-0.019	-0.019		
EY-5861D	Shackle	dia. 14T	0	-0.019	+0.012		
MM-6	Hinge bearing	dia. 14B	0	-0.012	-0.019		

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1	2	3	4	5	6	7	8
EY-5102D	Bolt	dia. 6C	0	-0.008	+0.021		
EY-5861D	Shackle	dia. 6A	+0.013	0	0		
EY-5102D	Bolt	dia. 6C	0	-0.008	+0.017		
MM-6	Hinge bearing	dia. 6H2a	20.009	-0.009	-0.009		
EY-5645M	Lug	dia. 22H	-0.006	-0.017	+0.008		
MM-12	Hinge bearing	dia. 22B	0	-0.014	-0.017		
EY-18411	Swivel nipple	dia. 11A ₂	+0.035	0	+0.105		
112-104	Filter frame	dia. 11X ₂	-0.02	-0.07	+0.020		
EY-98008	Slide valve	dia. 14T	0	-0.019	+0.012		
MM-6	Hinge bearing	dia. 14B	0	-0.012	-0.019		
EY-18250A	Cylinder bushing	dia. 47C	0	-0.017		+0.044	
EY-18254A	Cylinder	dia. 47A	+0.027	0		0	
EY-18250A	Cylinder bushing	dia. 32A	+0.027	0		+0.077	
EY-18018M1	Piston	dia. 32X	-0.025	-0.050		+0.025	
EY-18254A	Cylinder	dia. 32A	+0.027	0		+0.077	
EY-18018M1	Piston	dia. 32X	-0.025	-0.050		+0.025	
EY-18254A	Cylinder	dia. 47A	+0.027	0		+0.112	
EY-18018M1	Piston	dia. 47H	-0.050	-0.085		+0.050	
EY-18018M1	Piston	dia. 23A ₂	+0.045	0	-0.175		
EY-18017M1	Distributor	dia. 23M ₂	-0.06	-0.13	+0.06		
EY-36001A	Head body	dia. 28A	+0.023	0	+0.063	+0.047	
EY-18018M1	Piston	dia. 28X	-0.02	-0.04	+0.02	-0.004	

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1	2	3	4	5	6	7	8
EY-14018M1	Piston	dia.15A	+0.019	0	+0.007 +0.010 (to be adjusted by selection)		
EY-14021M1	Plunger	dia.15C	0	-0.012			
EY-14018M1	Piston	dia.21A	+0.023	0	+0.063		
EY-14021M1	Plunger	dia.21X	-0.02	-0.04	+0.02		
EY-18018M1	Piston	dia.15A	+0.019	0	+0.007 +0.010 (to be adjusted by selection)		
EY-14021M1	Plunger	dia.15C	0	-0.012			
EY-18018M1	Piston	dia.21A	+0.023	0	+0.063		
EY-14021M1	Plunger	dia.21X	-0.02	-0.04	+0.02		
EY-86004	Sleeve	dia.19A	+0.023	0	+0.010 +0.007 (to be adjusted by selection)		
EY-86002C	Duplicate slide valve	dia.19C	0	-0.014			
EY-86002C	Duplicate slide valve	dia.11A	+0.019	0	+0.010 +0.007 (to be adjusted by selection)		
EY-86008	Slide valve	dia.11	0	-0.012	+0.140 +0.045		
EY-86002C	Duplicate slide valve	dia.16M ₃	-0.045	-0.105			
EY-86005	Bushing	dia.16A ₃	-0.035	0			
EY-18254A	Cylinder	dia.32A	+0.027	0		+0.077 +0.025	
EY-18018M1	Piston	dia.32X	-0.025	-0.050			

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1	2	3	4	5	6	7	8
EY-18254A	Cylinder	dia.47A	+0.027	0		+0.112 +0.050	
EY-13018M1	Piston	dia.47M	-0.050	-0.085			
EY-13018M1	Piston	dia.23A ₃	+0.045	0	+0.175 +0.06		
EY-13017M1	Distributor	dia.23M ₃	-0.06	-0.13			
EY-36001A	Head unit	dia.28A	+0.023	0	+0.063 +0.02	+0.047 -0.004	
EY-13018M1	Piston	dia.38X	-0.02	-0.04			
EY-14018M1	Piston	dia.15A	+0.019	0	+0.007 +0.010 (to be adjusted by selection)		
EY-14021M1	Plunger	dia.15C	0	-0.012			
EY-14018M1	Piston	dia.21A	+0.023	0	+0.063 +0.02		
EY-14021M1	Plunger	dia.21X	-0.02	-0.04			
EY-13018M1	Piston	dia.15A	+0.019	0	+0.007 +0.010 (to be adjusted by selection)		
EY-14021M1	Plunger	dia.15C	0	-0.012	+0.063 +0.02		
EY-13018M1	Piston	dia.21A	+0.023	0			
EY-14021M1	Plunger	dia.21X	-0.02	-0.04			
EY-36004	Sleeve	dia.19A	+0.023	0	+0.010 +0.007 (to be adjusted by selection)		
EY-36002C	Duplicate slide valve	dia.15C	0	-0.014			

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1	2	3	4	5	6	7	8
EY-86002C	Duplicate slide valve	dia.11A	+0.019	0	+0.010 +0.007 (to be adjusted by selection)		
EY-86008	Slide valve	dia.11	0	-0.012			
EY-86002C	Duplicate slide valve	dia.16M ₃	-0.045	-0.105			
EY-86005	Bushing	dia.16A	+0.035	0	+0.140 +0.045		
EY-14018M1	Piston	dia.75M	-0.095	-0.145			
EY-14254C	Cylinder	dia.75A	+0.03	0		+0.175 +0.095	
EY-14018M1	Piston	dia.32X	-0.025	-0.050			
EY-14254C	Cylinder	dia.32A	+0.027			+0.077 +0.025	
EY-14250	Cylinder nut	dia.75C	0	-0.02			
EY-14254C	Cylinder	dia.75A	+0.03	0		+0.050 0	
EY-18018M1	Piston	dia.15A	+0.019	0			
EY-18017M1	Distributor	dia.15X ₃	-0.02	-0.07	+0.026 +0.02		
EY-86002	Duplicate slide valve	dia.11A	+0.019	0	+0.010 0.007 (to be adjusted by selection)		
EY-38003	Slide valve	dia.11	0	-0.012			
EY-18017M1	Distributor	dia.26M	-0.06	-0.095			
EY-86001A	Head body	dia.26A	+0.023	0	+0.118 +0.06		

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APPENDIX 11

Table of clearances and fits. Boosters EY-1V, EY-1M and EY-1M-PB

Designation	Description	Size and fit	Maximum deviation, mm		Clearance (+) and interference (-) without coating, mm	Clearance (+) and interference (-) with coating, mm
			upper	lower		
1	2	3	4	5	6	7
EY-1101B	Piston ^{1/}	20A	+0.023		-0.025	
EY-1102B	Sleeve ^{1/}	20M	+0.039	+0.025	-0.010	
EY-1101B	Piston ^{1/}	21.5A	+0.023		-0.030	
EY-1206B	Sleeve ^{1/}		+0.039	+0.025	-0.010	
EY-1101B	Piston ^{1/}	9A	+0.016		+0.020	
EY-1206B	Plunger ^{1/}	9X	-0.013	-0.027	+0.013	
EY-1101B	Piston ^{1/}	5A	+0.013		+0.025	
EY-1064	Needle ^{1/}	5A	-0.004	-0.012	+0.010	
EY-1064M-PB						
EY-1101B	Piston	22A	+0.023		+0.063	-
EY-1091A	Bushing	22X	-0.020	-0.040	+0.020	
EY-1101B	Piston	22A	+0.023		+0.153	
EY-1091A	Bushing	22M ₃	-0.060	-0.130	+0.060	

^{1/} To be adjusted by selection.

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1	2	3	4	5	6	7
EY-1205E	Red	27.5A	+0.023		+0.037	
EY-1101E	Piston	27.5C		-0.014	+0.00	-
EY-1101E	Piston	32X	-0.025	-0.050		+0.077
EY-1058E	Cylinder bushing	32A	+0.027			+0.025
EY-1101E	Piston	56H	-0.095	-0.145		+0.175
EY-1010E	Cylinder	56A	+0.030			+0.095
EY-1010E	End piece unit ^{1/}	13.1C		-0.012	+0.012	
EY-1510E	Bushing ^{1/}	13.1A	+0.019		+0.007	-
EY-1091A						
EY-1102E	Sleeve ^{1/}	13A	+0.019		+0.012	-
EY-1518E	Slide valve ^{1/}	13C		-0.012	+0.007	-
EY-1518M						
EY-1936	Link	6A ₃	+0.025		+0.069	+0.063
EY-1040	Bolt	6X ₃	-0.011	-0.044	+0.011	-0.003
EY-1729	Shackle	19T		-0.023	+0.021	-
State Standard FOCT 5720-51	Ball bearing No.1006	19	+0.002	-0.021	-0.025	-
EY-1936	Link	19T		-0.023	+0.021	-
State Standard FOCT 6121-39	Ball bearing No.6	19	+0.002	-0.021	-0.025	-

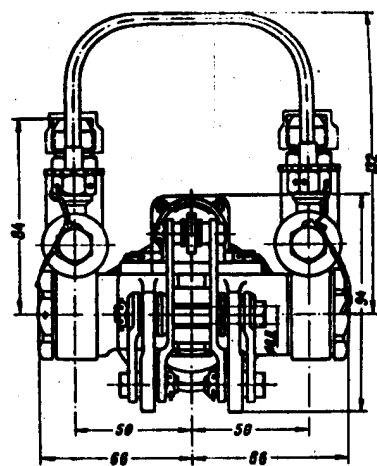
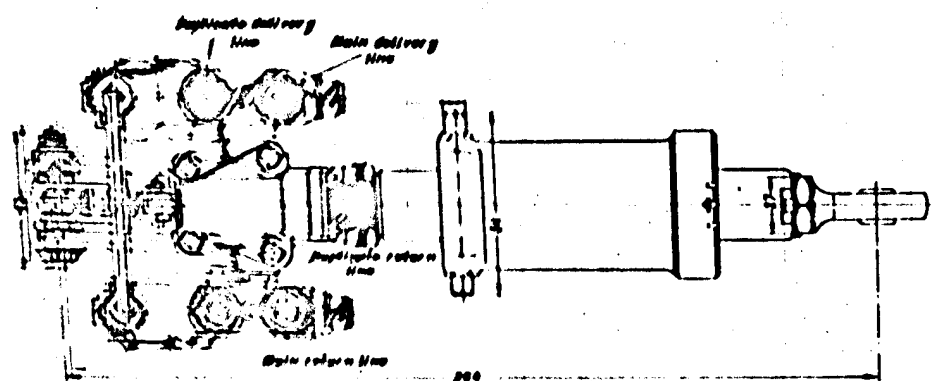
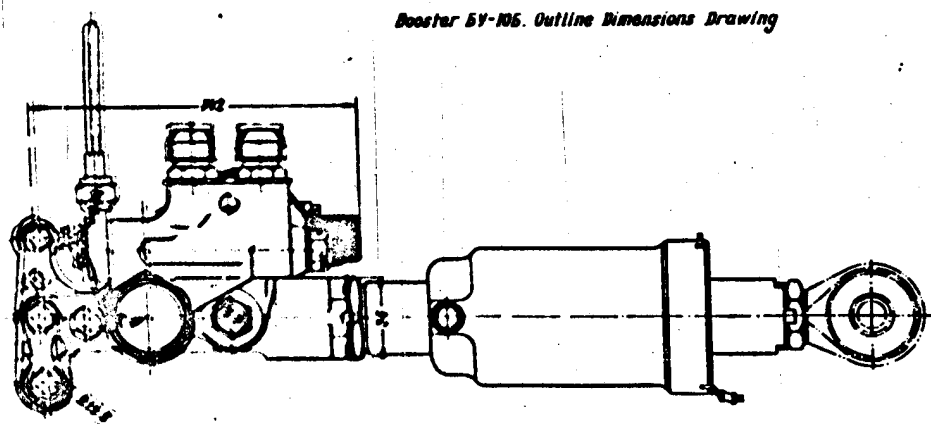
^{1/} To be adjusted by selection.

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1	2	3	4	5	6	7
EY-1010E	Cylinder	56A	+0.030			+0.090
EY1058E	Cylinder bushing	56X	-0.030	-0.060		+0.030
EY-1205E	Red	32X	-0.025	-0.050		+0.077
EY-1010E	Cylinder	32A	+0.027			+0.025
EY-1010M						
EY-1070	Filter	11X ₃	-0.020	-0.070	+0.105	-
EY-1415A	Pipe connection	11A ₃	+0.035		+0.020	-
EY-1518E	Slide valve	7.5A	+0.016		+0.015	-
EY-1518M					-0.016	-
EY-1567B	Damper body	7.5H2a	+0.016	+0.001		
EY-1831B	Head	8A ₃	+0.030		-0.065	-0.105
EY-1838	Bushing	8Hp.18	+0.065	+0.035	-0.005	-0.029
EY-1518B	Slide valve	7.5A	+0.016			
EY-1518M					+0.166	-
EY-1568E	Washer	7.5X ₄	-0.050	-0.150	+0.050	
EY-1510E	End piece body	6A	+0.013		+0.057	+0.051
EY-1041	Bolt	6X ₃	-0.011	-0.044	+0.011	-0.003
EY-1800A	Head unit	6A ₃	+0.025		+0.069	+0.063
EY-1040	Bolt	6X ₃	-0.011	-0.044	+0.011	-0.003
EY-1626Y	End piece	42T	-0.007	-0.035	+0.017	-
TY 100-58	Ball bearing No.900706	42	+0.003	-0.024	-0.038	

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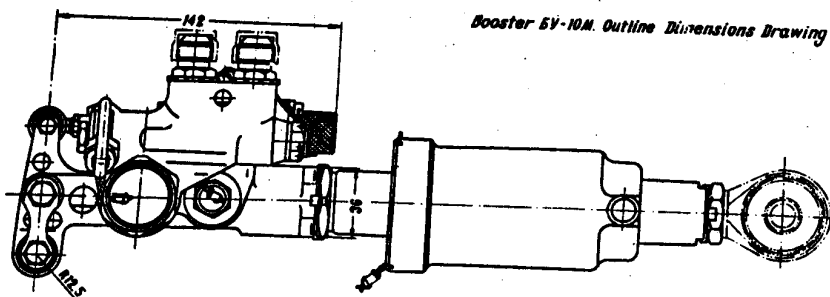
Booster 5Y-105. Outline Dimensions Drawing



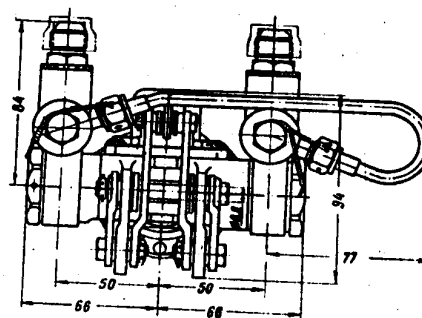
Operating rod travel is 15 mm in either direction from neutral position

Appendix 12

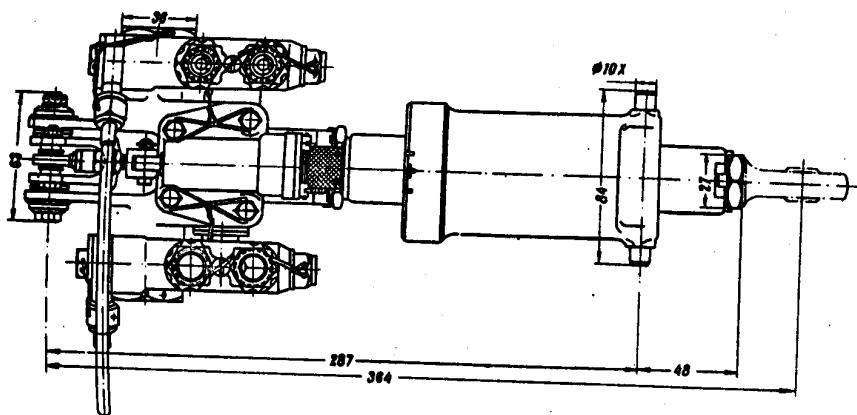
Booster 6V-10M. Outline Dimensions Drawing



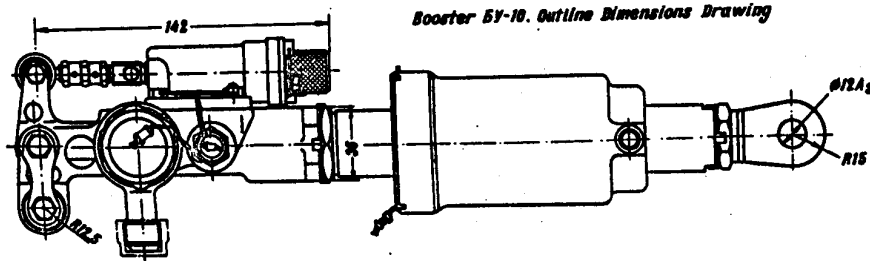
Appendix 13



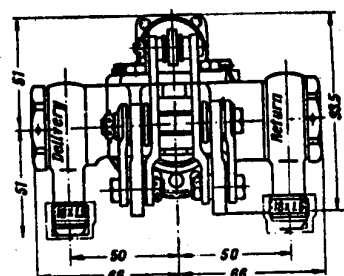
*Operating rod travel is
25 mm in either direction
from middle position
shown in the drawing*



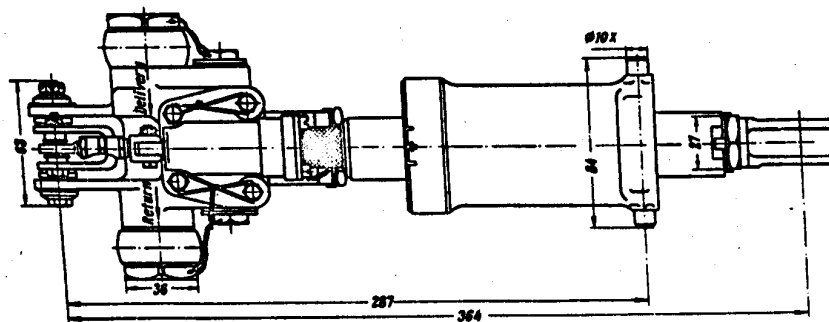
Booster BY-10. Outline Dimensions Drawing



Appendix 14

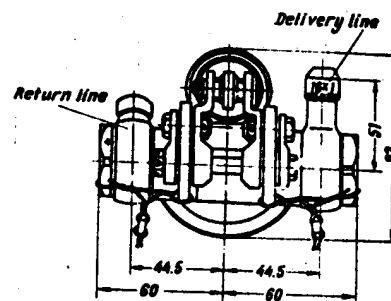
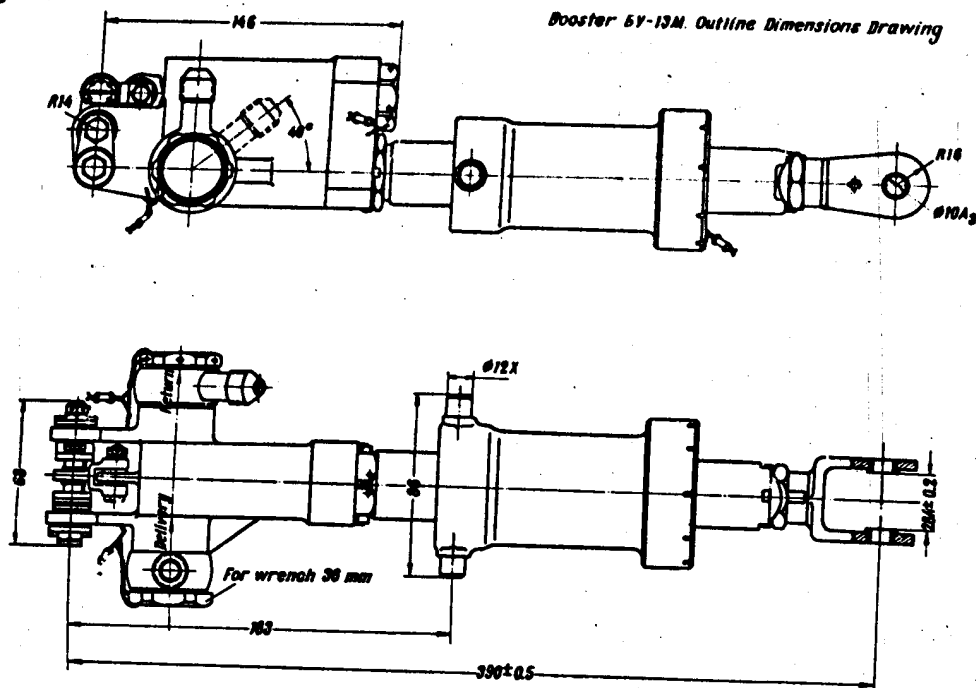


Operating rod travel is 15 mm in either direction from middle position shown in the drawing



Booster 5V-13M. Outline Dimensions Drawing

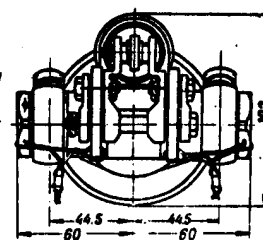
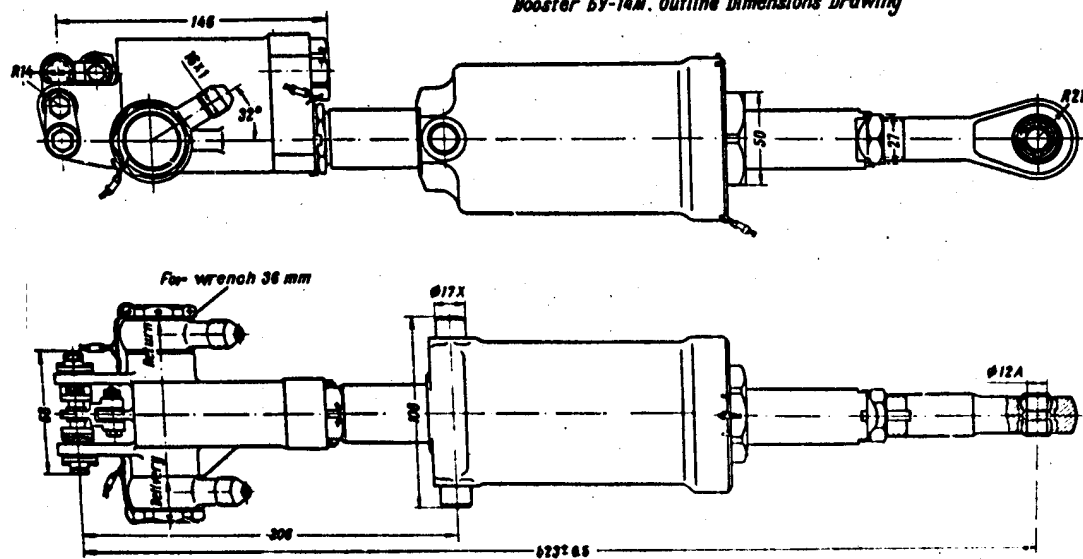
Appendix 17



Operating rod travel is
15 mm in either direction
from middle position
shown in the drawing

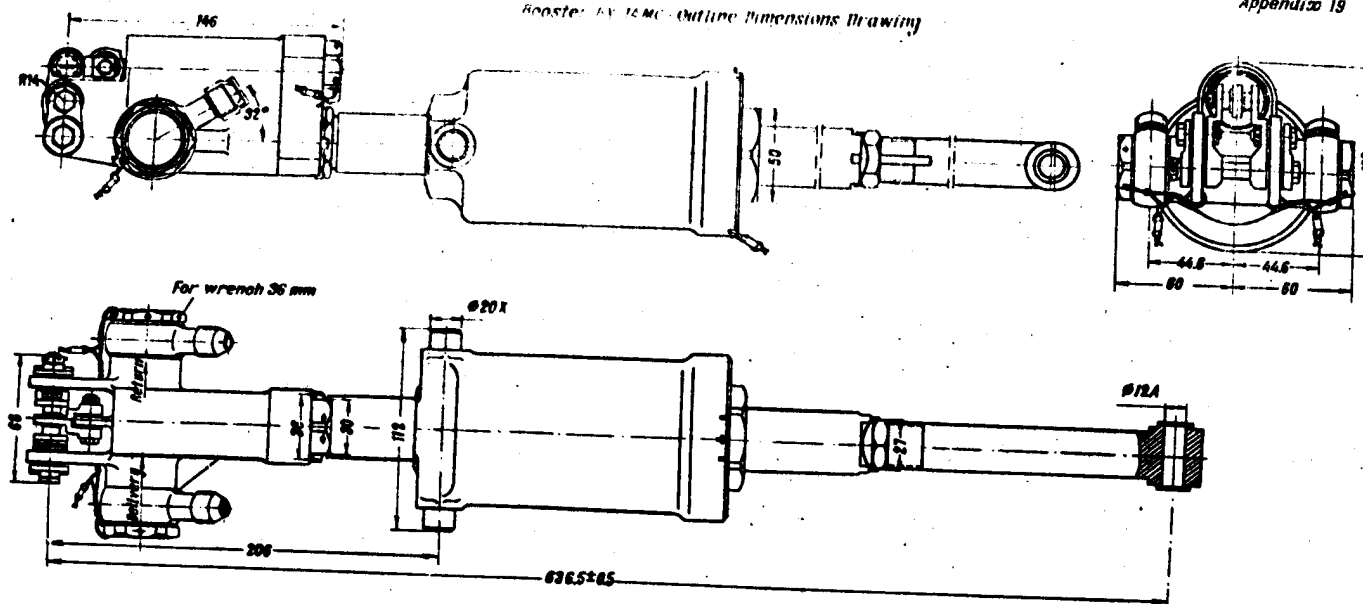
Booster 5V-14M. Outline Dimensions Drawing

Appendix 18



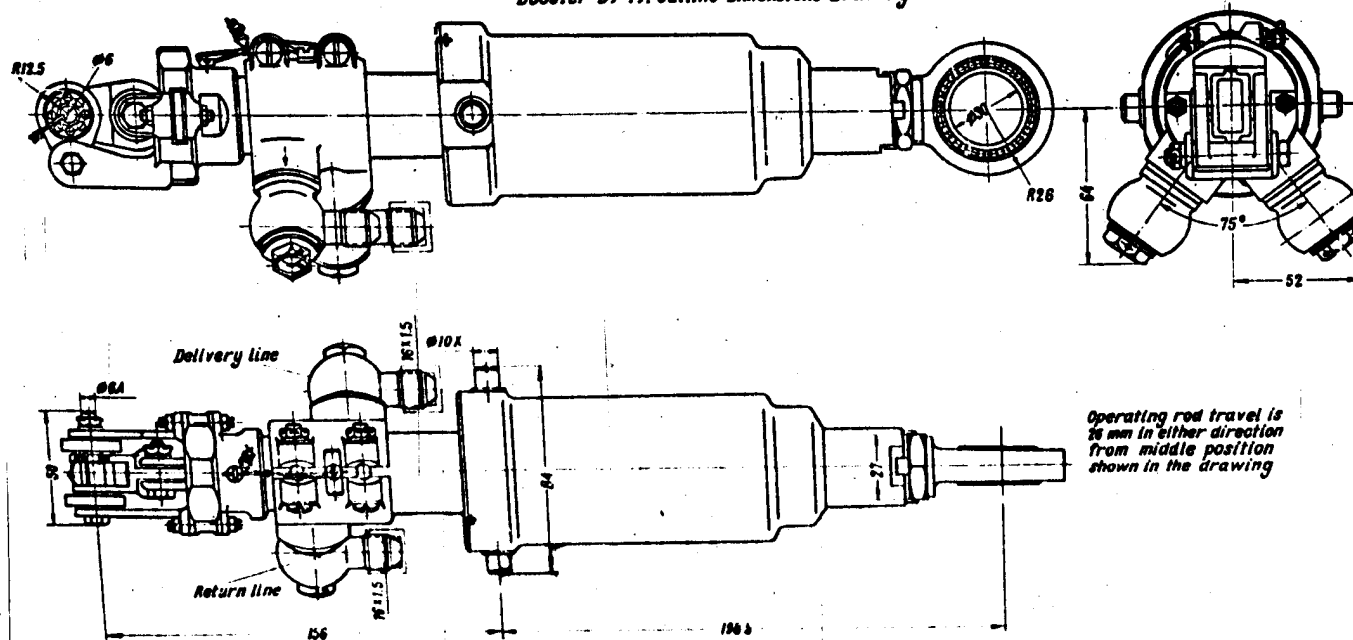
Booster 1Y-12 MC Outline Dimensions Drawing

Appendix 19

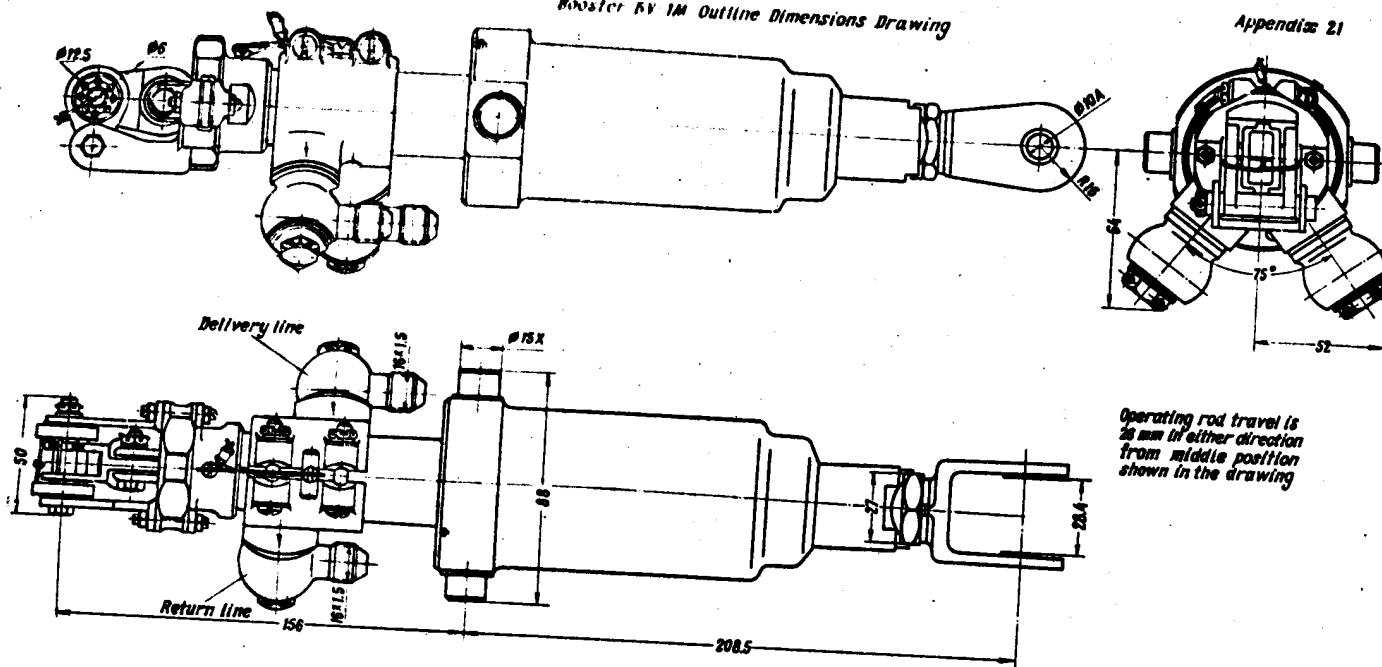


Booster 5Y-1V Outline Dimensions Drawing

Appendix 20

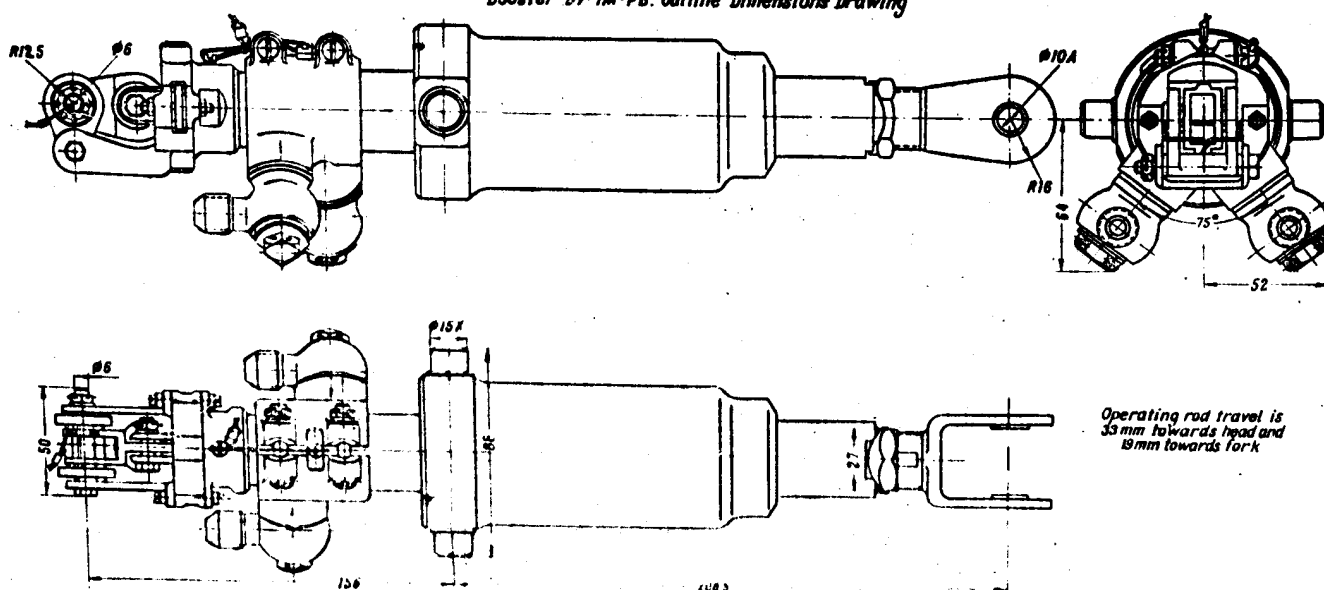


Booster 5V-1M Outline Dimensions Drawing



Appendix 21

Booster 5V-1M-PB. Outline Dimensions Drawing



Appendix 22

SUPPLEMENT

Checking EV-13M and EV-14M Hydraulic Boosters
for Condition and Fluid Pressure in Return Line

Special research work carried out with the aim of determining the operating capability of hydraulic boosters EV-13M and EV-14M has revealed that in some boosters the manufacturing sizes of the damper unit of the main distributing slide valve (the length of ball travel and the diameter of the damper body seat) deviate from the rated sizes. This results in uneven and jerky motion of the aircraft control stick (Fig.1).

In view of the above it will be necessary to check hydraulic boosters for fluid pressure in the return line and to discard the boosters whose dampers are found to be defective.

In subsequent operation such checks should be carried out during the 25-hour inspections and also on requests of pilots who are not satisfied with the functioning of the aircraft control system.

When checking hydraulic boosters for condition and fluid pressure in the return line, use should be made of a special apparatus. The check must be performed as follows:

1. Open the panel providing access to hydraulic boosters EV-14M (or EV-14MC) or the access hatch to booster EV-13M.
2. Disconnect the spring feel mechanisms of the elevator (stabiliser) and aileron control systems. Move aside the spring feel mechanisms so that they do not interfere with free motion of control ball cranks.

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3. Disconnect the return line flexible hoses from the pipe connections of the hydraulic booster.

4. Install the apparatus for measuring the operating fluid pressure in the booster return line between the booster return pipe connection and the return line flexible hose (Fig.2).

5. Open the throttle valve of the apparatus to capacity.

6. Connect a ground hydraulic trolley to the aircraft booster hydraulic system. Switch on the trolley pump.

7. Switch on the hydraulic booster of the elevator (stabilizer), and check the apparatus for tightness by shifting the control stick several times.

8. Check the pressure in the return line of the hydraulic booster by the pressure gauge of the testing apparatus by shifting the control stick with maximum speed in the longitudinal direction through the whole length of its travel.

The pressure in the line must not exceed 8 kg/sq.cm.

9. If the pressure in the return line beyond the hydraulic booster is too high (above 8 kg/sq.cm.):

(a) check to see that the throttle valve of the apparatus is open to full capacity;

(b) check the return line filters for cleanliness and wash the filters;

(c) check the hose inner section for separation and bulging of the rubber; if such defects are evidenced, replace the booster flexible hose;

(d) check the return pipe line for crumpling of pipes.

The above checks should be carried out in the sequence just outlined; after each operation check the pressure in the return line in accordance with Item 8.

10. Check operating condition of the elevator (stabilizer) hydraulic booster, to this end:

(a) close the throttle valve of the testing apparatus; with the aircraft control stick shifted with maximum speed

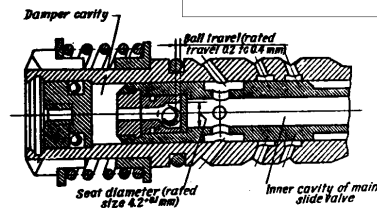


Fig. 1. Diagram of Main Slide Valve Operation of Hydraulic Boosters 85-14M and 85-15M with Increased Length of Ball Travel and Diameter of Damper Body Seat as Compared to Rated Sizes

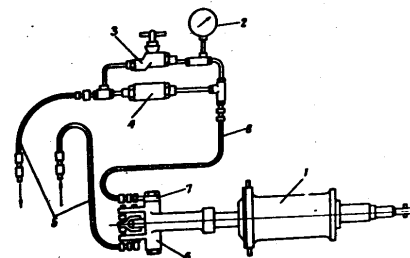


Fig. 2. Connection of Testing Apparatus to Hydraulic Booster System
1-hydraulic booster under check; 2-pressure gauge, rated for 60 kg/sq.cm.;
3-throttle valve; 4-return valve; 5-aircraft hoses; 6-outlet pipe con-
nection; 7-outlet pipe connection; 8-apparatus base (beginning of
return line); 9-outlet pipe connection, full opening at 12 kg/sq.cm.

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through the whole length of its travel, the pressure of the hydraulic fluid must be within 10 to 20 kg/sq.cm. as registered by the pressure gauge of the testing apparatus;

(b) shift uniformly the aircraft control stick 4 or 5 times through the whole length of its travel. After that pull the control stick towards yourself as far as it will go, and supporting it slightly with one finger push it slightly off by short jerks through 20 - 30 mm. If the hydraulic booster is in good repair, the aircraft control stick will move to the forward position without any twitching or jerks towards diving.

In case of twitching or jerks towards diving the hydraulic booster is not fit for service and must be replaced by a new one.

11. Check operating condition of the aileron hydraulic booster after installing the testing apparatus in accordance with Item 4 of the present Supplement, to this end:

(a) switch on the aileron hydraulic booster;
(b) close the throttle valve of the apparatus;
(c) shift uniformly the aircraft control stick from right to left and vice versa 4 or 5 times through the whole length of its travel. After that move the control stick to the left as far as it will go, and supporting it slightly with one finger, push it slightly to the right by short jerks through 20 - 30 mm each time. The aircraft control stick must move to the extreme right position without any twitching or jerks towards the starboard bank.

In case of twitching or jerks of the control stick towards the starboard bank, the hydraulic booster should be replaced by a new one.

12. After the installation of the hydraulic booster in the aircraft, check its operating capability in accordance with Item 10 (in case of boosters EV-14M and EV-14MC) or Item 11 (in case of boosters EV-13M).

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13. Connect the ground hydraulic trolley to the main hydraulic system of the aircraft. Switch on the pump of the hydraulic trolley.

14. Check the pressure in the return line of hydraulic boosters EV-18M and EV-14M (or EV-14MC) when they are operated from the main hydraulic system in accordance with Items 8 and 9.

15. Connect the spring feel mechanisms of the elevator (stabilizer) and aileron control systems.

16. Check the aircraft control in accordance with the Operating Instructions in effect.

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